

ELEVON DESIGN REVIEW

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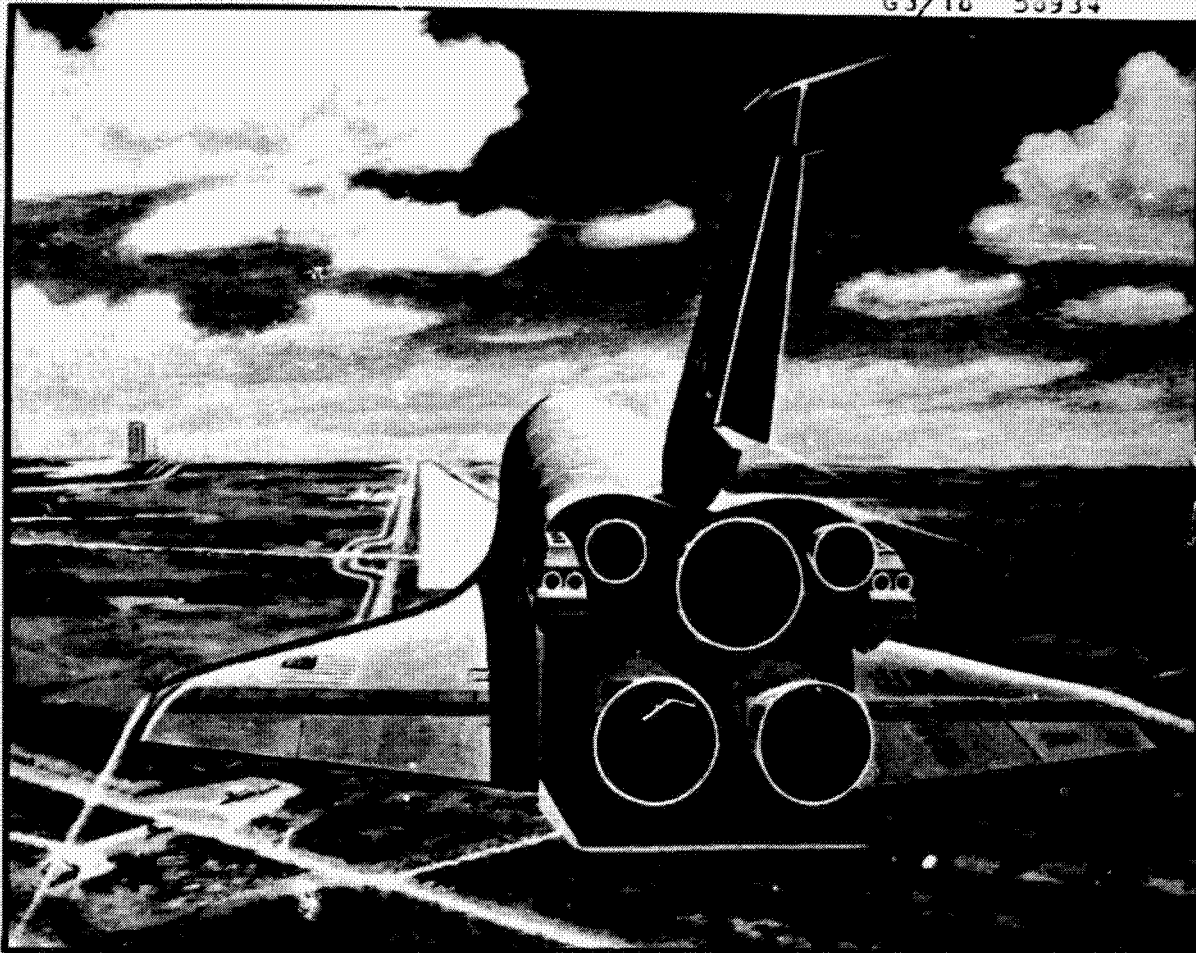
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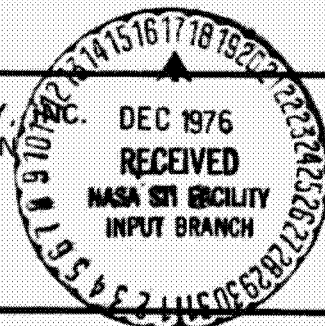
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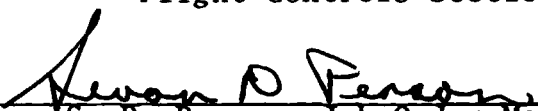
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13. ABSTRACT The document describes the mathematical model implementation of the elevon subsystem for the Shuttle Actuators Simulator (SAS) in the Shuttle Avionics Integration Laboratory (SAIL). The computer subsystem will simulate two outboard and two inboard elevon actuators. ORIGINAL PAGE IS OF POOR QUALITY		
14. SUBJECT TERMS		
<u>actuator</u>	<u>simulator</u>	<u>CSMP</u>
<u>mathematical model</u>	<u>flight controls</u>	<u>stiction</u>
<u>elevon</u>	<u>SAIL</u>	<u>Space Shuttle</u>

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LIST OF ABBREVIATIONS, ACRONYMS, AND SYMBOLS

ASA	aerosurface servo amplifier
°C	degrees Celsius
CH	channel
cis	cubic inches per second
CKT	circuit
CMOS	complementary metal oxide semiconductor
deg/sec	degrees per second
EG	Control Systems Development Division (NASA)
EJ	Avionics Systems Engineering Division (NASA)
°F	degrees Fahrenheit
F/B	feedback
FET	field effect transistor
FS	full scale
GND	ground
H	henry
Hz	hertz
IC	initial condition
in	inch
ISO	isolation
J-box	junction box
lb	pound
LEC	Lockheed Electronics Company, Inc.
LVDT	linear variable differential transformer
mA	milliampere

MDM	multiplexer/demultiplexer
mH	millihenry (1×10^{-3} henry)
mV	millivolts (1×10^{-3} volts)
PC	printed circuit
pF	picofarad (1×10^{-9} farad)
POS	position
PRI	primary
rad	radian
REF	reference
rms	root-mean-square
SAIL	Shuttle Avionics Integration Laboratory
SAS	Shuttle Actuators Simulator
S/C	signal conditioner
SDS	Shuttle Dynamics Simulator
sec	second
SEC	secondary
SW	switch
TBD	to be determined
TOC	Test Operations Center
TTL	transistor-transistor logic
V	volts
Vac	volts alternating current
VDS	Vehicle Dynamics Simulation
XDUCER	transducer
ΔP	delta pressure
ΔP_p	delta pressure primary

ΔP_s	delta pressure secondary
Ω	ohms

1. INTRODUCTION

1.1 PURPOSE

This document presents the design of the elevon subsystem for the Shuttle Actuators Simulator (SAS). This simulator will replace the elevon actuator hardware in the Shuttle Avionics Integration Laboratory (SAIL). It will consist of four elevon actuators.

1.2 SCOPE

The scope of this document encompasses all technical aspects for the elevon subsystems. See figure 1-1. It details interface design, signal characteristics, and system performance. Nontechnical requirements are introduced where required to aid in comprehension.

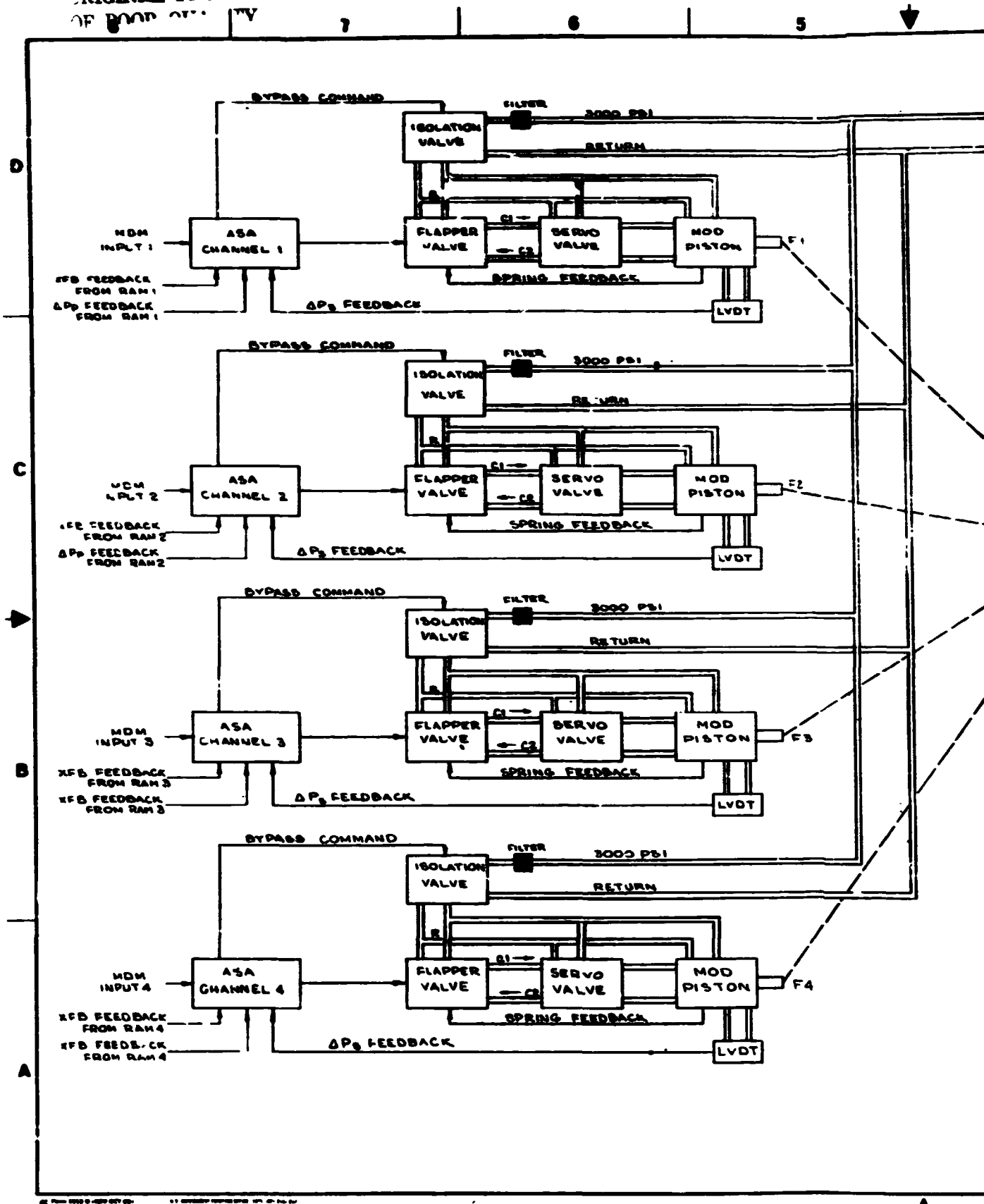
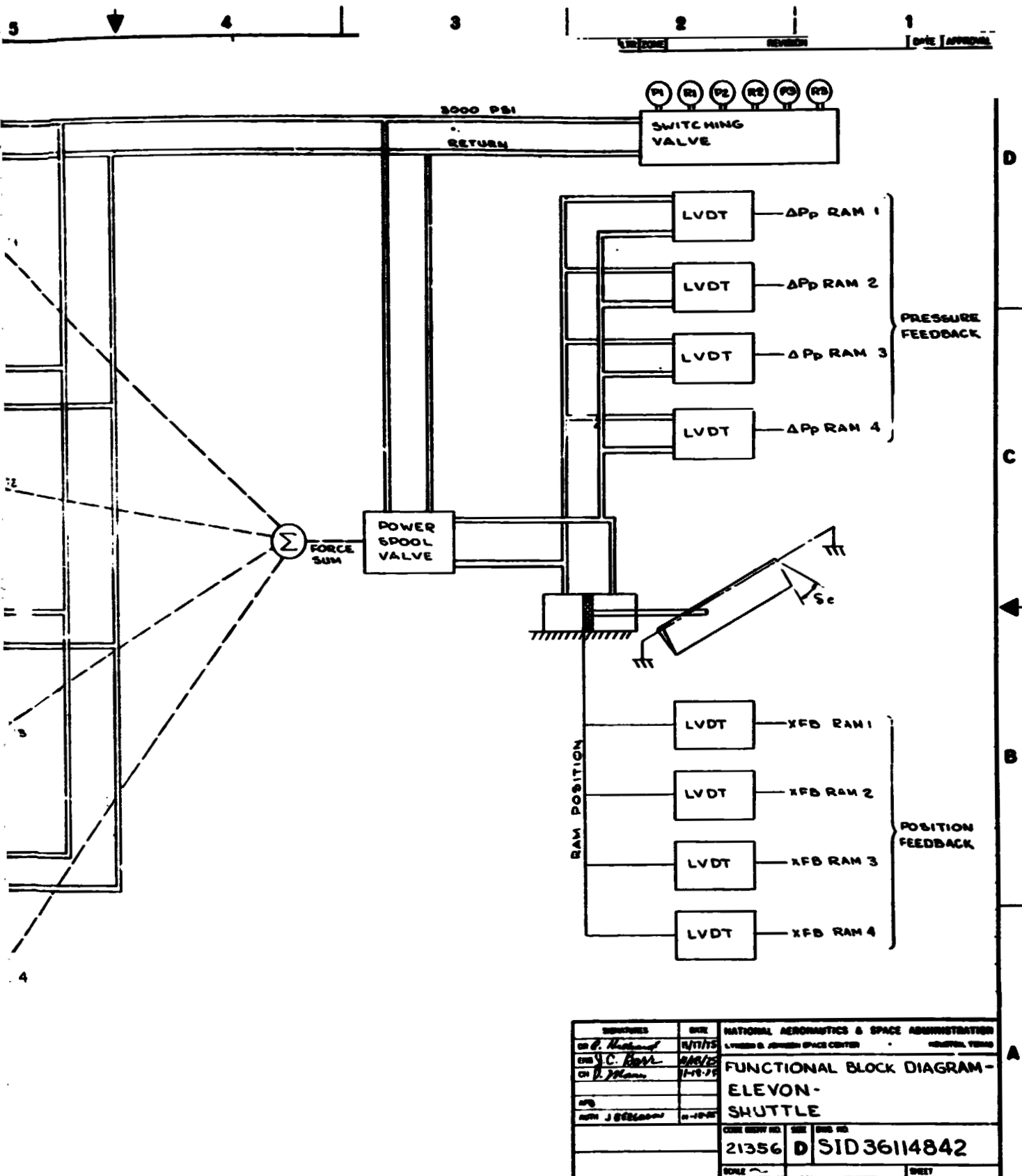


Figure 1-1. - Functional block diagram of Shuttle Actuators S.



block diagram of the elevon subsystem for the
tiple Actuators Simulator.

2. MATHEMATICAL MODELS

2.1 DEFINITION OF MATHEMATICAL MODELS

Four mathematical models are to be used. Two are from Rockwell International document SD74-SH-0324A, *Descriptions and Mathematical Models of Aerosurface Actuators*, September 1975. Associated material is presented in tables 2-I, 2-II, and 2-III.

2.1.1 FULL-UP MODEL

The full-up model that is assumed to be complete and representative of the hardware is shown in figure 2-1 and is identified as model 1 in the Rockwell document. The implementation of this model on the IBM-360 computer defines the baseline data for the simulator correlation and checkout.

2.1.2 TOLERANCE MODEL

The tolerance model which defines the minimum acceptable standard of performance is shown in figure 2-2 and is identified as model 2 in the Rockwell document. It defines the tolerance data for the elevon simulator correlation and checkout.

2.1.3 IMPLEMENTATION MODEL

The implementation model in figure 2-3 is to be mechanized in simulator hardware. Its performance is between the full-up and the tolerance models and has been implemented on the IBM-360 computer.

2.1.4 MECHANIZATION MODEL

The physical constants for the mechanization model have been resolved into resistor ratios. This model is suitable for direct implementation on an analog computer.

2.1.5 PARAMETERS

The inboard and outboard elevon models differ only in the value of physical constants. All temperature sensitive constants used in the mathematical models were selected at the 100° F value. Table 2-IV is a list of the values of the constants used and their class designations. The elevon constants are divided into three classes by anticipated adjustment probability. The classifications are referred to as A, B, and C. The classifications correspond to:

- Class A – Those parameters recognized as likely to change and, as such, are mounted on readily accessible and removable socket connectors as shown in figure 2-4.
- Class B – Those parameters recognized as unlikely to change but may do so under unusual conditions or circumstances. These will be permanently soldered onto the printed circuit (PC) board assemblies.
- Class C – Those parameters defined as definitely unchanging and whose values are fixed. These could be elements of a transfer function buried in circuitry, constants lumped together, or potted modules. Changes to these would likely require major circuit design and/or layout revisions.

The plan is to keep the number of class A components at a minimum and maximize the number of components into classes B or C.

2.2 MODEL IMPLEMENTATION VERIFICATION

2.2.1 FULL-UP MODEL

It is assumed that this model represents the hardware accurately. The program output will be compared with the hardware design specification in Rockwell International documents. Four tests were run to verify the computer implementation:

- Actuator stroke limits were compared to table III page 14, Rockwell International document MC621-0014.

- Velocity gain was compared to the gain requirements in figure 5 and table 2 of Rockwell International document 392-200-75-340 for actuator loads of 0, 2×10^4 , and 3.945×10^4 .
- Gain-phase frequency response data were compared to the data of figure 4 of Rockwell International document 392-200-75-340.
- The step response was compared to figure 05.10-2 of Rockwell International document ML0101-0001-005, sheet 3 of 185, Rev. A-01.

2.2.2 IMPLEMENTATION MODEL

This model is verified by three tests:

- Actuator stroke limits were compared to model 1.
- Gain-phase frequency response data were compared to model 1.
- Step response locus comparisons were made to model 1 for δ_e , $\dot{\delta}_e$, I , X_{PS} , Q_L , P_S , and P_L .

2.2.3 MECHANIZATION MODEL

The control valve was implemented on an Astrodata CI-175 general-purpose, analog computer. The step response, shown in figure 2-5, was compared to model 1.

TABLE 2-I. - ELEVON SCALING FACTORS

Factor	Variable	Units	Inboard	Outboard
K1	I	mA	8.60	8.60
K2	X _{FL}	in	0.0016	0.0016
K3	X _S	in	0.015	0.015
K4	F _I	lbs	579.0	579.0
K5	\dot{X}_{PS}	in/sec	62.0	62.0
K6	X _{PS}	in	0.065	0.065
K7	P _{VS}	psi	3000.	3000.
K8	P _V	psi	3000.	3000.
K9	P _L	psi	3000.	3000.
K10	—	—	—	—
K11	Q _L	in ³ /sec	400.0	180.0
K12	P _I	psi	3000.	3000.
K13	—	—	—	—
K14	—	—	—	—
K15	T _{TOT}	in-lbs	1.0×10^6	5.0×10^5
K16	F _L	lbs	65400.	54060.
K17	T _R	in-lbs	1.0×10^6	5.0×10^5
K18	TAERO	in-lbs	1.0×10^6	5.0×10^5
K19	$\dot{\delta}_e$	deg/sec	100.	100.
K20	δ_e	deg	36.5	36.5
K21	X _R , X _{STR}	in	7.320	4.266
K22	X _{FB}	in	7.320	4.266
K23	V _{PL}	volts	5.000	5.000
K24	V _{XFB}	volts	5.000	5.000
K25	V _{CI}	volts	5.000	5.000
K26	M _R	in	15.10	8.80

TABLE 2-II. - ELEVON MODEL NONLINEARITIES (NL)

NL*	Description	Value	Units
A	Servo Amplifier Current Limiter	$\pm 8.6 \pm 1.0$	mA
B [†]	Torque Motor Hysteresis (Full Band)	0.029	mA
C	Torque Motor (Nozzle) Stroke Limite	± 0.0016	in
D	Second Stage Friction (Stiction)	0.20	lb
E	Second Stage Spool Stroke Limit	± 0.015	in
F	Mod Piston Friction (Stiction), Total/Per Channel		
	Four Channels Active	11.6/2.9	lb
	Three Channels Active, One Bypassed	12.5/4.17	lb
	Three Channels Active, One Hardover	40.2/13.3	lb
	Three Channels Active, One at Null/Open	40.2/13.3	lb
	Two Channels Active, Two Bypassed	12.4/6.2	lb
G	Power Spool Stroke Limit	± 0.065	in
H	Power Ram Friction (Stiction), Inboard	$1470.R(\delta_E)$	in-lb
	Outboard	$1452.R(\delta_E)$	in-lb
J	Pressure Transducer Hysteresis (Full Band)	62.5	psi
K	Elevon Seal Panel Friction (Stiction)	15000.	in-lb

*These letters correspond to the letters circled in figures 2-2 and 2-3.

$$^{\dagger}B = 0.02 + \left(\frac{0.16 \pm 0.06}{7.6} \right) I$$

NOTE: Assume a running (coulomb) friction magnitude at 1/3 of Stiction value.

TABLE 2-III. - ELEVON MODEL CONSTANTS

Constant	Description	Inboard 100° F	Outboard 100° F	Units
A_R	Ram Piston Area	21.80	18.02	in^2
D_E	Elevon Viscous Damping (Mechanical)	45000.	15000.	in-lb-sec
I_E	Elevon Inertia About Hinge Line	9473.	2663.	in-lb-sec^2
K_B	Flow Force (Bernoulli) Coefficient	0.755	0.319	in
K_{FB}	Actuator Position Transducer Gain (LVDT)	0.683	1.173	V/in
K_{QPS}	Power Spool Flow Gain	124.7	51.8	$\text{in}^3/\text{sec} \sqrt{\text{lb}}$
K_S	Local Structural Stiffness External to Actuator	298000.*	154000.*	in/lb
K_T	Total Actuation System Stiffness	177446.	128583.	lb/in
K_0	Pressure Loss Constant	7.67	4.24	psi
K_1	Linear Pressure Loss Coefficient (Primary Valve)	0.10756	0.16476	psi/cis
K_2	Quadratic Pressure Loss Coefficient (Primary Valve)	0.01765	0.05228	psi/cis^2
K_{12}	Linear Pressure Loss Coefficient (Second Stage Valve)	0.0922	0.1494	psi/cis
K_{22}	Quadratic Pressure Loss Coefficient (Second Stage Valve)	0.00997	0.0446	psi/cis^2
V_R	Total Effective Volume of Ram Cylinder	337.1	165.1	in^3

*Nominal value (tolerance range = 60% to 150% nominal)

TABLE 2-IV. - ELEVON PARAMETERS

Constant	Program name	Description	Value	Units	Parameter class
A _C	AC	Area, ΔP Feedback Piston	0.008975	in ²	C
A ₂	AP	Area, Second Stage Spool	0.02761	in ²	C
A _{PS}	APS	Area, Mod Piston	0.1930	in ²	C
B	BETA	Hydraulic Fluid Bulk Modulus	171700.	psi	C
B _P	BP	Viscous Damping, Second Stage Spool	0.0648	lb-sec/in	B
B _{PS}	BPS	Viscous Damping Mod Pistons and Power Spool	1.386	lb-sec/in	B
C ₁	C1	Flow/Displacement Characteristics, Nozzle	185.2	in ² /sec	B
C _L	CL	Power Spool Laminar Leakage Flow Coefficient	1.08 × 10 ⁻⁸	in ⁶ /lb-sec	B
C _Q	CQ	Second Stage Flow Gain Coefficient	4.59	in ³ /sec /TB	B
C ₃	CTH	Flow/Pressure Characteristic, Fixed Restriction	0.0000876	in ⁵ /lb-sec	B
C ₂	CTW	Flow/Pressure Characteristic, Nozzle	0.1096	in ⁴ /lb-sec	B
K _A	KAMP	Servo Amplifier Position Error Gain	15.0	mA/V	C
K _C	KC	Dynamic Load Damping Gain	1.71	mA/V	B
K _δ	KL	Elevon Aerodynamic Spring Rate	0.21 × 10 ⁻⁶	in-lb/rad	C
K _N	KN	Nozzle Pressure Feedback Constant	0.000138	in-lb/psi	B
K _P	KP	Spring Rate, Second Stage Spool (total)	1200.	lb/in.	C
K _{PS}	KPS	Secondary Pressure Feedback Gain (Model 2 only)	7.479 × 10 ⁻⁶	in-lb/psi	C
K _{PT}	KPT	Pressure Transducer Gain	0.00167	V/psi	B
K _{QS}	KQS	Secondary Flow Gain Coefficient (Model 2 only)	4.9769	in ³ -lb ^{3/2} /sec	C
K _{TM}	KTM	Torque Motor Gain	0.045	in-lb/mA	A
K _{XPS}	KXPS	Wire Feedback Gain, Mod Piston to Torque Motor	6.22	in-lb/in	B
L	L	Power Spool Overlap	0.001	in	C
L _{AP}	LAP	Power Spool Effective Overlap	0.00118	in	B
C _D	LD	Demodulator Filter Damping Factor	0.707	—	C
L _N	LN	Effective Torquer Inverse Stiffness	0.01693	in/in-lb	B
M _P	MP	Mass, Second Stage Spool	0.0000683	lb-sec ² /in	C
M _{PS}	MPS	Mass, Mod Pistons and Power Spool (total)	0.00207	lb-sec ² /in	C
P _{SS}	PSS	Supply Pressure to Actuator (nominal)	3000.	psi	A
R _{CL}	RCL	Power Valve Spool/Sleeve Radial Clearance	0.0000475	in	C
τ _{DT}	TDT	LVDT Time Constant	0.004	sec	C
V ₁	V	Nozzle Spool Volume (both sides)	0.0838	in ³	C
V _{T2}	VT	Mod Piston Effective Volume (total)	0.62	in ³	C
ω _D	WD	Demodulator Filter Natural Frequency	314.	rad/sec	C
X ₀	XO	Flapper-Nozzle Spacing	0.00185	in	C
X _{PSL}	XPSL	Power Spool Maximum Travel	0.065	in	C
τ _{XS}	TXS	Second Stage Time Constant	0.002	sec	C

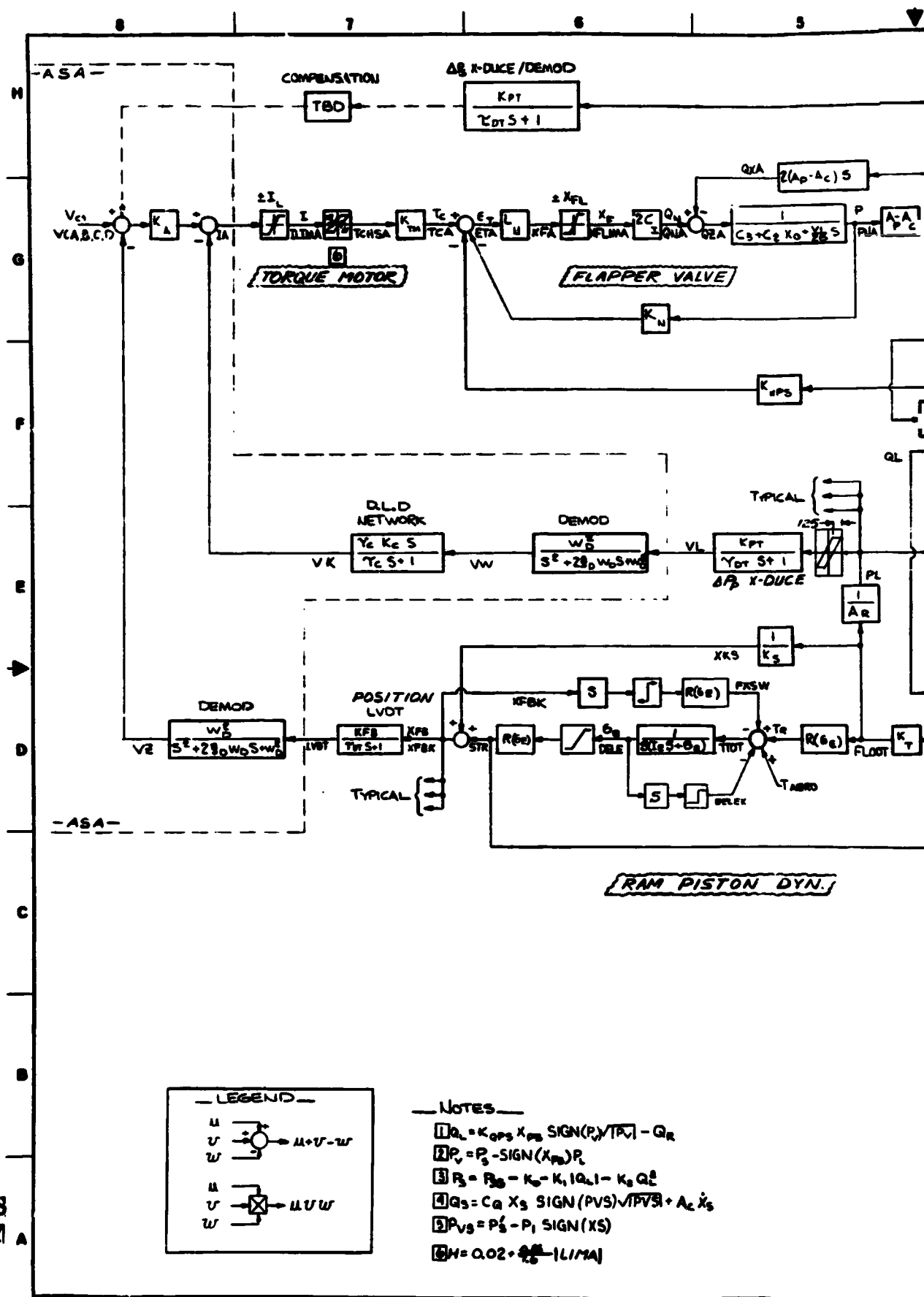
Note: Adjust B_E and I_E for δ_e in degrees (÷57.3)

$$K_{QS} = \frac{2C_1(A_P - A_s)}{K_P(C_2X_0 + C_3)} = 19.8099 \quad B_E = 261.799 \frac{\text{in-lb}}{\text{deg/sec}}$$

$$K_{PS} = \frac{A_C}{K_P} = 7.4791 \times 10^{-6} \quad I_E = 46.4781 \frac{\text{in-lb}}{\text{deg/sec}^2}$$

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Figure 2-1. - F

Figure 2-1. — Full-up model.

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The diagram illustrates the hydraulic control system for the elevator, divided into two main sections: the Secondary Valve and the Elevon Actuator.

Secondary Valve Section:

- Input V_C is summed with a feedback signal from the TSD (Total Secondary Demand) block.
- The resulting signal passes through a gain block K_A and is summed with a feedback signal from the TSD block.
- The signal then passes through a derivative block $\frac{d}{dt}$ and a saturation block $\frac{1}{1 + \frac{1}{K_A}}$.
- The output is summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .

Elevon Actuator Section:

- Input V_C is summed with a feedback signal from the TSD (Total Secondary Demand) block.
- The resulting signal passes through a gain block K_A and is summed with a feedback signal from the TSD block.
- The signal then passes through a derivative block $\frac{d}{dt}$ and a saturation block $\frac{1}{1 + \frac{1}{K_A}}$.
- The output is summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .
- The signal is then summed with a feedback signal from the TSD block and passes through a gain block K_{TS} .

Notes:

1. U and V are summed to produce $U+V$.
2. U , V , and W are summed to produce $U+V+W$.
3. U is summed with W to produce $U+W$.

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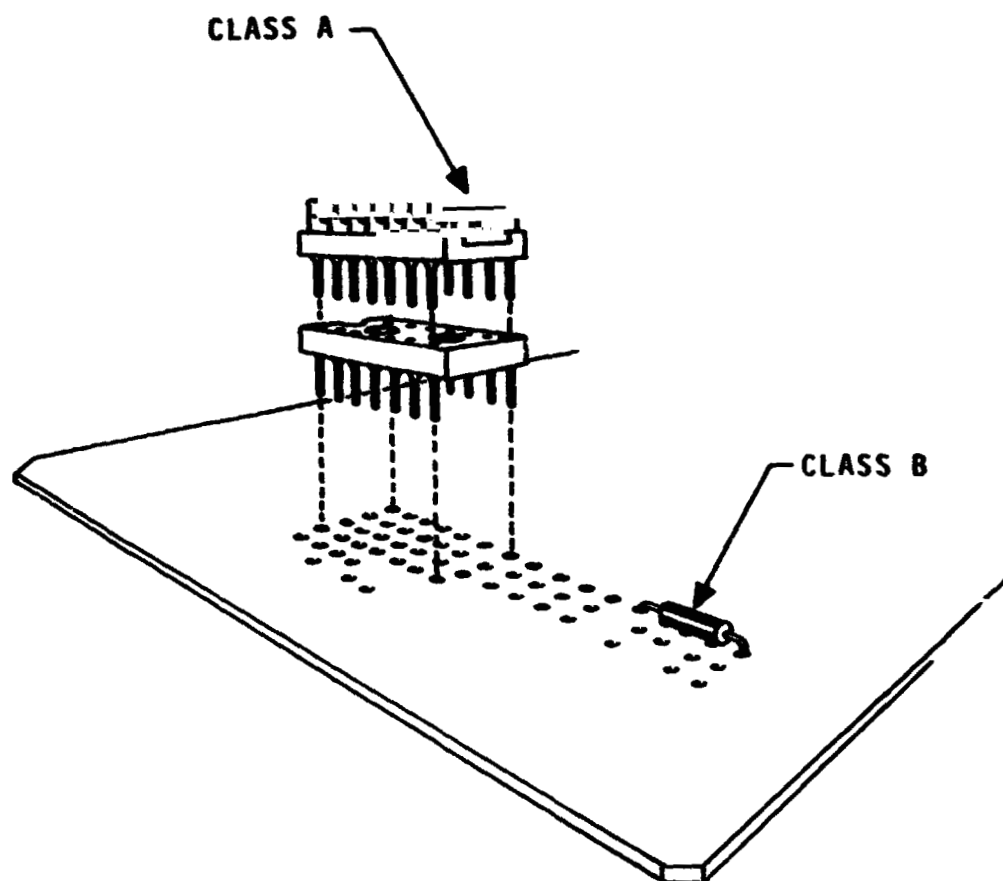
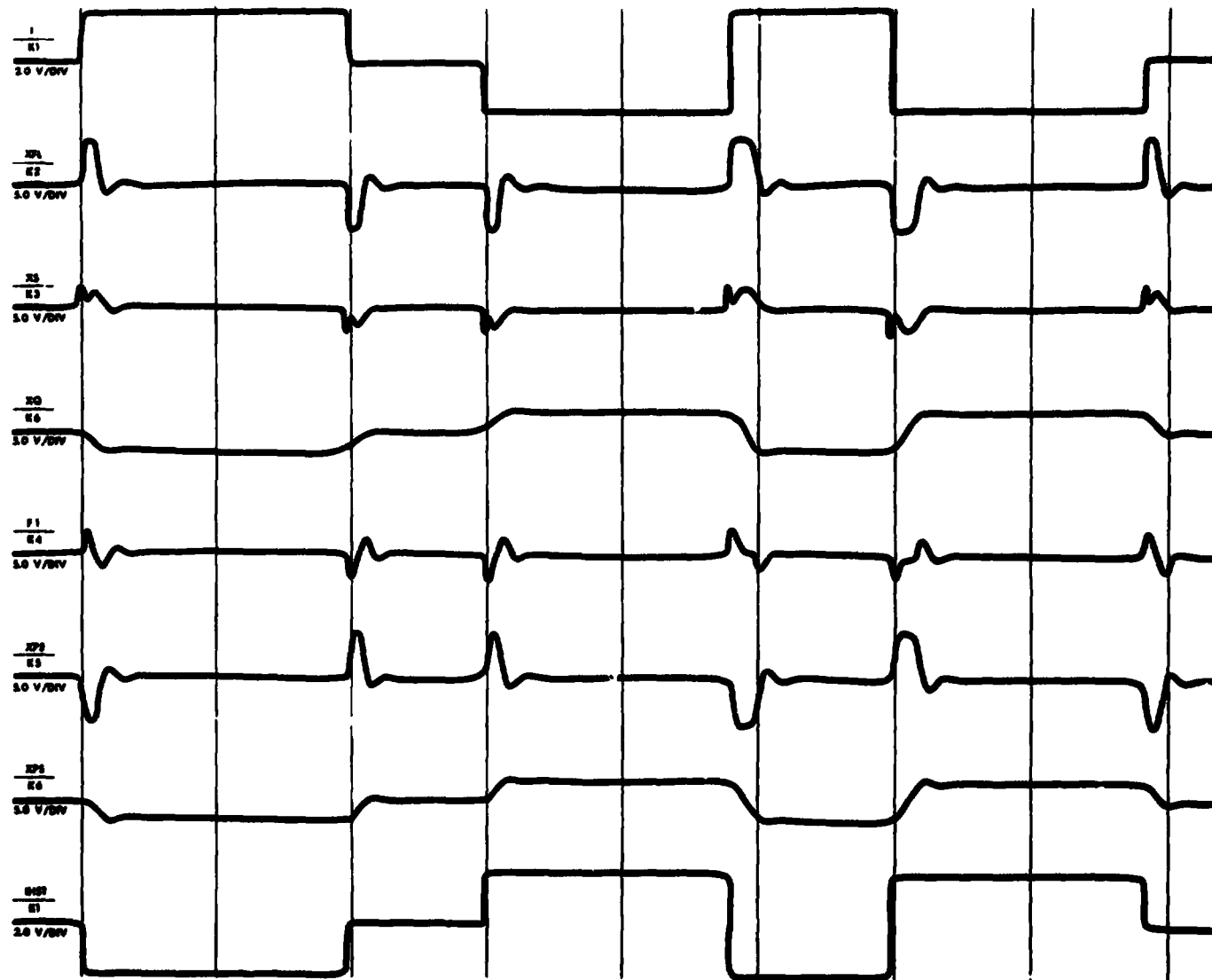


Figure 2-4. - Class A and B components.

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RUN ON 2473

Figure 2-5. — Mechanization verification.

3. INTERFACE REQUIREMENTS

The four interfaces that have been established for the SAS are shown in figure 3-1. These are the interfaces between the SAS and the Shuttle Dynamics Simulator (SDS), between the SAS and the Test Operations Center (TOC) (via J-box 3), between the SAS and the Aerosurface Servo Amplifier (ASA), and the checkout interface which is used for functional checkout and maintenance. The cables required to complete these interfaces are shown in figure 3-2.

3.1 GENERAL REQUIREMENTS FOR BUFFERING

3.1.1 OUTPUT SIGNALS

All SAS output signals except ASA-related signals will be buffered by isolation amplifiers. All output from the SAS to the ASA will be transformer isolated.

3.1.2 INPUT SIGNALS

All SAS input signals except ASA-related signals will be buffered by differential amplifiers or by optical isolators. All SAS inputs from the ASA will be buffered by differential amplifiers, optical isolators, or transformer coupling.

3.1.3 GROUNDS

No external signal grounds will be referenced within the subsystem. All signal ground paths will be isolated by appropriate devices to prevent any ground connection within the subsystem. One and only one ground reference within the subsystem unit will be brought out of the chassis. This ground point in the subsystem unit will be available for reference to the SAIL single-point ground. The equipment ground reference will be connected to the SAS chassis. See figure 3-3.

3.2 ASA/SAS INTERFACE

3.2.1 GENERAL DESIGN

This section outlines the general systems design of the interface. The detailed circuits are presented in section 7, Detailed Circuit Design.

3.2.2 CONNECTORS

The interface for the ASA/SAS will be four 66-pin connectors. There is one connector arranged to connect directly into the back of each elevon chassis. See figure 3-4.

3.2.3 SIGNALS REQUIRED

See figure 3-5.

- Input

Servo valve – four required

Isolation solenoid – four required

Excitation 400 Hz 26 Vac – eight required

- Output

Position transducer – four required

ΔP_s differential pressure transducer – four required

ΔP_p differential pressure transducer – four required

3.2.4 SIGNAL CHARACTERISTICS

- Input

Servo valve – 1100 ohms (Ω) and 6.0 henrys (H) inductance and 3300 picofarads (pF) capacitance in parallel. See figure 3-6.

Isolation solenoid – 75 Ω and 400 millihenry (mH) inductance in parallel with 3300 pF capacitance. See figure 3-7.

- Output

Position transducer – Position transducer output is transformer isolated from SAS subsystem and is a 400 Hz amplitude modulated signal. Reference will be from the ASA 400 Hz excitation power. Null voltage will be 100 mV rms or less and phase shift between excitation and output will be $\pm 6^\circ$ maximum. See figure 3-8. Phasing is positive for trailing edge down.

Differential pressure transducer – Differential pressure transducer output is transformer isolated from SAS subsystem and is a 400 Hz amplitude modulated signal. Reference will be from the ASA 400 Hz. Null voltage will be 100 mV rms or less and phase shift between excitation and output will be $\pm 6^\circ$ maximum. See figure 3-9.

3.3 J-BOX 3/SAS INTERFACE

3.3.1 SIGNALS REQUIRED

- Input. There will be an analog initial conditions (IC) signal which will be generated in the TOC. This signal will be used to control the signals δ_E and X_{FB} for each elevon subsystem. There will also be a discrete input which will cause an integrator in each elevon actuator subsystem to switch from "IC" to "Operate." There will also be two fault lines from TOC to the SAS which will cause insertion of fault conditions in the SAS. Fault insertion is discussed in section 4, and initialization is discussed in section 6.
- Output. There will be four rate signals, one from each elevon sent to the TOC via J-box 3. There will be four position signals, one from each elevon, sent to the TOC via J-box 3. See figure 3-10. These will be analog signals with 0 to 5 V amplitude.

3.3.2 SIGNAL CHARACTERISTICS

- Input. All discrete inputs will be transistor-transistor logic (TTL) levels capable of driving or sinking 25 mA. See figure 3-11. All analog inputs will be 0 to 5 V nominal with ± 15 maximum input voltage at the SAS. All analog input signals will be via differential amplifiers as shown in figure 3-12.
- Output. All analog signals will be via an isolation type amplifier to prevent ground loops.

3.4 SDS/SAS INTERFACE

3.4.1 SIGNALS REQUIRED

3.4.1.1 Position Signal

A position signal from each elevon is required for use in the SDS. The panel positions are labeled δ_{ELO} , δ_{ELI} , δ_{ERI} , and δ_{ERO} . See figure 3-13.

3.4.1.2 Acceleration Signals

Four acceleration signals are required. These are the four elevon panel accelerations - $\ddot{\delta}_{ELO}$, $\ddot{\delta}_{ELI}$, $\ddot{\delta}_{ERI}$, and $\ddot{\delta}_{ERO}$. See figure 3-13.

3.4.1.3 Hinge Moment

SDS will provide a return signal to SAS which is proportional to the hinge moment for each elevon - M_{HERO} , M_{HERI} , M_{HELI} , and M_{HELO} . See figure 3-13.

3.4.1.4 Simulated Hydraulic Pressure

An internally generated adjustable reference voltage will be used to set the system hydraulic supply pressure; however, provisions will be made for the elevon subsystem to accept an external voltage proportional to the system hydraulic supply pressure. See figure 3-14.

3.4.2 SIGNAL CHARACTERISTICS

All input and output signals will comply with paragraph 3.1, *General Requirements for Buffering*.

- Input. All analog input from the SDS will be ± 10 V nominal amplitude with ± 15 V maximum amplitude at the SAS interface.
- Output. All analog signals output from the subsystem units for use in the SDS will be ± 10 V nominal amplitude with ± 15 V maximum amplitude.

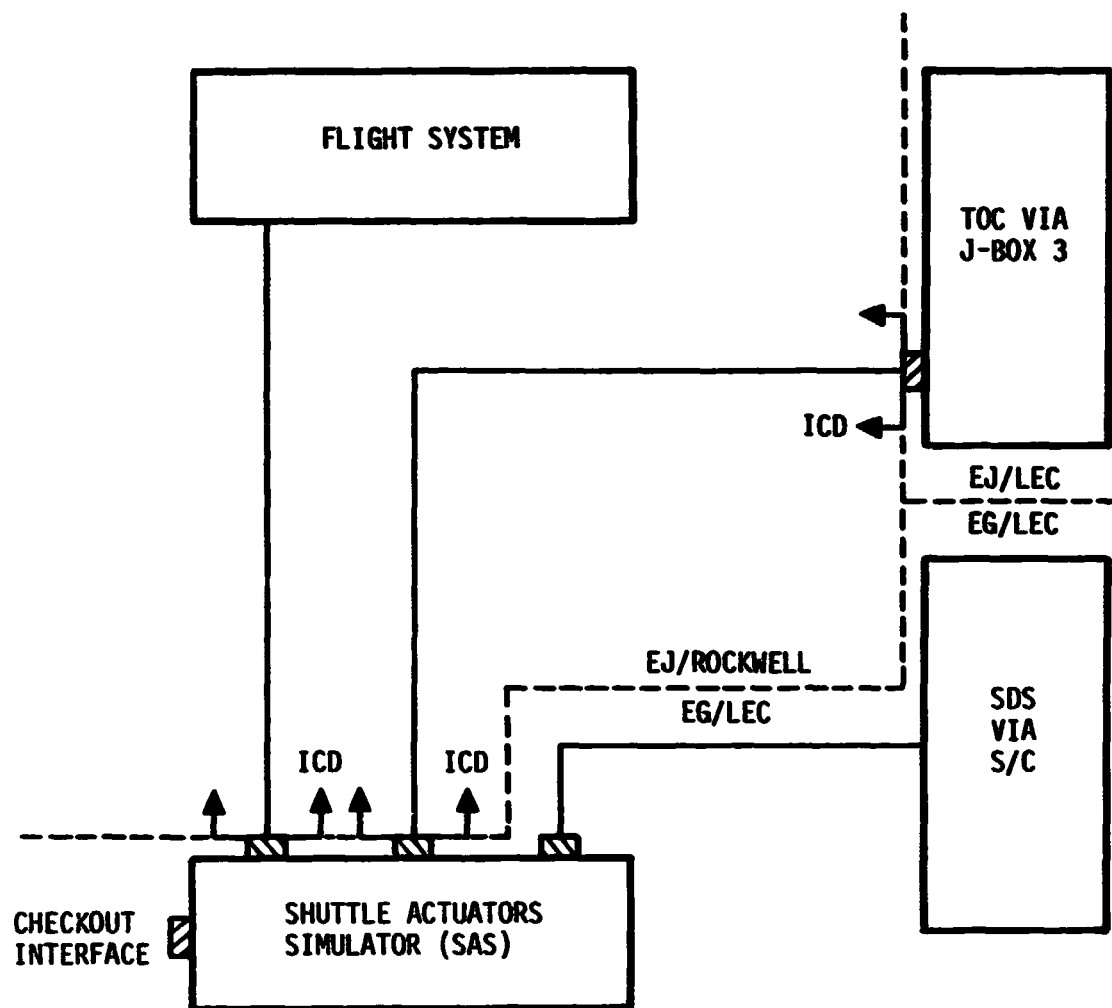
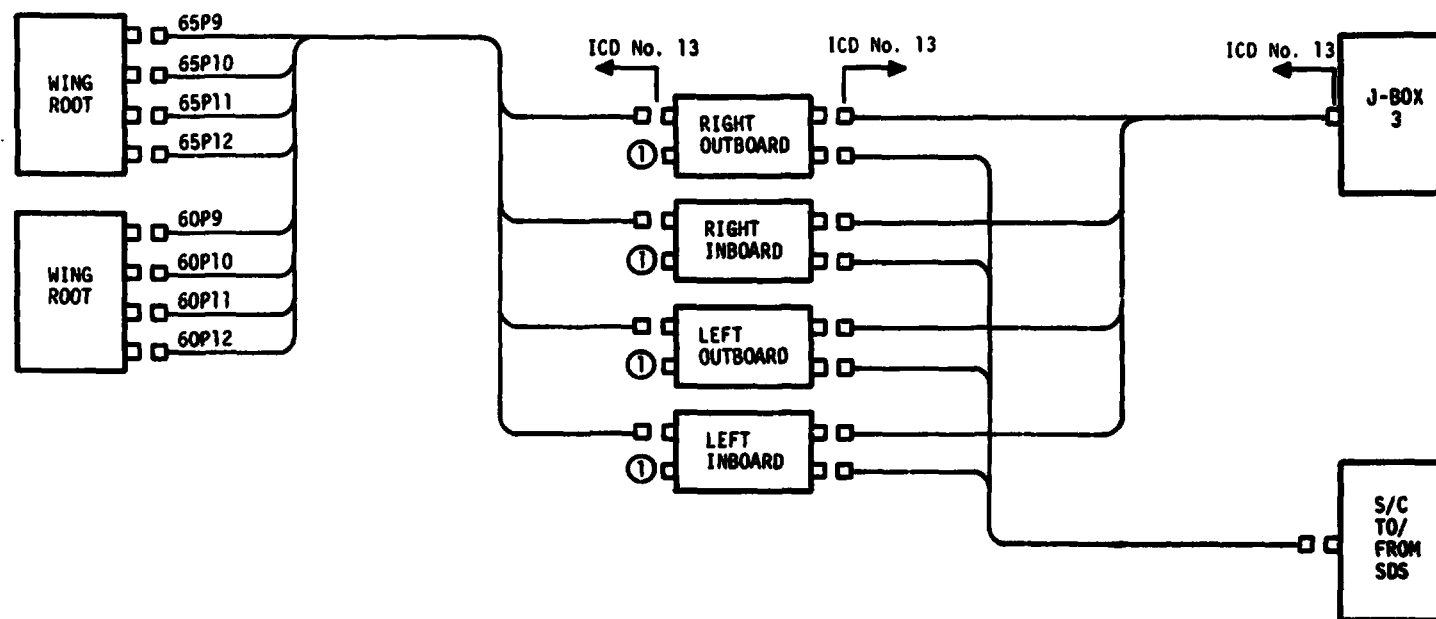


Figure 3-1. - Overall interface diagram.



NOTES: ① TEST CONNECTOR

Figure 3-2. - Cable requirements.

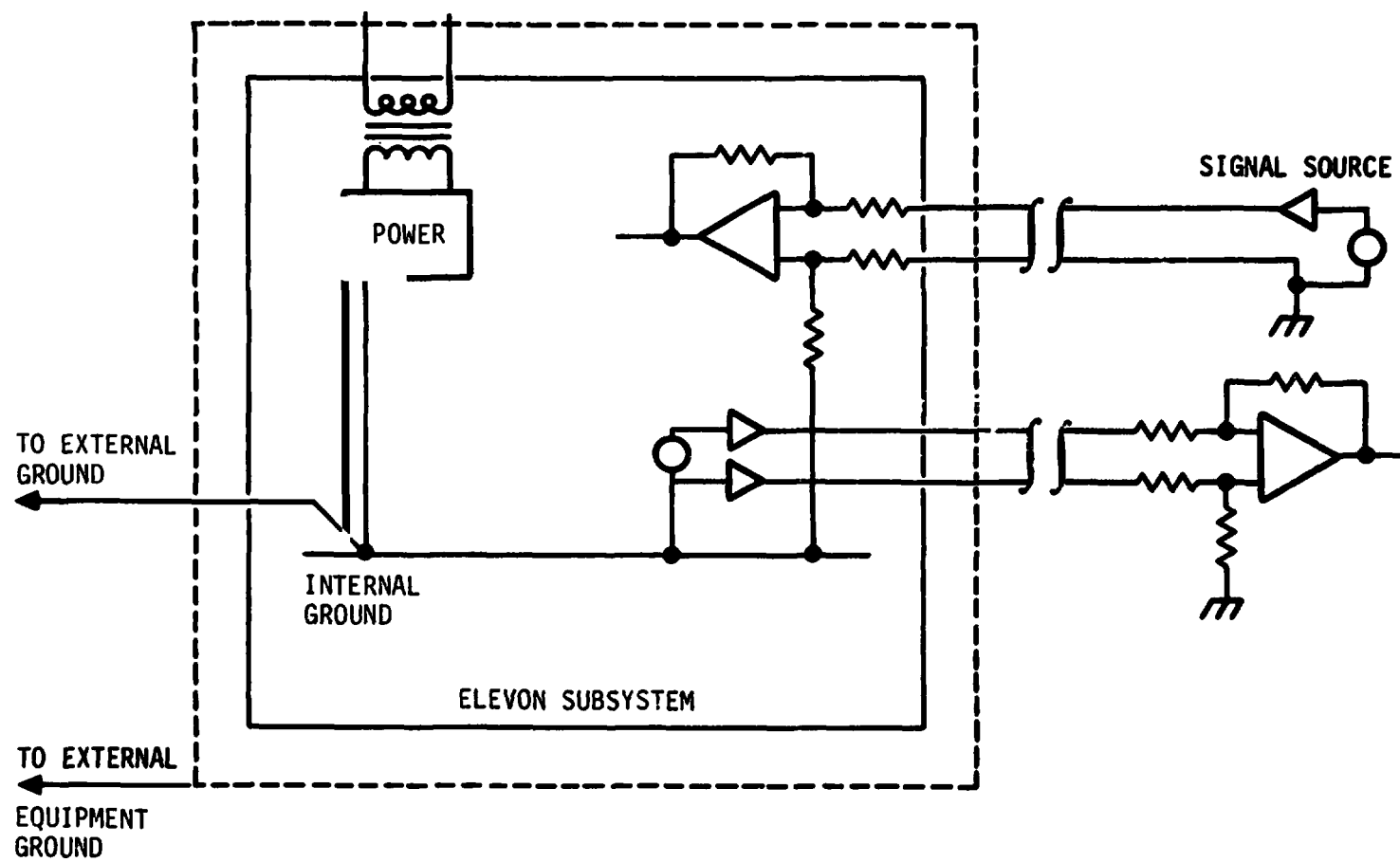


Figure 3-3. - Grounding.

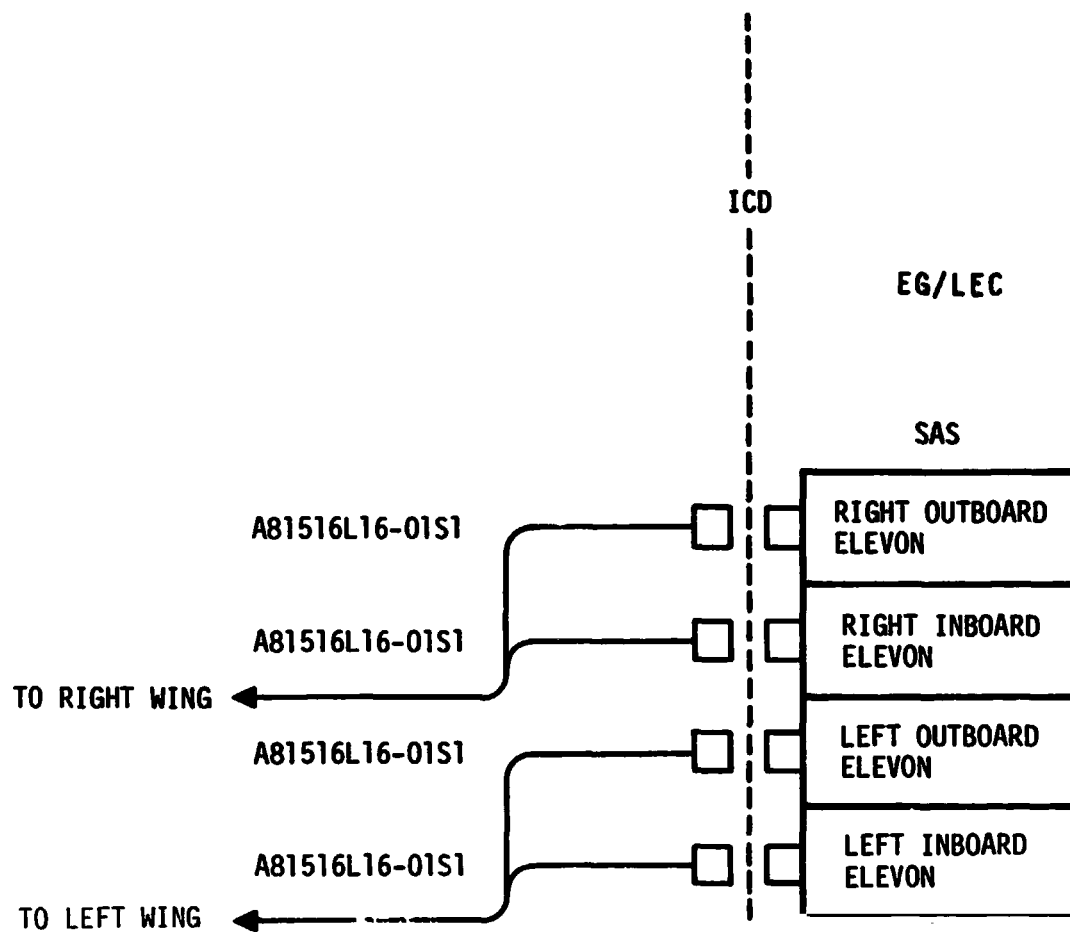


Figure 3-4. - ASA/SAS interface (connectors).

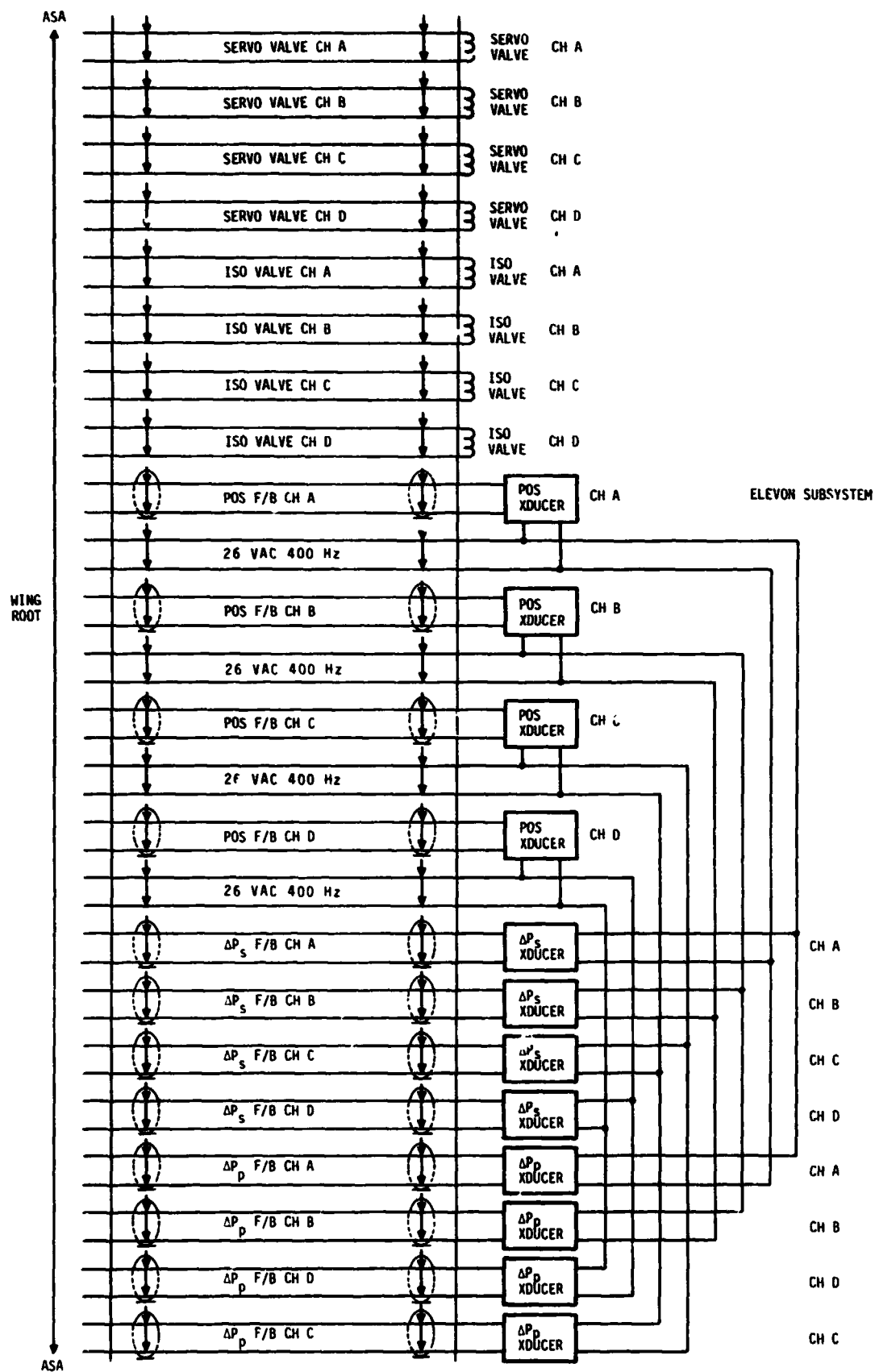


Figure 3-5. - Elevon actuator subsystem.

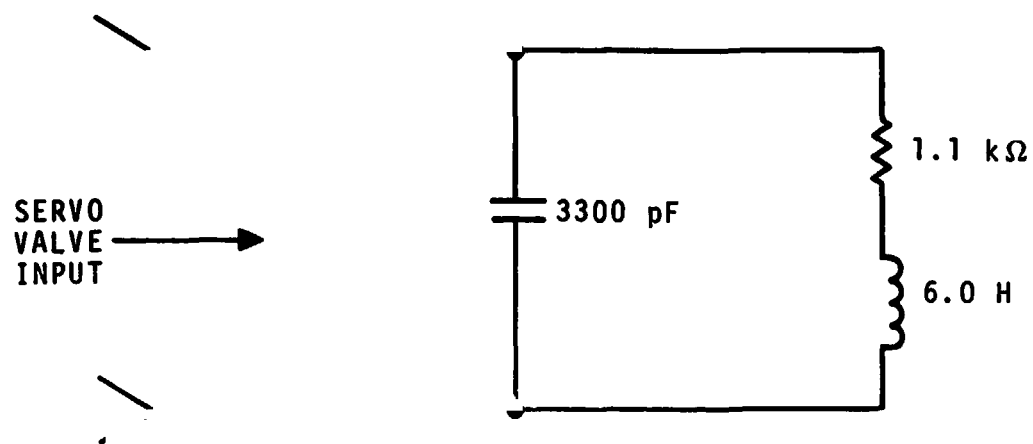


Figure 3-6. - Servo valve.

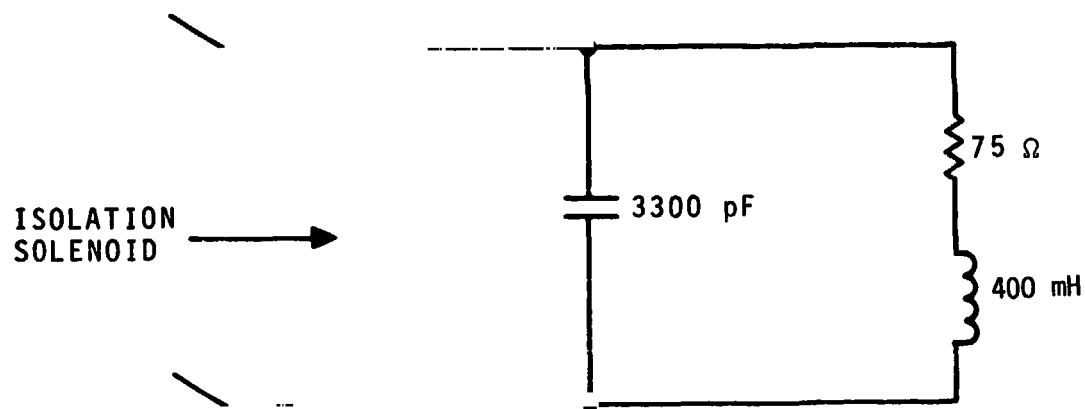


Figure 3-7. - Isolation valve.

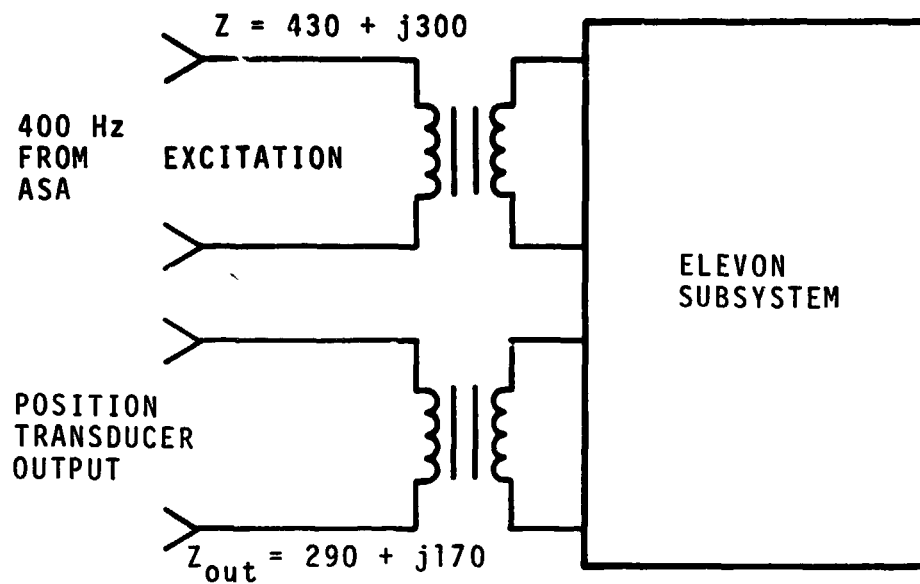


Figure 3-8. — Position transducer.

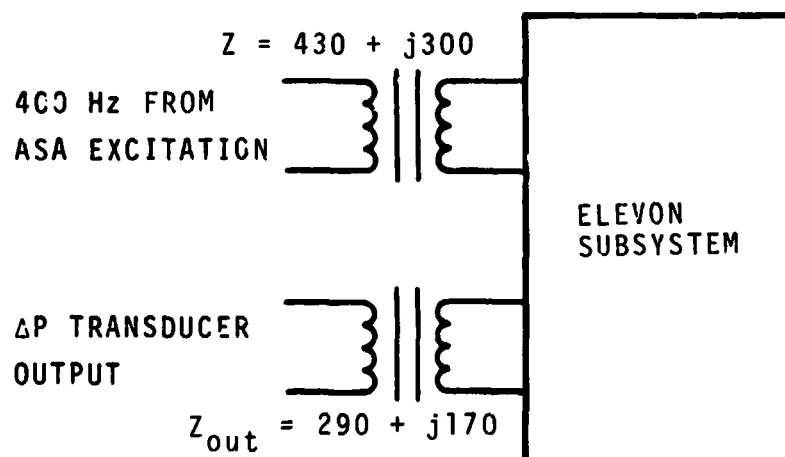


Figure 3-9. — ΔP transducer.

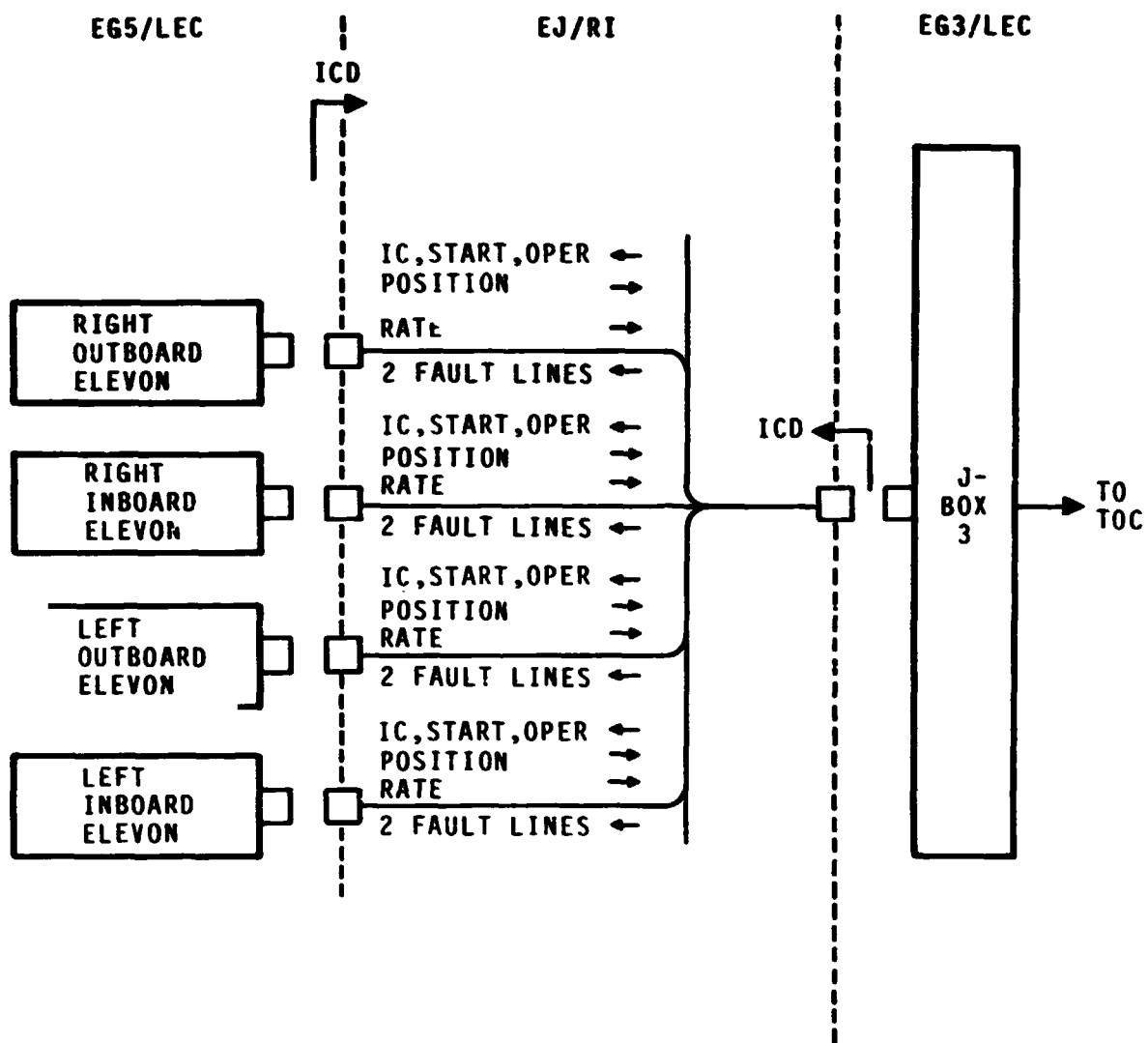


Figure 3-10. - SAS/J-box 3 interface.

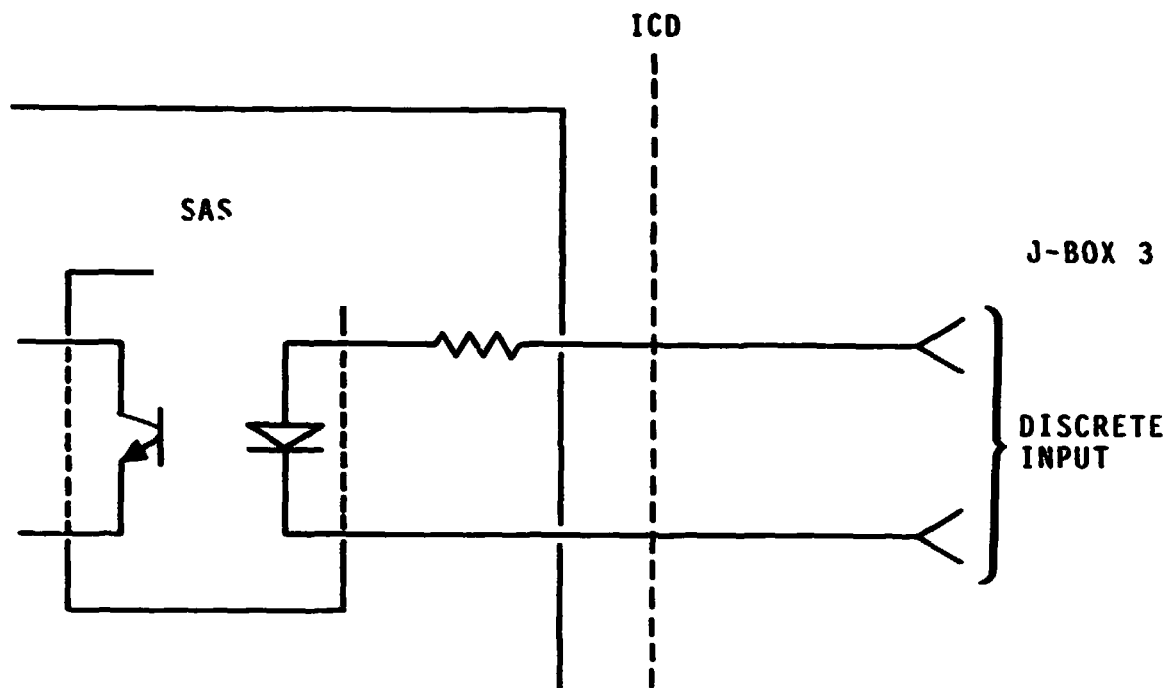


Figure 3-11. - Discrete input to SAS from J-box 3.

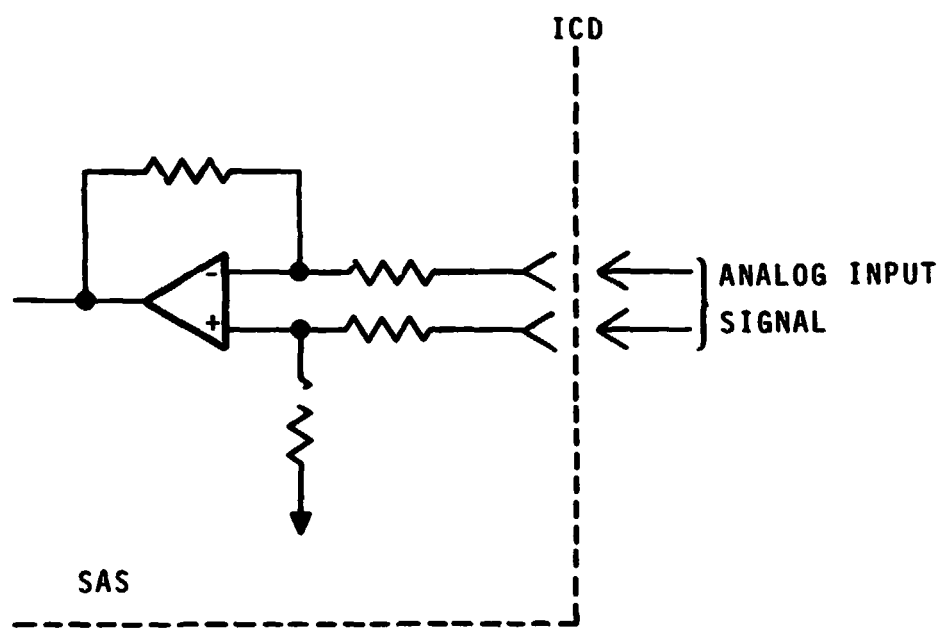


Figure 3-12. - Analog input to the SAS from J-box 3.

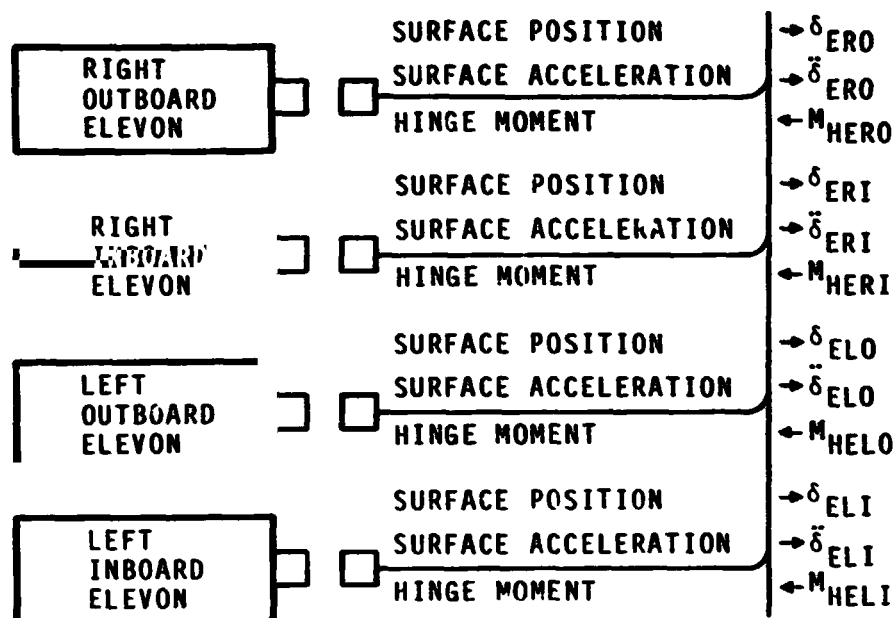


Figure 3-13. - SAS/SDS interface.

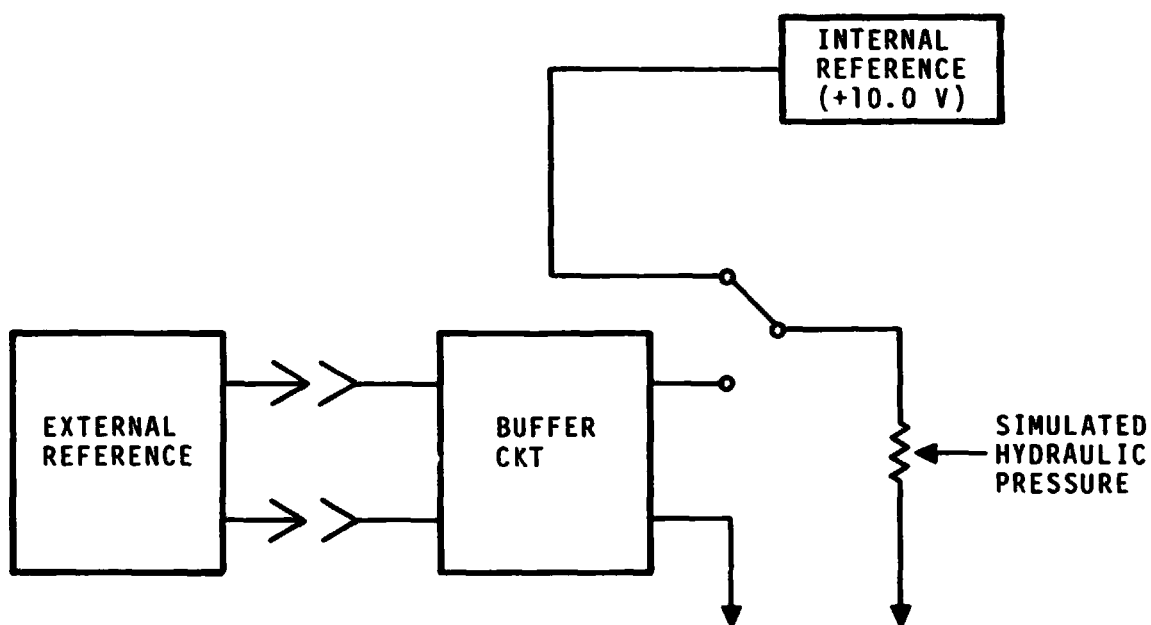


Figure 3-14. - System hydraulic pressure.

4. FAULT INSERTION

The capability to insert faults into the elevon is limited to those faults which will be detected by the redundancy management system. Faults which affect only elevon performance and are not detectable by the redundancy management system will not be implemented. Because faults are detected only via the elevon feedback to the ASA, any detectable fault will result in a significant change in some feedback parameters.

4.1 GENERAL

Fault insertion capability will be provided so that two faults may be remotely inserted in groups of two, with one fault labeled or designated as Fault Set 1 and the other as Fault Set 2. The elevon subsystem will contain manual switching circuitry to allow selecting:

- Faulted channel (see figure 4-1)
- Fault location
- Fault type
- Fault order

Fault capability will be as follows:

- ΔP Primary feedback (four channels)
- ΔP Secondary feedback (four channels)
- Position feedback (four channels)
- Isolation (ISO) valve (four channels)

The types of faults to be provided are \pm hardover, zero, and open (ISO valve only).

To have a well defined identification of each fault, a maximum of two sets of faults will be located and classified at a given time. Faults will be distinguished as Fault 1 or Fault 2.

All fault request input data will be entered locally at the SAS by positioning switches on the fault insertion panel as depicted in figure 4-1.

Using one or both of two possible fault selection lines, TOC may initiate fault sets remotely as required. The order of the selected fault set level will be selected by TOC. It must be manually selected at the actuator subsystem.

Secondary faults will not be allowed. A secondary fault is defined as one which exists as a result of a previous fault.

4.2 CONTROL INTERFACE

4.2.1 SUBSYSTEM LINES

Two lines will be provided from the TOC for each elevon. One line will correspond to Fault Set 1 and the other to Fault Set 2.

4.2.2 LOGIC "Ø" STATE

A no-fault condition will be defined as a logic "Ø" state. The fault will be activated by setting the desired fault line to a logic "1" state. The ISO valve fault will require manual operation of the ISO valve switch at the selected elevon subsystem.

4.2.3 CLEARING FAULTS

Clearing of faults may be accomplished by:

- Setting the fault line back to logic "Ø" from TOC
- Setting the order switch to "no-fault" or resetting the master fault enable switch

- Resetting the ISO valve switch to normal (for the ISO valve fault only)

4.2.4 MASTER FAULT ENABLE SWITCH

A lockout switch to prevent inadvertent fault insertion will be provided. This will be implemented with a master fault enable switch on the subsystem in which the fault is to be inserted. This switch must be set to "enable" before any faults can be initiated. This switch will not affect operation of the ISO valve fault switch.

4.2.5 FAULT INSERTION LOGIC

Figure 4-2 shows schematically how faults will be inserted into the elevon subsystem. All faults except the ISO valve fault will be inserted into the appropriate circuits.

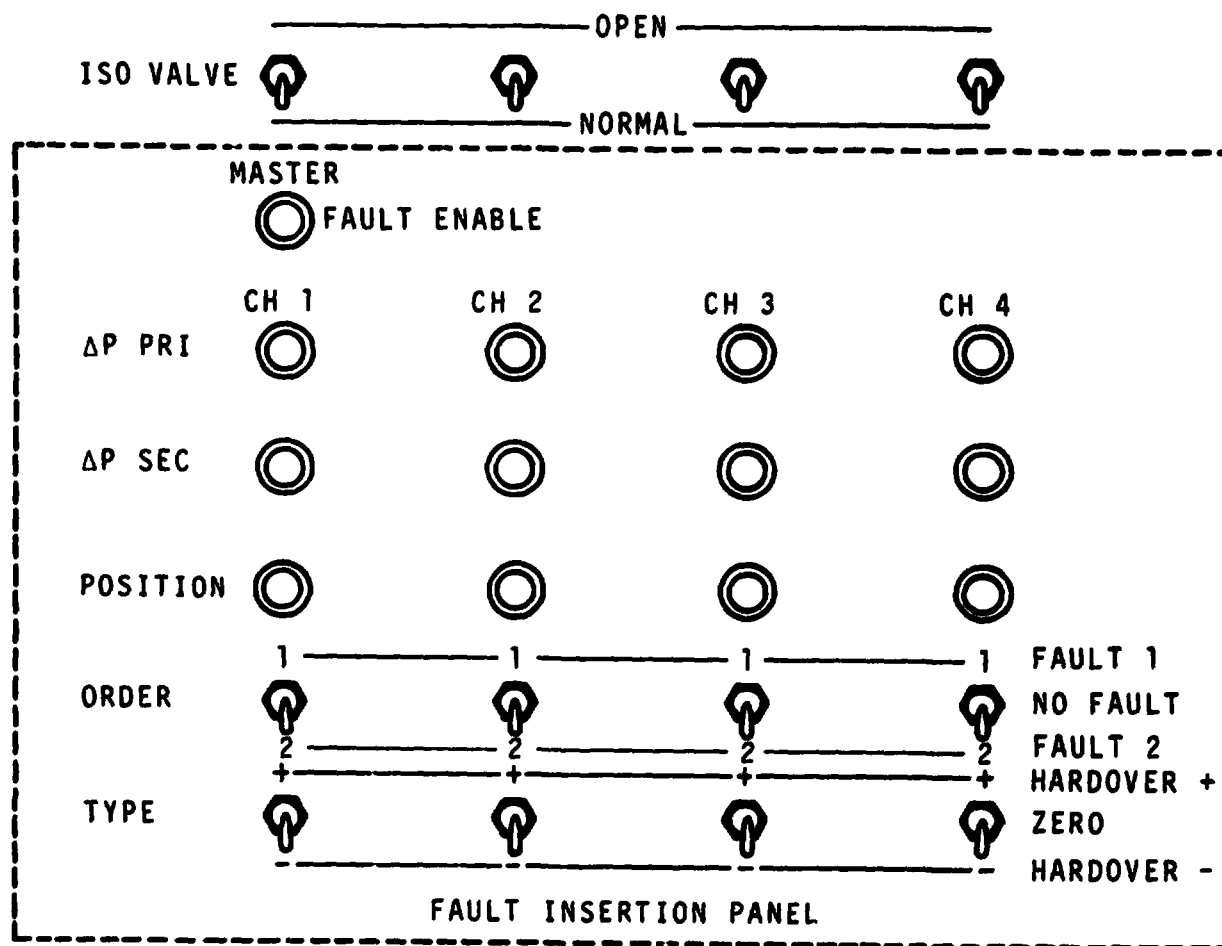


Figure 4-1. - Typical fault insertion panel.

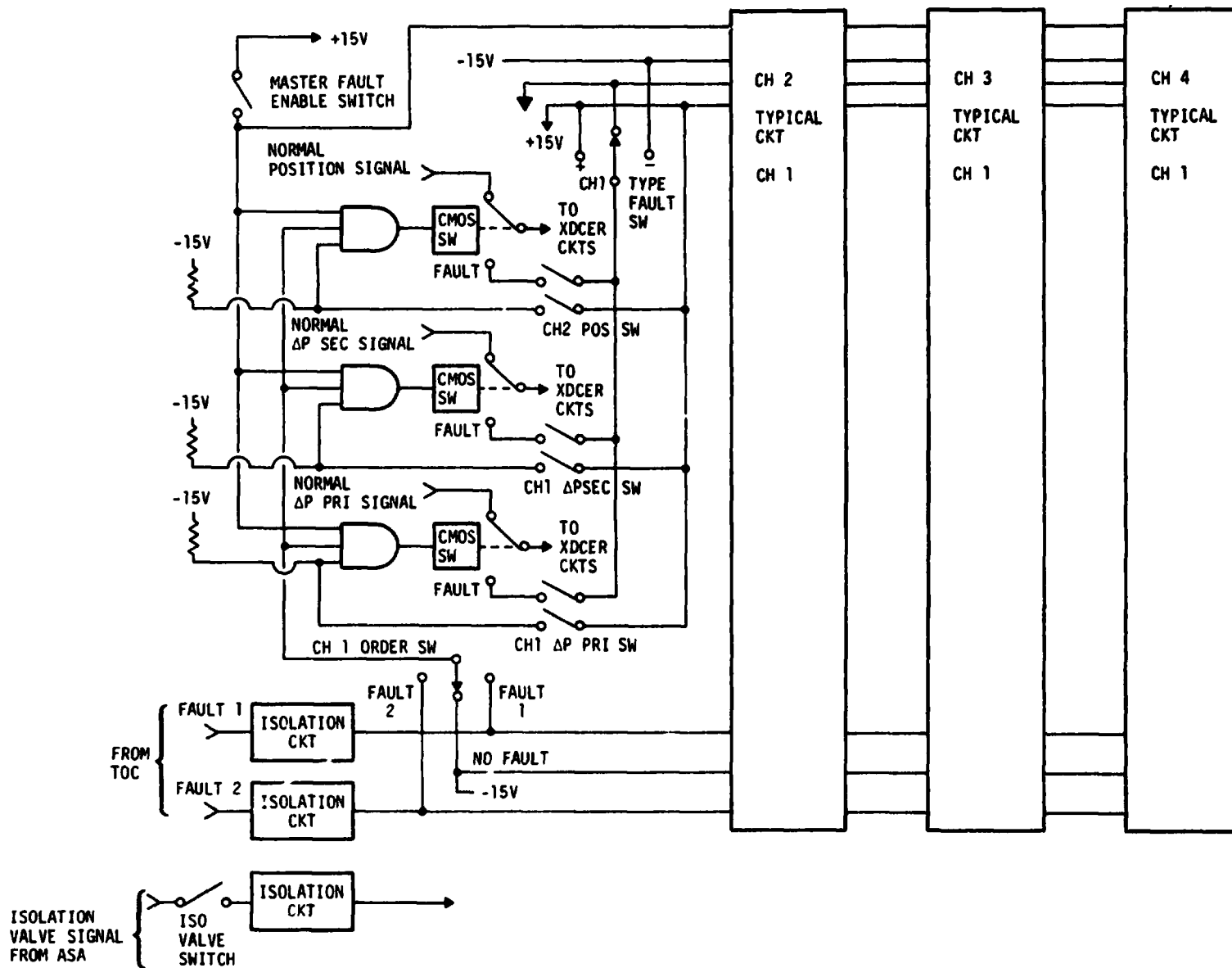


Figure 4-2. - Fault insertion logic.

5. MAINTENANCE

5.1 MAINTAINABILITY

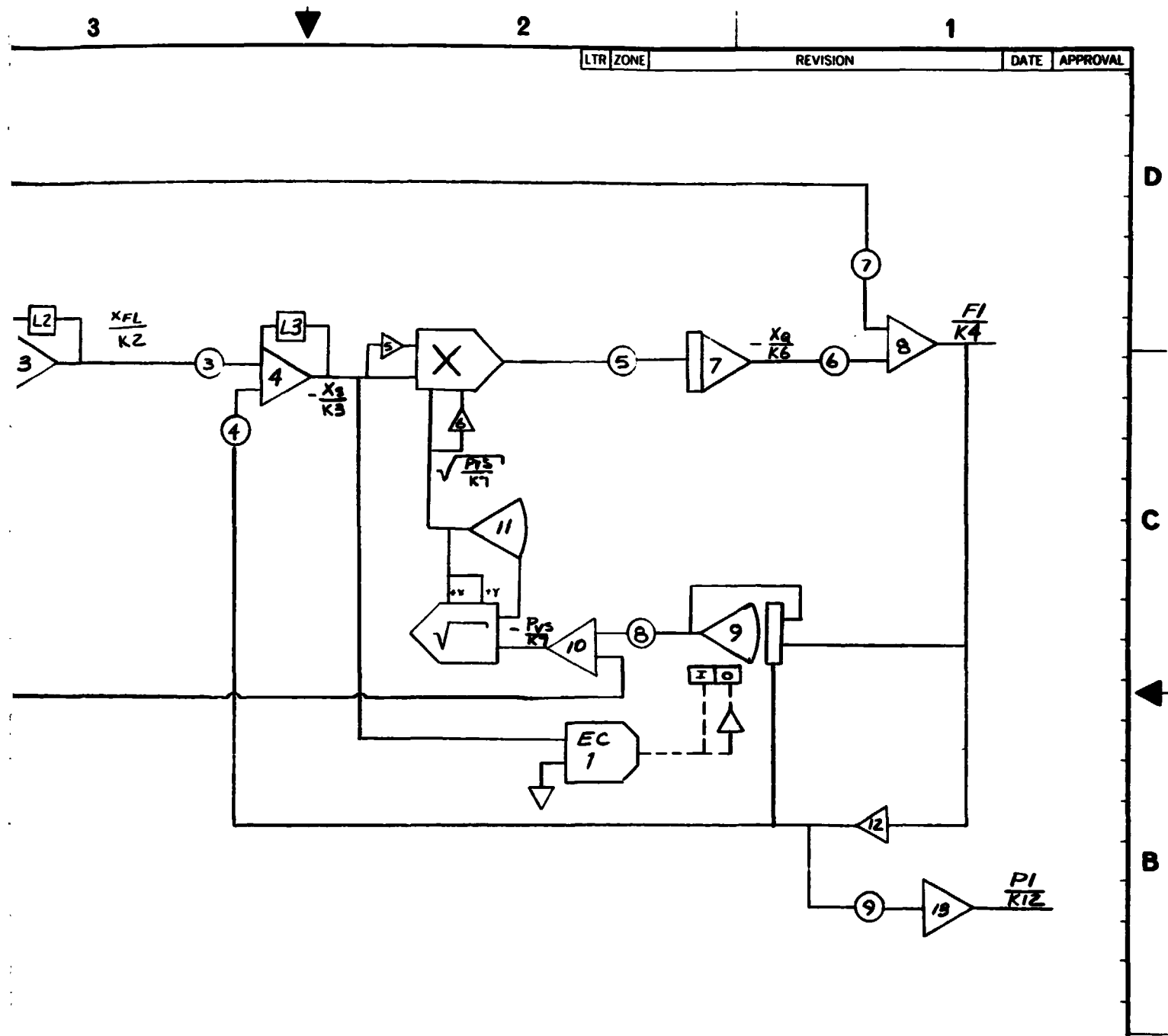
The elevon will be partitioned on a function block basis to facilitate maintenance. Recommended spare boards and parts will be provided at the completion of the detailed design phase.

5.2 INTERCHANGEABILITY

- a. Subsystems will not be interchangeable except for like units (inboard for inboard, outboard for outboard).
- b. PC cards will be interchangeable on an identical function basis within a subsystem. Some cards will be interchangeable across subsystem lines (e.g., mod piston driver, buffer cards). See figure 5-1.



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PART NUMBER	DESCRIPTION	MATERIAL	SPECIFICATION
<div style="display: flex; justify-content: space-between;"> <div> <p>DIMENSIONAL TOLERANCE UNLESS NOTED OTHERWISE</p> <p>0 - 10</p> <p>00 ± 02</p> <p>000 ± 005</p> <p>ANGULAR °</p> </div> <div> <p>SIGNATURES</p> <p>DR</p> <p>ENG <i>J. BARR</i></p> <p>CH</p> </div> <div> <p>DATE</p> <p>11/75</p> </div> </div>			
<p>NATIONAL AERONAUTICS & SPACE ADMINISTRATION</p> <p>LYNDON B. JOHNSON SPACE CENTER HOUSTON, TEXAS</p>			
<p>POWER SPOOL DRIVER</p>			
<p>SURFACE FINISH IN MICROINCHES RMS UNLESS NOTED OTHERWISE</p> <p>✓</p>		<p>CODE IDENT NO</p> <p>21356</p>	
<p>APP</p> <p>AUTH</p>		<p>SIZE</p> <p>C</p>	
<p>SCALE</p>		<p>DWG NO</p>	
<p>NEXT ASSEMBLY</p>		<p>SHEET</p>	

Figure 5-1. - Mod piston driver.

6. SUBSYSTEMS INITIALIZATION

The elevon subsystems will be initialized by either the Flight Systems software via the multiplexer-demultiplexer (MDM) or by the TOC. Initialization from the TOC will place an initial condition voltage on an integrator in each elevon actuator subsystem. The "operate" signal will be from the mode control via the TOC/J-box 3. The initial condition signal, IC, will be inserted into the integrator as shown in figure 6-1. See the discussion under section 3.3.2.

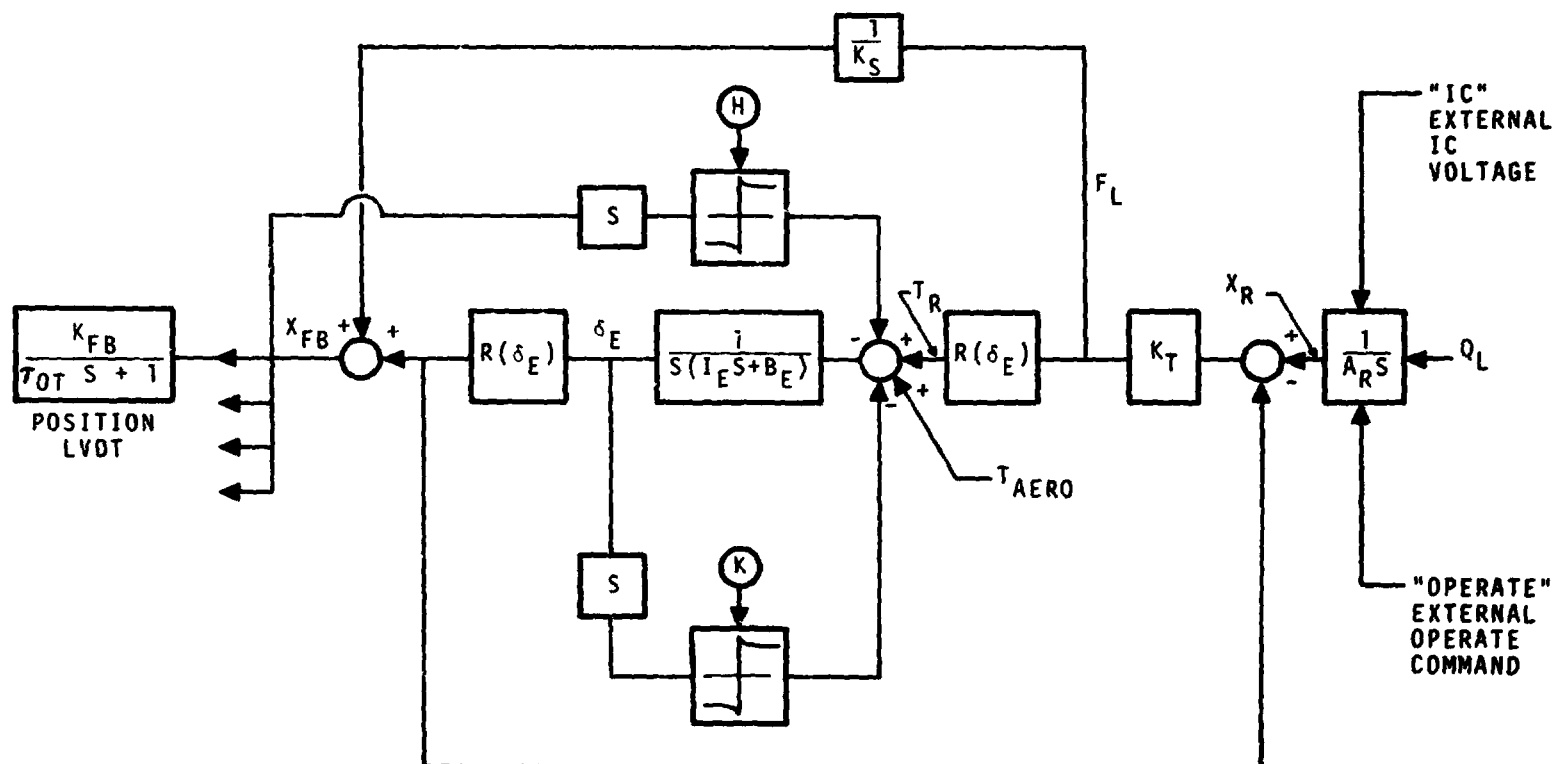


Figure 6-1. - Subsystem initialization.

7. DETAILED CIRCUIT DESIGN

The electronic circuitry shown in this section includes the interface circuits to the ASA, to TOC via J-box 3, and to SDS via the Shuttle Interface Subsystem (SIS). Also included are the schematics for the pressure/position transducer simulator actuator and part of the fault insertion circuitry.

7.1 GENERAL REQUIREMENTS

The interface circuitry will be designed to meet the signal requirements of the system/subsystem with which it is interfacing. These include transformer-coupled signals, buffered differential output signals, high-input impedance input amplifiers, and optical isolators. The signal levels and characteristics are discussed in section 3.

7.2 INTERFACE CIRCUITRY

7.2.1 HINGE MOMENT INTERFACE CIRCUIT

The hinge moment will be supplied to the SAS from SDS via SIS. The interface circuit is shown in figure 7-1. The signal will be a differential ± 5 V. The differential signal will be received at the SIS/SAS interface with two unity-gain follower circuits, converted to ± 5 V single ended, then amplified to ± 10 V as required by the SAS.

7.2.2 INITIALIZATION INTERFACE CIRCUIT

The initialization signal will originate in the TOC as a 0.0 to 5.0 V full scale analog signal. The SAS circuitry requires ± 10 V full scale, so the interface circuit will include the level translation as well as the isolation circuit. The circuit is shown in figure 7-2. The input will be received with two voltage follower circuits, then go to a differential input, single-ended output, unity-gain circuit. The signal will then be offset and amplified to ± 10 V full scale single-ended.

7.2.3 RATE/POSITION INTERFACE

The rate/position interface to TOC will require level translation from ± 10 V full scale to 0.0 to 5.0 V full scale. The circuit is shown in figure 7-3. The first amplifier will attenuate and translate the signal to the proper level. It will also invert the signal so a unity-gain inverter will be added to achieve the proper polarity.

7.2.4 TYPICAL TRANSFORMER ISOLATION INTERFACE

Transformer isolation will be used in the transducer interface circuits. The 400 Hz reference from the ASA will be transformer coupled into the SAS. Position feedback, ΔP_s , and ΔP_p will be transformer coupled out of SAS to the ASA. Figure 7-4 shows a simple block diagram of the circuit. Figure 7-5 (SAS1003S) shows the Pressure/Position Transducer Simulator Actuator schematic. The circuit will include a provision for fault insertion through a complementary metal oxide semiconductor (CMOS) switch U-4.

7.2.5 ELEVON SERVO VALVE INTERFACE

The elevon servo valve interface will receive the servo drive current from the ASA. The input signal will be a current loop of approximately ± 8.6 mA. The voltage developed across the 470-ohm resistor will be applied to the input of a high-impedance amplifier (see fig. 7-6). The output will then feed the SAS. A schematic (SAS1004S) of the circuit board with four channels is shown in figure 7-7.

7.2.6 ELEVON ISOLATION VALVE INTERFACE

The elevon isolation valve interface circuit is shown in figure 7-8. The signal will originate in the ASA; it is referred to as a dc command signal in the Interface Control Document (ICD). It will be a nominal 28 V signal. It will be optically isolated

from the SAS to maintain the grounding isolation as required by the SAIL program. See figure 7-9.

7.2.7 POSITION/ACCELERATION INTERFACE

The position and acceleration interface is shown in figure 7-10. The signal flow will be from SAS to SDS via the SIS. The SIS interface will require ± 5 V full scale differential. The interface circuit will attenuate the ± 10 V single-ended signal from SAS to the required differential level.

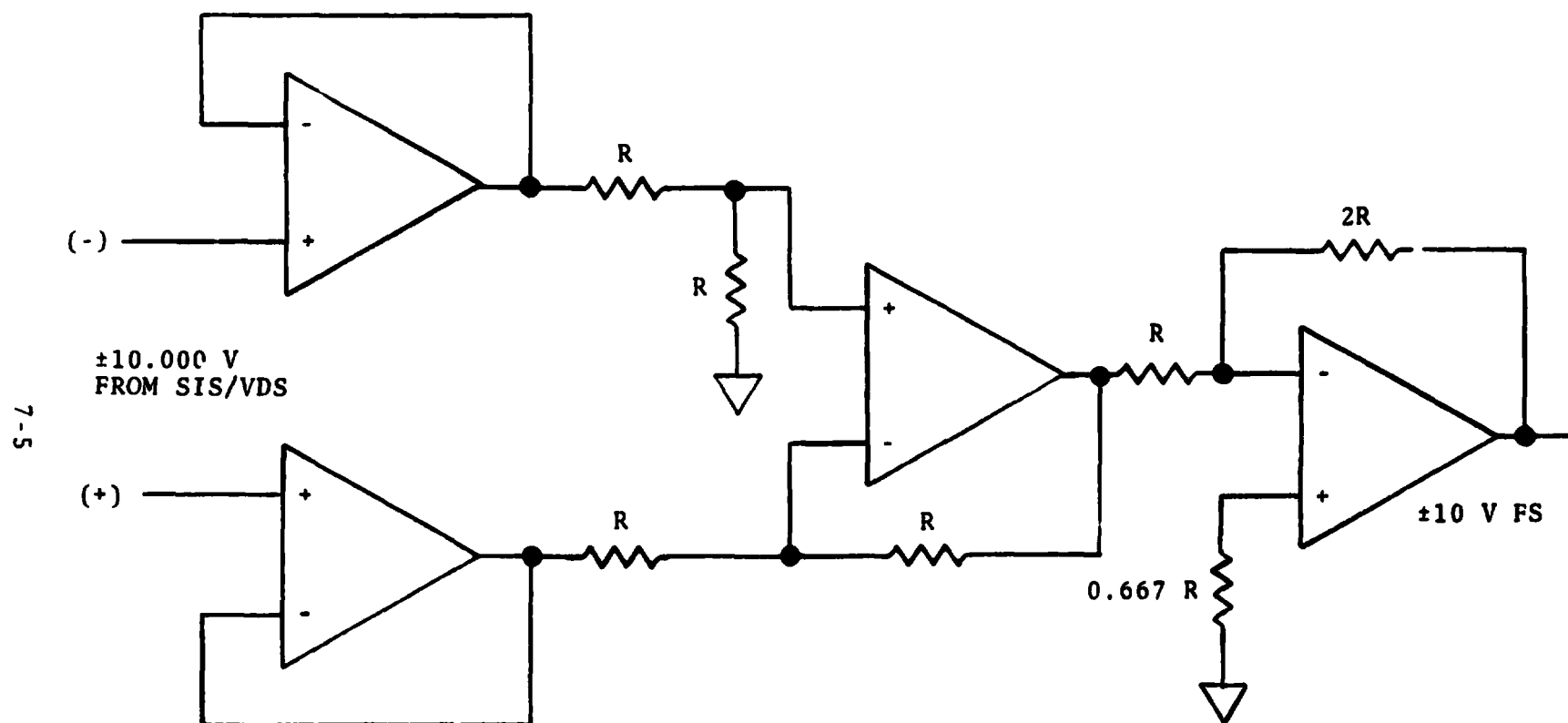
7.2.8 OPTICAL-ISOLATOR INTERFACE

The optical-isolator interface circuit is shown in figure 7-11 for the mode/fault insertion control signals. This type of circuit will be used throughout the SAS for interfacing input signals of a discrete nature. The mode control circuit is shown in figure 7-12. When TOC initiates the start discrete, the "Q" output of the flip-flop will go high, which in turn will close the switches for initialization (not shown). The circuits will remain in this initialization state until TOC sends an operate signal. At that time the flip-flop will be reset, the "Q" output will go low, and the initialization switches will be switched back to the operate mode.

7.2.9 FAULT INSERTION

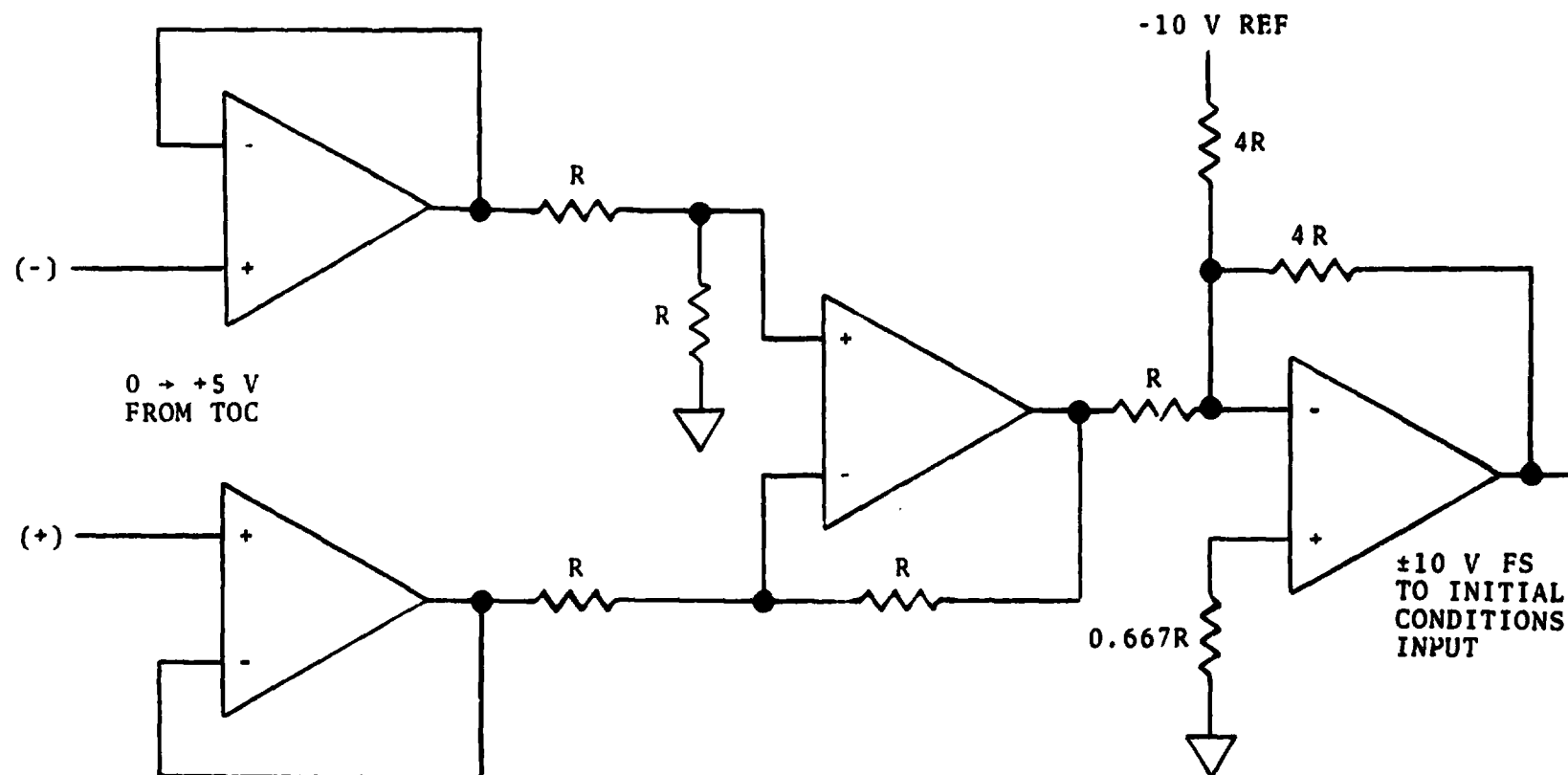
The fault insertion is schematically shown in figure 7-13. The fault insertion control circuits are shown in figures 7-14 and 7-15. There will be two fault types, Fault 1 and Fault 2, initiated by TOC. Fault switching will be set up manually by toggle switches on the SAS front panels. Then the system will be ready to be initiated by Fault 1 or Fault 2. There will be separate switches for each of the four channels per elevon. This will allow all four channels to have either Fault 1 or Fault 2 initiation exclusive of which fault enables the other

channels. Faults will be inserted into ΔP_p , ΔP_s , and position either individually or together as chosen by the manual switching. However, in a given channel, only hardover (+), hardover (-), or zero will be allowed in a given fault setup.



UIC

Figure 7-1. - Hinge moment SIS/SAS interface.



UIC

Figure 7-2. - Initialization TOC/ASA interface.

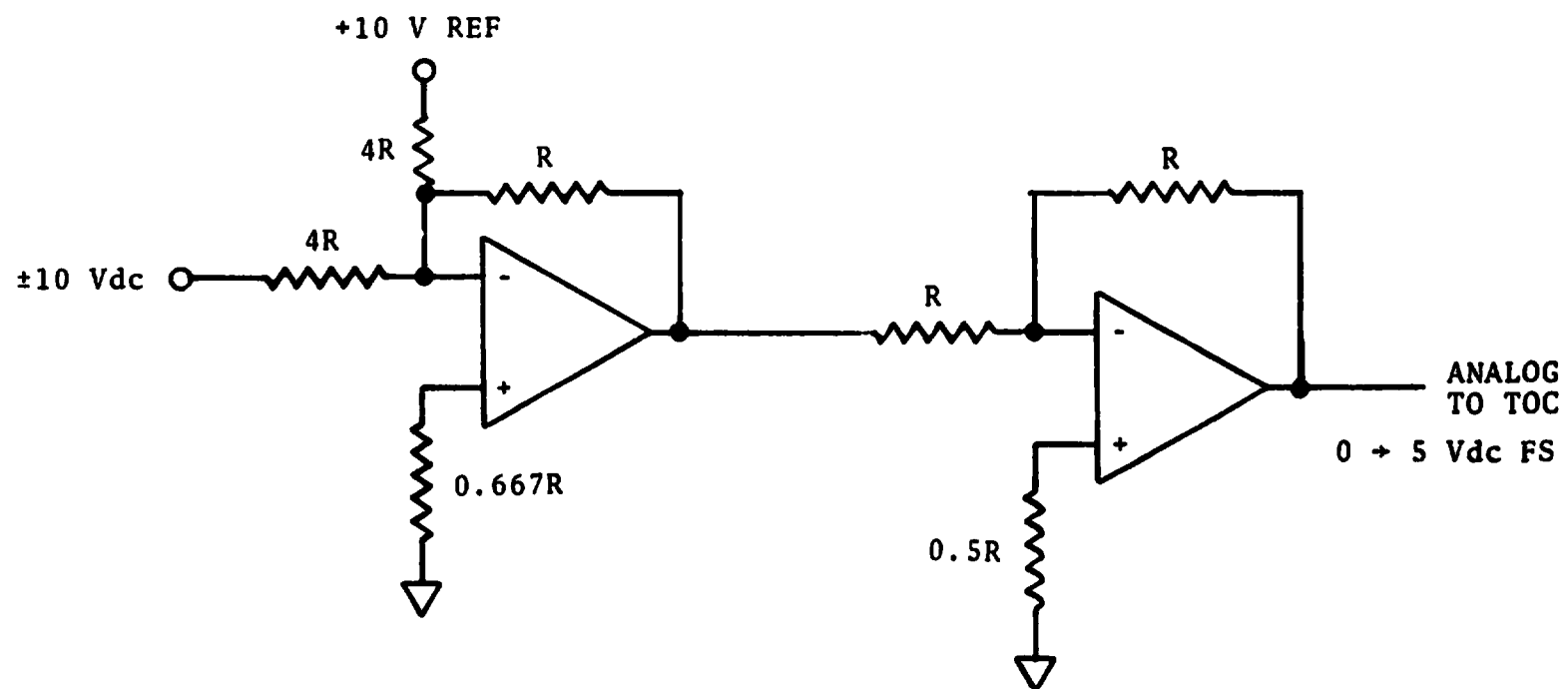


Figure 7-3. - Rate position SAS/TOC interface.



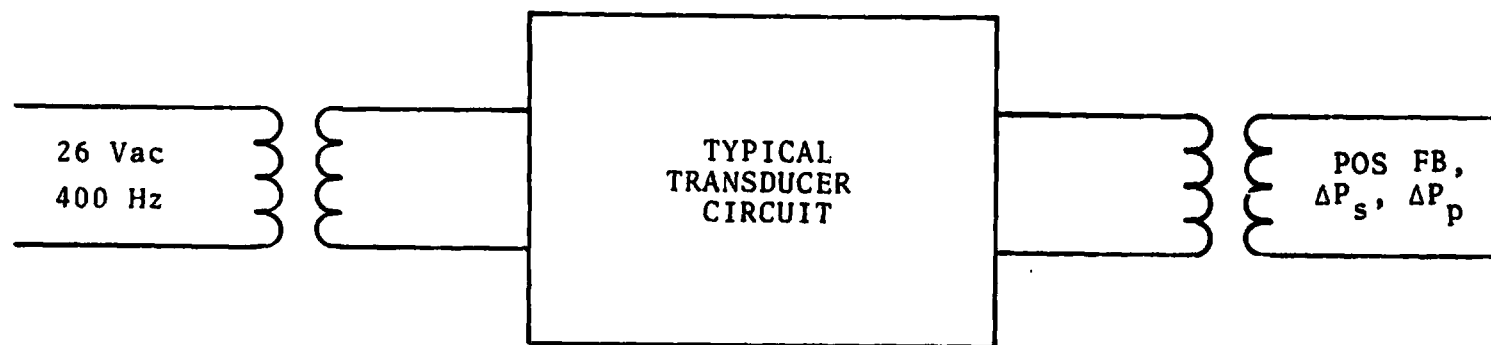


Figure 7-4. - Typical transformer - coupled isolation circuit ASA/SAS. **LEC**

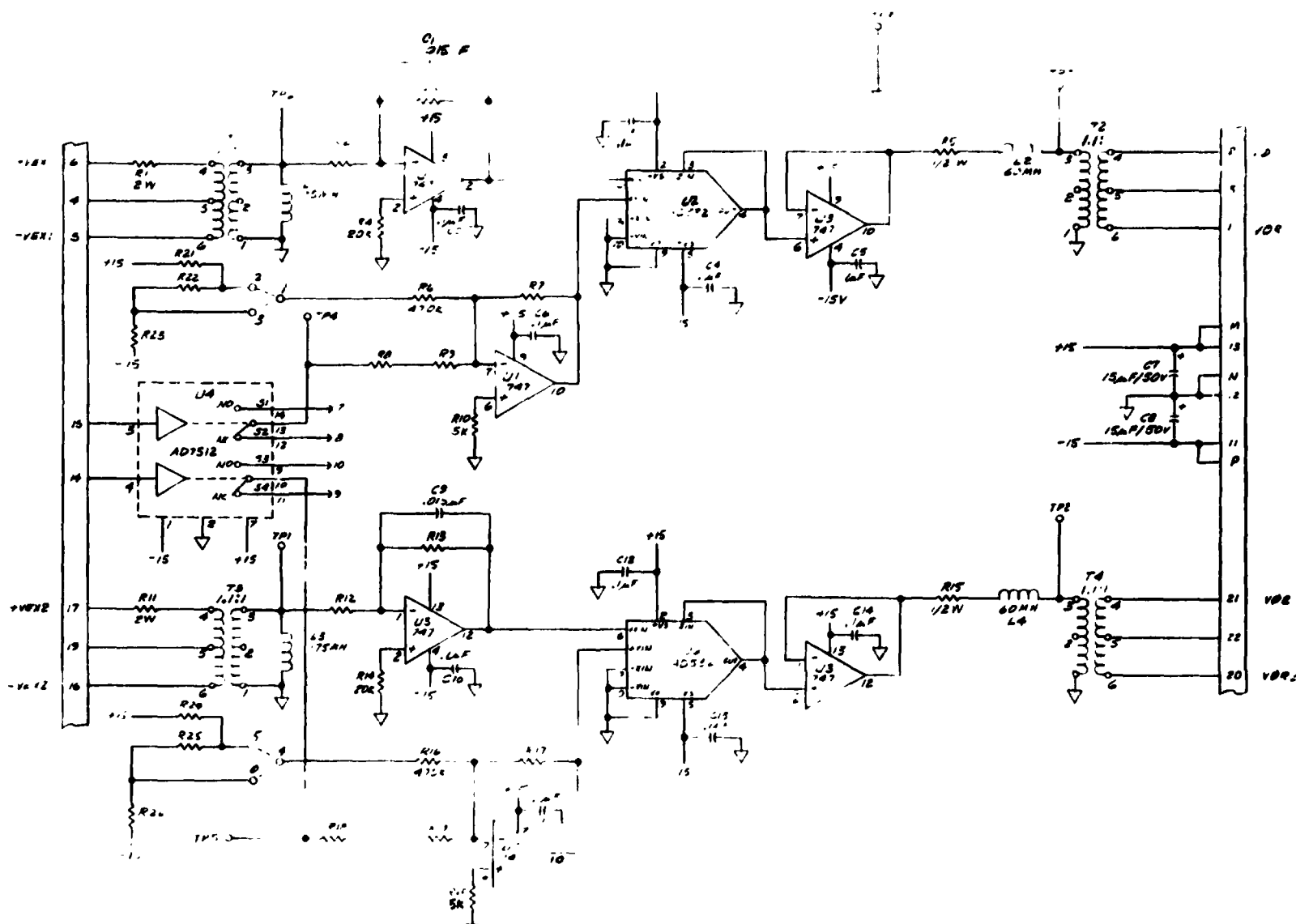


Figure 7-5. — Pressure/Position Transducer
Simulator Actuator Subsystem.

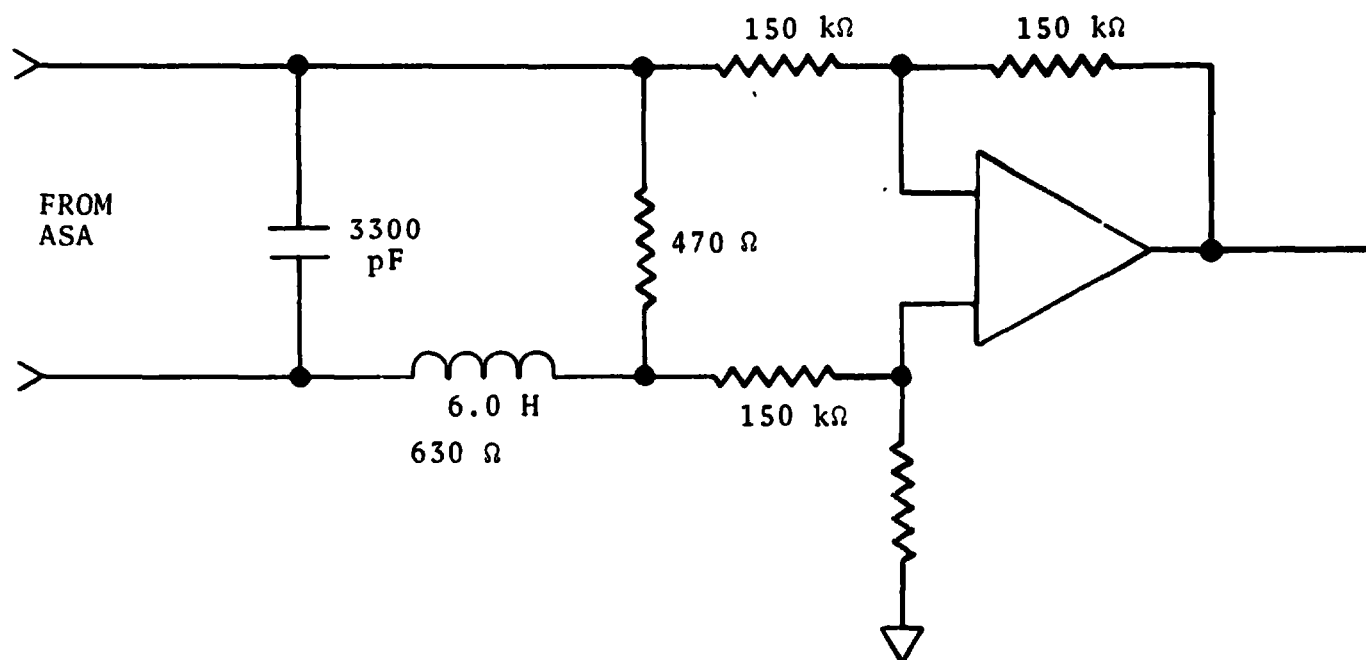


Figure 7-6. – Elevon servo value ASA/SAS interface

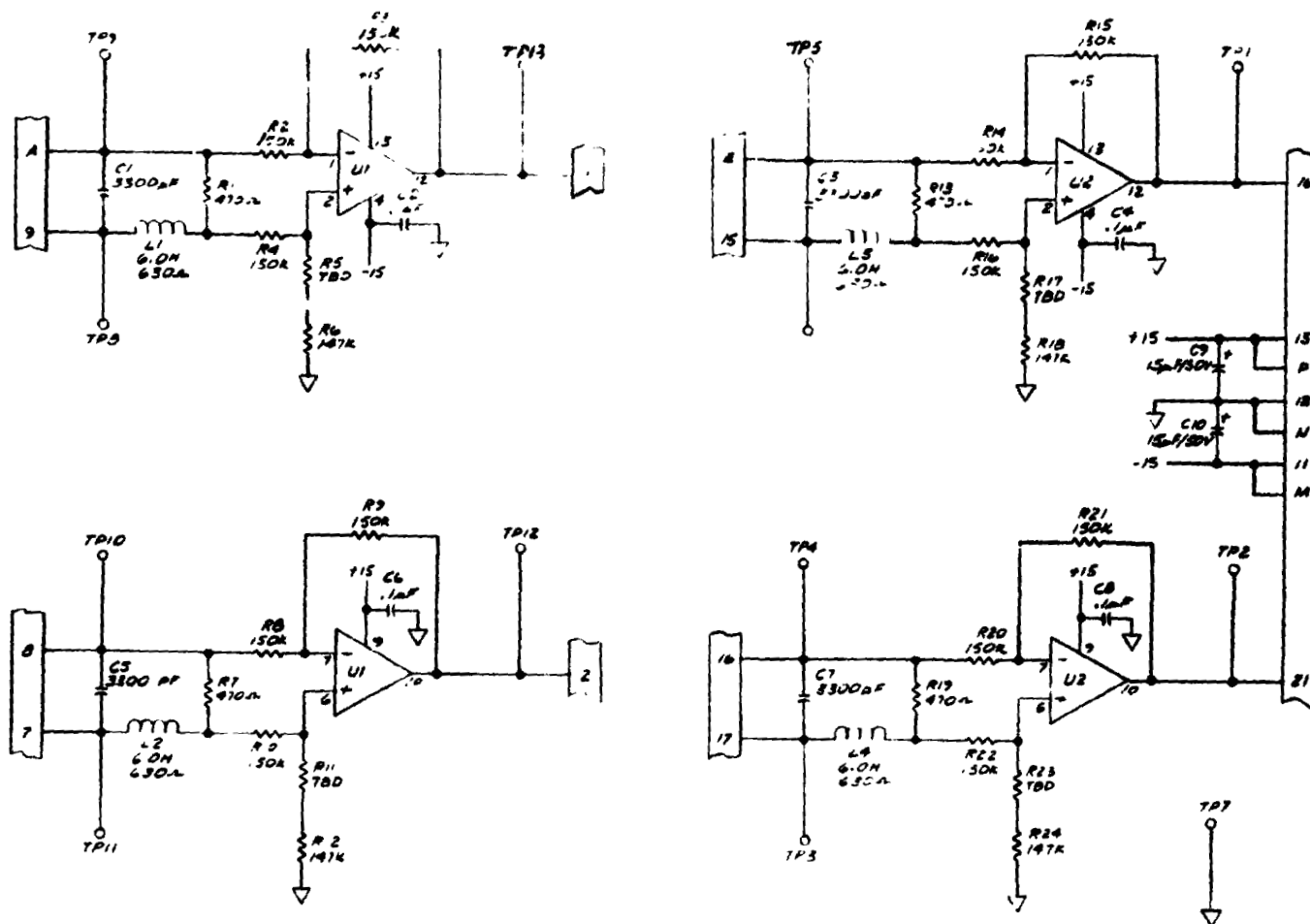
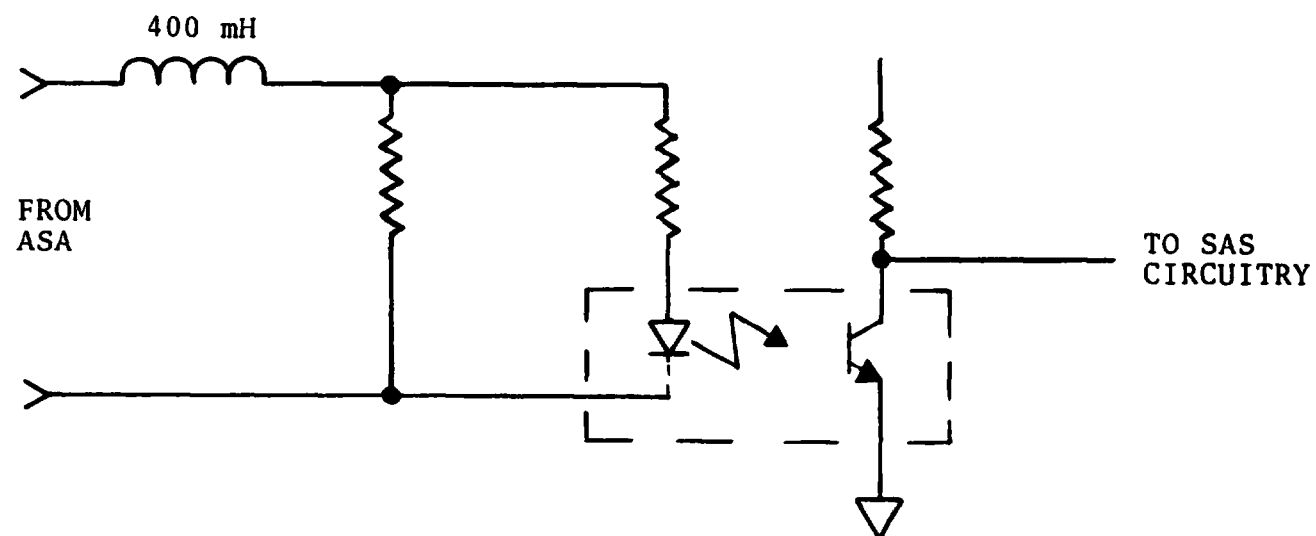


Figure 7-7. - Servo Valve Interface
Simulator Actuator Subsystem.



LEG

Figure 7-8. - Elevon isolation valve ASA/SAS interface

7-13

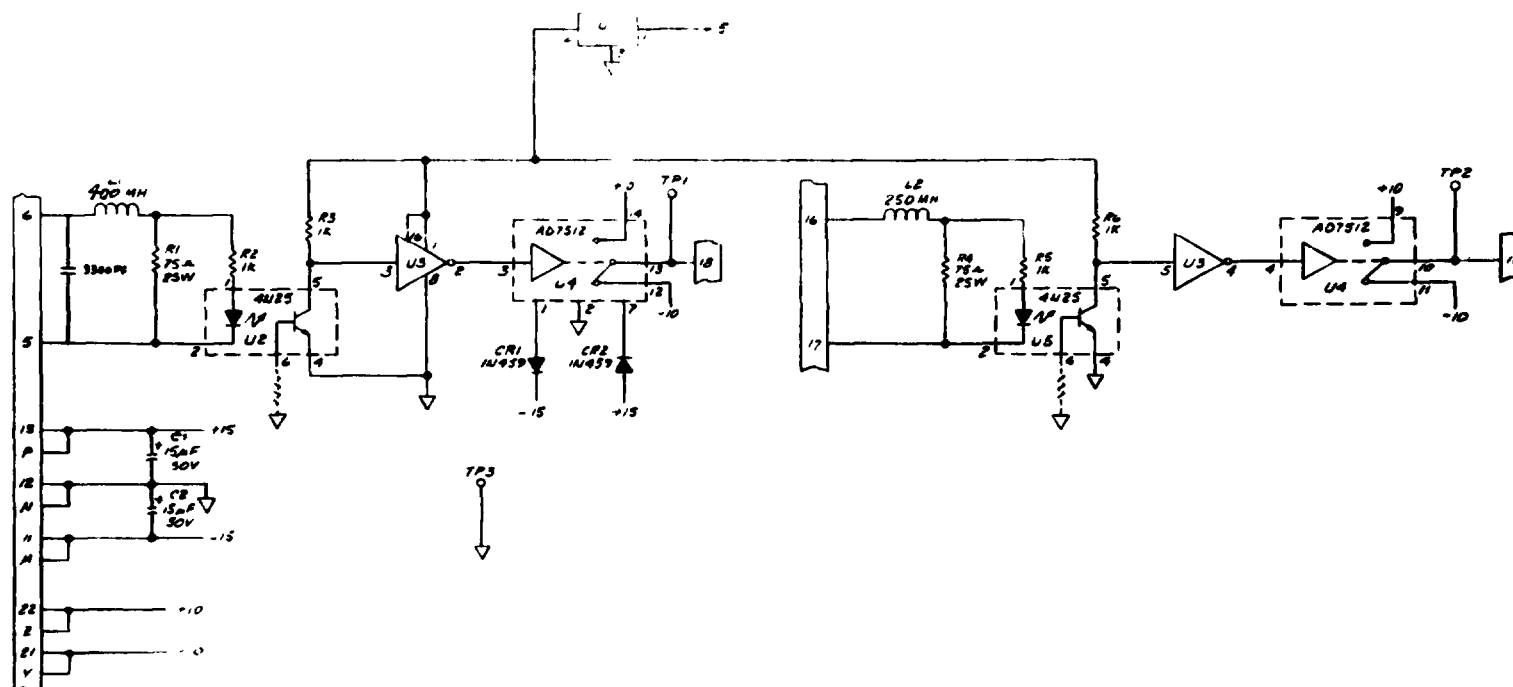
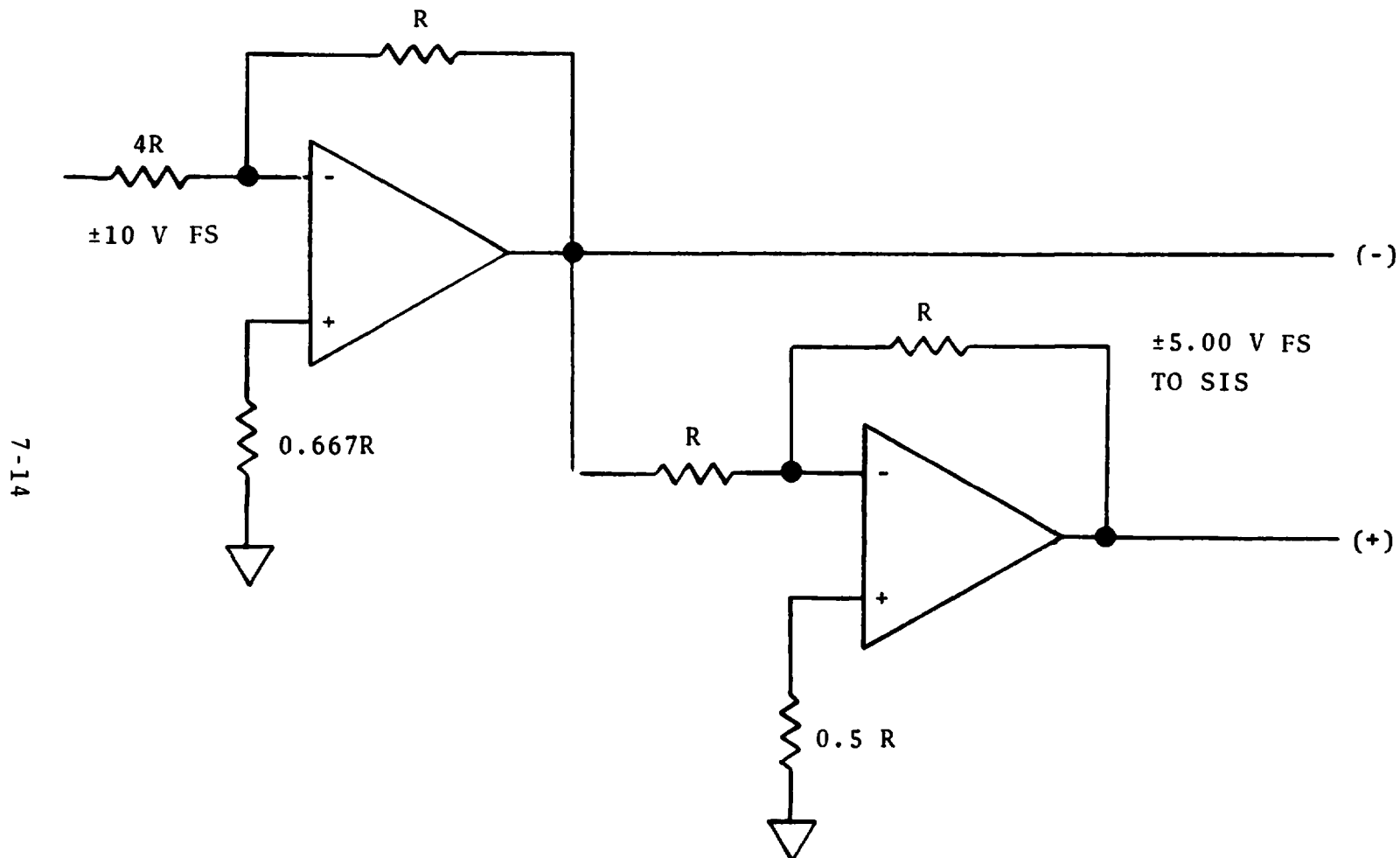


Figure 7-9. - ISO Valve Interface Simulator
Actuator Subsystem.



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Figure 7-10. – Position acceleration SAS/SIS interface.

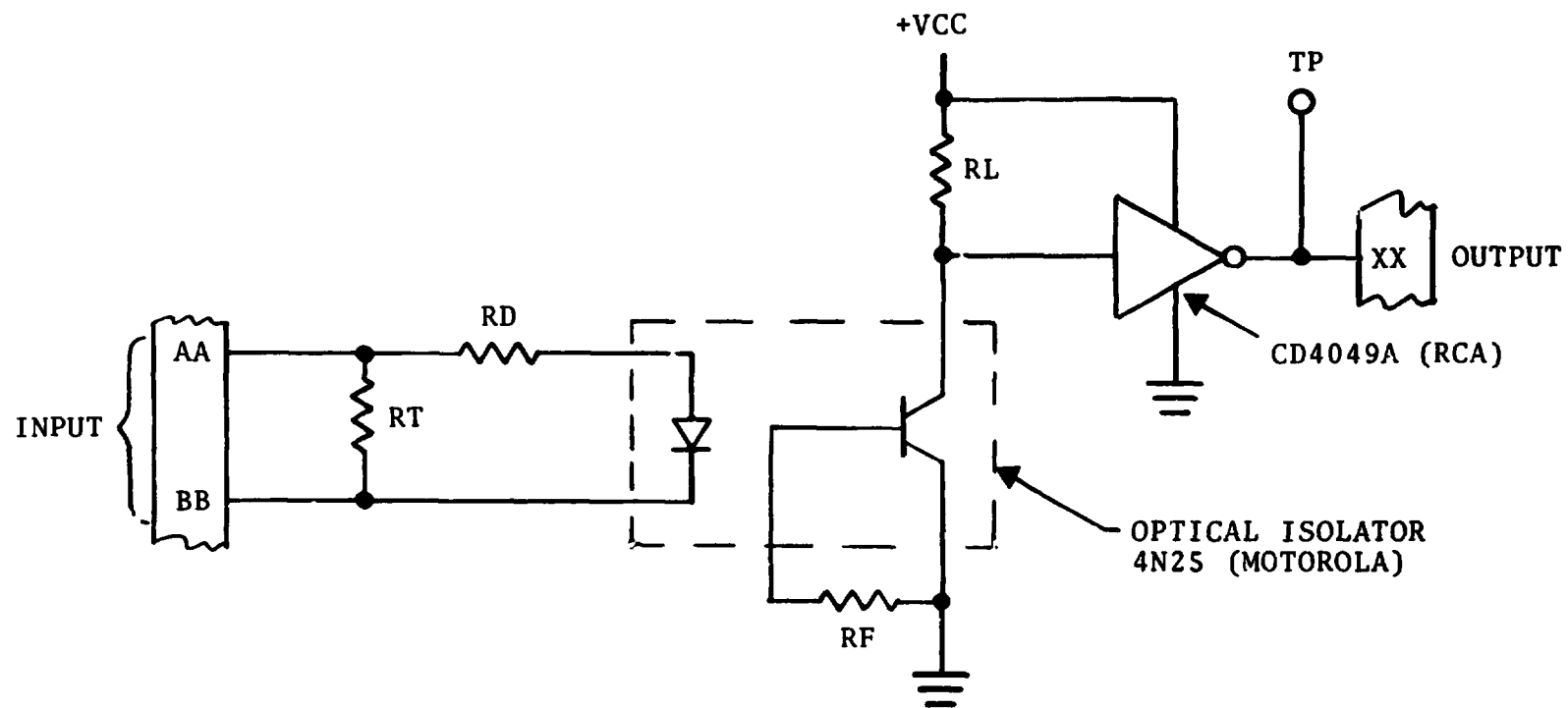


Figure 7-11. - Optical-isolator circuit (typical) mode/fault insertion.

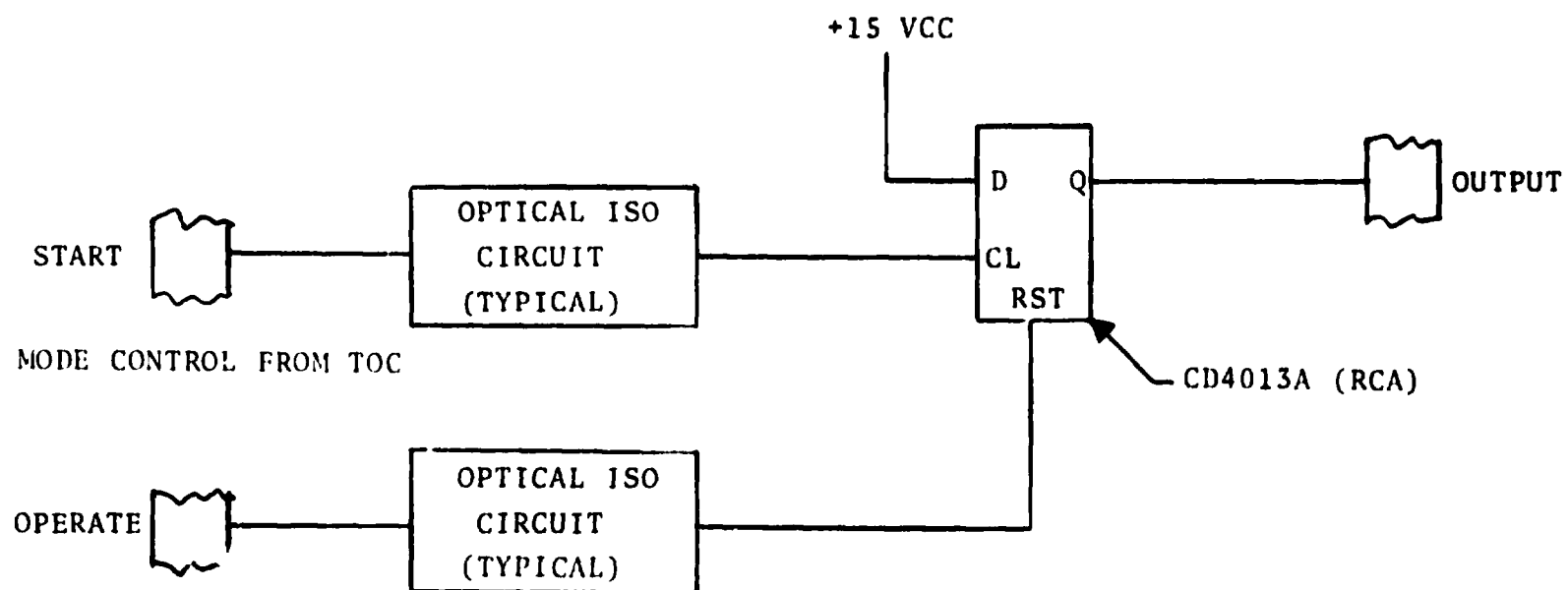
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Figure 7-12. - Mode control circuit (typical).

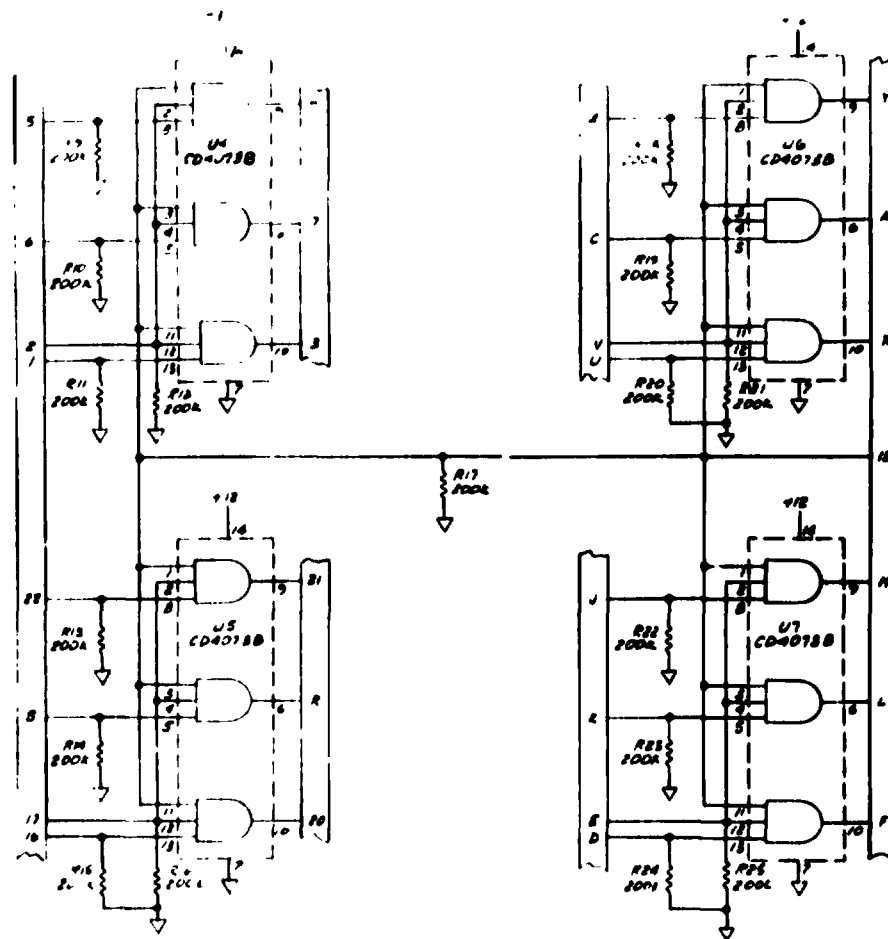
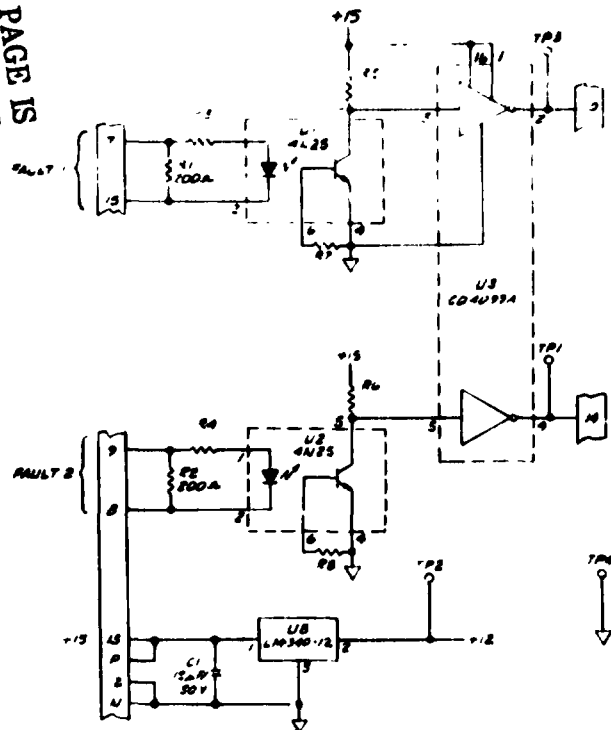


Figure 7-13. - Fault Insertion Simulator
Actuator Subsystem.

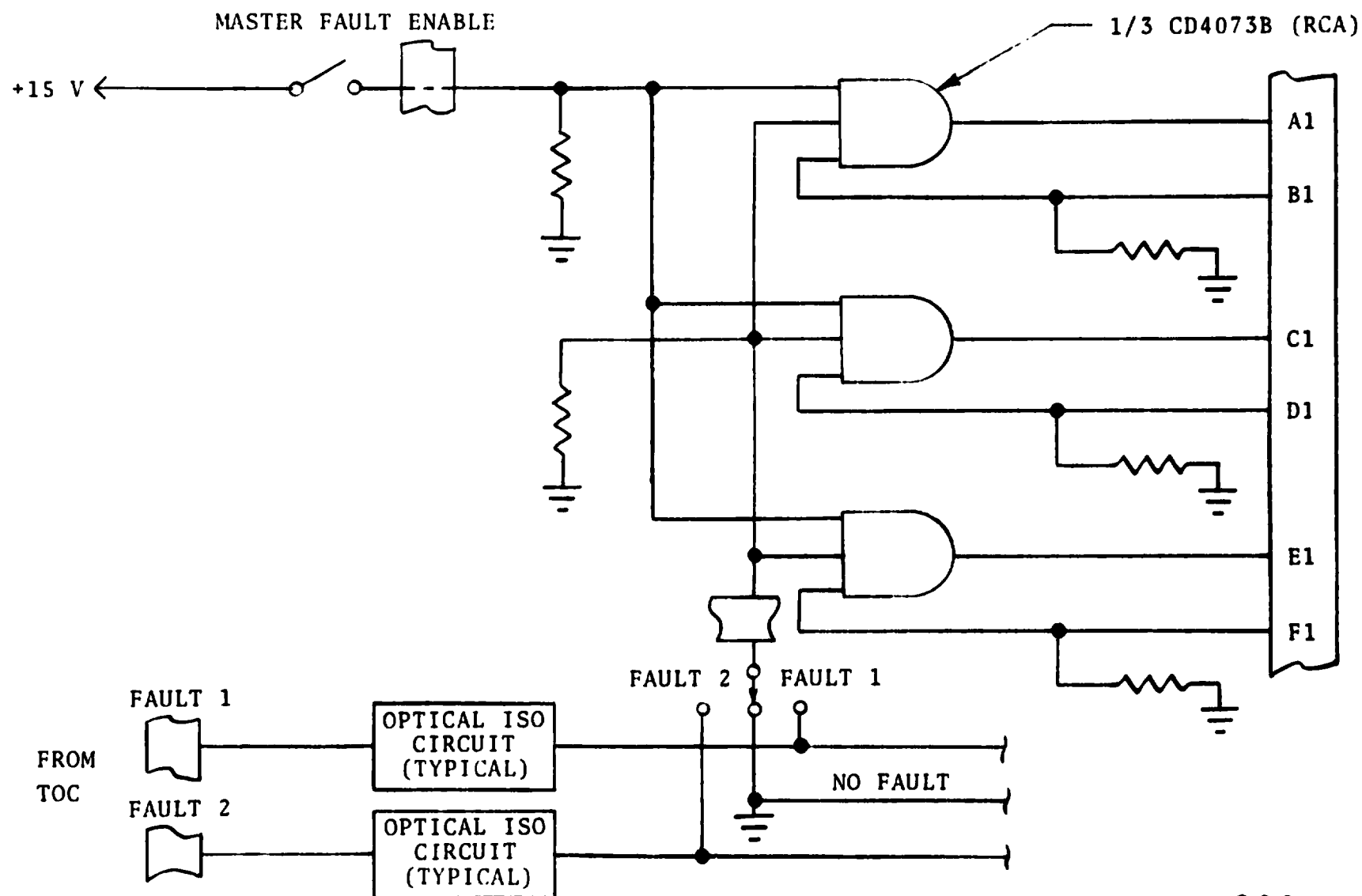


Figure 7-14. - SAS/TOC interface fault insertion switching circuit (typical) part A.



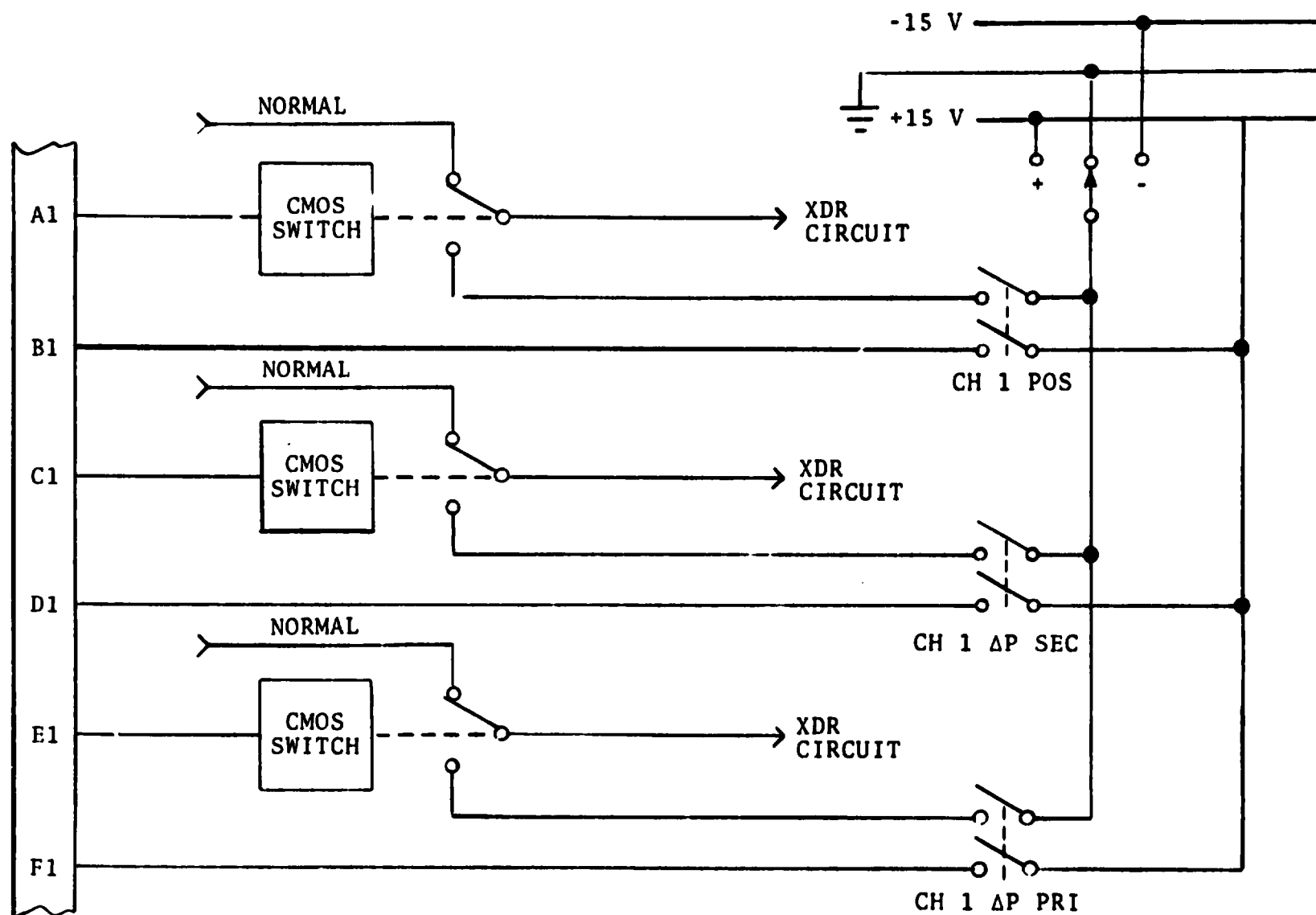


Figure 7-15. - SAS/TOC interface fault insertion switching circuit (typical) part B.



APPENDIX A
MODEL 1 LISTING

TITLE 625/34-5479 ELEVON ACTUATOR MODEL 1
 * FULL-UP MODEL FREQUENCY RESPONSE
 *
 * SYSTEM MACROS
 *

11.25.75

* LIM1 IS A LIMITED FIRST ORDER SYSTEM
 MACRO Y=LIM1(YCOT,P1,P2)
 PROCEDURE DYDT=LIM1(Y,YCOT,P1,P2)
 DYCT=YDCT
 IF(Y.LE.P1) DYCT=AMAX1(0.,YDCT)
 IF(Y.GE.P2) DYCT=AMIN1(0.,YDCT)
 ENDPROCEDURE
 Y = INTEGRAL(0.0,DYCT,
 ENDMAC

* LIM2 IS A LIMITED SECOND ORDER SYSTEM
 MACRO Z,ZDOT=LIM2(ZDC1,P3,P4)
 PROCEDURE ZDDOT1,ZDCT1=LIM2(Z,ZDC1,ZDDC1,P3,P4)
 IF(Z.LE.P3) GO TO 30
 IF(Z.GE.P4) GO TO 31
 ZDCOT1=ZDCCT
 ZDCT1=ZDCT
 GO TO 32
 30 ZDCOT1=AMAX1(0.,ZDDOT1)
 ZDCT1=AMAX1(0.,ZDCT)
 GO TO 32
 31 ZDCOT1=AMIN1(0.,ZDDOT1)
 ZDCT1=AMIN1(0.,ZDCT)
 32 CONTINUE
 ENDPROCEDURE
 ZDCT=INTEGRAL(0.,ZDDCT1)
 Z = INTEGRAL(0.,ZDCT1)
 ENDMAC

*
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 *

ORIGINAL PAGE IS
 OF POOR QUALITY

A-1

```
*****
*      FULL-UP STROKE: ELUVON MODEL
*      FULL-UP STROKE: ELUVON MODEL
*****
```

***** CONSTANTS AND FRACTIONS ARE FOR THE OUTBOARD ACTUATOR SYSTEM ***

```
CONST AC=8.975E-3, AP=2.761E-2, APS=1.930E-1, AS=1.802E1
CONST EE=1.500E4, RE(A)=1.717E5, RP=6.480E-2, EPS=1.346
CONST CI=1.852E2, CL=1.030E-8
CONST COUL=11.60, CP=3.430E-3, CU=4.530, CUM=8.750E-5
CONST CTW=1.096E-1, DELK=1.550E4
CONST CELEIN=1.000
CONST IC=0.0, IE=2.663FJ, IL=8.000
CONST KAMP=15.00
CONST KASA=0.352712, KH=3.190E-1, KC=1.710, KFA=1.173
CONST KL=1.00, KN=1.380E-4
CONST KP=1.200E3, KPG=5.030E-3, KPT=1.670E-3
CONST KQPS=5.180E1, KS=1.740E3, KT=1.246E5, KTM=4.500E-2
CONST KXPS=6.220, KU=4.24, KI=1.548E-1, KZ=3.224E-2
CONST K12=1.494E-1, K22=4.460E-2
CONST L=1.000E-3, LAP=1.100E-3, LD=7.070E-1, LU=1.543E-2
CONST VP=6.830E-5, MPS=2.070E-3, PSS=3.000E3
CONST F3=0.0, F4=(15.0,20.0,25.0), F5=0.1
CONST RAD=57.2958, RCL=4.750E-5
CONST RLE=1.452E3, T4UC=1.000E-1, T4ERC=0.0, T4I=4.000E-3
CONST V=8.380E-2, VT=6.20E-1, WU=3.14F2, XPL=1.000E-3
CONST X0=1.850E-3, XPSL=7.500E-2
```

```
FUNCTION MOMENT = (-36.5,7.767), (-35.0,7.884), ...
(-30.0,8.223), (-25.0,8.481), (-20.0,8.661), ...
(-15.0,8.766), (-10.0,8.830), (-7.705,8.793), ...
(-5.00,8.767), ( 0.00,8.671), ( 5.00,8.517), ...
(10.0,8.307), (15.0,8.047), (20.0,7.740), (21.5,7.639)
```

```
FUNCTION STROKE = (-36.5,-4.266), (-35.0,-4.061), ...
(-30.0,-3.358), (-25.0,-2.628), (-20.0,-1.880), ...
(-15.0,-1.116), (-10.0,-0.352), (-7.705,0.000), ...
(-5.00, 0.418), ( 0.00, 1.176), ( 5.00, 1.927), ...
```

(10.0, 2.661), (15.0, 3.375), (20.0, 4.064), (21.5, 4.266)

DYNAMIC

NOSORT

```
PNA= LIMIT(-300.,300.,PNA)
IF(XSALM.GE.0.015)  OXSALM= AMIN1(0.0,CXSALM)
IF(XSALM.LE.-0.015) OXSALM= AMAX1(0.0,CXSALM)
XSALM= LIMIT(-0.015,0.015,XSALM)
XPS= LIMIT(-XPS,XPSL,XPS)
XR= LIMIT(-5.442,3.090,XR)
DELE= LIMIT(-0.477045,0.375245,DELE)
```

SORT

```
*
* ASA SIGNAL OUTPUT
*
* THE ASA OUTPUT IS RATE LIMITED TO 20 DEGREES/SECOND
  CMDELE= DELEIN*CTNE(P3,P4,P5)
  DELINC= AFGEN(STROKE,CMDELE)
  VCA = (DELINC-1.176)*KFB
  IA = (VCA-VZ)*KMF - VK
  ILIMA=LIMIT(-IL,IL,IA)
*
* ELEVON TORQUE MOTCH
*
  H = 0.020 + (0.14/7.6)*ABS(ILIMA)
  TCHSA=HSTRSS(0.,-H,H,ILIMA)
  TCA=KTM*TCHSA
*
* FLAPPER VALVE DYNAMICS
*
```

```
ETA=TCA-(PNA*KN)-(KXPS*XPSLIM)
XFA=ETA*LN
XFLIMA=LIMIT(-XFL,XFL,XFA)
QNA=XFLIMA*2.*C1
QXA=2.*(AF-AC)*CXALM
QZA=UNA-QXA
```

```

DPNA=((-PNA*(CTL+CTW*XO))+GZ)/((V/(2.0*FETA))
PNA=LIM1(DPNA,-3000.,3000.)
FCHA=PNA*(AP-AC)

```

SECONDARY VALVE DYNAMICS

```

FCHA=PNA*AC
XSAISW=DEADSP(-0.20+0.20*(FCHA-FCHTA)
XSAFSW=FCNSW(DXSALM,-0.2,XSAISW,0.2)
FORCEA=FCHA-FCHTA-XSAFSW
DDXSA=(FORCEA-PP*DXSALM-KP*ASALIM)/MP
XSALIM,DXSALM=LIM2(DDXSA,-0.015,0.015)

```

SECONDARY FLOW EQUATIONS

```

PSP=PSS-(K12+APS*(U1*K22))*AHS(GL)
PVS=PSP-PCA*SIGN(1.,XSALIM)
QQA=CQ*XSALIM*CIGN(SQRT(APS(PVS)),PVS)
DXSALM=DXSALM*AP
QSA=QQA+QXSAIM
DXQA=QSA/APS
XQA=INTGR1(IC,DXQA)
XCQA=XQA-XPSLIM
FA=XQSA*4.*PETA*APS*AHS/VT
PCA=FA/APS

```

NOE PISTON DYNAMICS

```

XFSDSP=DEADSP(-L+L*XPSLIM)
AGL=ABS(GL)
PSL=PSS-(K0+(K1+K2*AGL))*AGL
FPH=(PSL-PPH)*XFSDSP*KH
COULF1=DEADSP(-COUL+COUL*(4.0*FA)-FPH)
COULF=FCNSW(DXPSL,-COUL+COULF1,COUL)

```

```

FTCT=(4.*FA)-FR-COLLF
DDXPS=(FTCT-(APC*(XPS)))/MPS
DXPSL=DERIV(1C,XPSLIM)
XPS,DXPS=LIM2(CDXPS,-XPSL,XPSL)
XPSLIM=LIMIT(-XCSL,XPSL,XPS)

```

LOAD FLOW EQUATIONS

```

XPSEFF= SQRT((1-ABS(XPSLIM))**2+RCL**2)
XPSFL= AMAX1(0.1E-3,L*XPSLIM)
XPSML= AMAX1(0.1E-3,L-XPSLIM)
PPRE=PI*SIGN(1.,XPSLIM)
QL= IMPL(0.0+0.01,FCQL)
PLKG= K0*(K1+K2*ABS(QL))*APS(QL)
PS=FSS-PLKG
PV=FS-PPRE
APV= ABS(PV)
APSMPL= APS(PS-PL)
APSPPL= APS(PS+PL)
TERM1= CL*APSMPL/XPSML
TERM2= KGPS*HCL*SQRT(4PSMPL)
TERM3= CL*APSPPL/XPSPL
TERM4= KGPS*HCL*SQRT(4PSPL)
QLL= SIGN(AMIN1(TERM1,TERM2),PS-PL)-SIGN(AMIN1(TERM3,TERM4),PS+PL)
QLH= KGPS*XPSEFF*SIGN(SQRT(APV),XPSLIM*F/) ...
-SIGN(CL,XPSLIM)*(PS+PI*SIGN(1.0,XPSLIM))/(L+ABS(XPSLIM))
DFCGL= ABS(XPSLIM)-LAP
FCQL = FCNSW(DFCGL, QLL, CLH, CLF)

```

RAM FISTON DYNAMICS

```

DXR= QL/AP
XR = LIM1(CXR,-9.442+3.090)
XTOT= XR-XSTR
FLOCT= KT*XTOT
PL=FLOOT/AP
MR=4FGEN(MOMENT,DELED)
TR=MR*FLOCT

```

DFLEK=DFLE*KL

* THE FOLLOWING PROCEDURE COMPUTES THE TOTAL TORQUE INCLUDING
* THE EFFECTS OF ACTUATOR STICKION AND ELEVEN STICKION.

RLE1= RLE*MR

PROCEDURE TTOT= STICTN(DDELED,DXFRK,DELK,RLE1,TR,TAERC,DFLEK)

TLIMIT= DELK*RLE1

T1= TR+TAERC-DEIFK

T2= FCNSW(DXFRK, RLE1,0.0,RLE1)

T3= FCNSW(DDELED,-DELK,0.0,DELK)

IF(DXFRK.NE. 0.0) GO TO 200

IF(DDELED.NE. 0.0) GO TO 120

TC HERE IF DXFRK.EQ.0 AND DDELED.EQ.0

TTOT= CEACSP(-TLIMIT,TLIMIT,T1)

GO TO 400

TC HERE IF DXFRK.EQ.0 AND DDELED.NE.0

120 TTOT= CEACSP(-RLE1,RLE1,T1-T3)

GO TO 400

200 CONTINUE

IF(DDELED.NE. 0.0) GO TO 300

TC HERE IF DXFRK.NE.0 AND DDELED.EQ.0

TTOT= CEACSP(-DELK,DELK,T1-T2)

GO TO 400

TC HERE IF DXFRK.NE.0 AND DDELED.NE.0

300 TTOT= T1-T2-T3

400 CONTINUE

ENDPROCEDURE

DDDELE=(TTOT-(HF*DDELED))/TF

DFLE,DDDELE= LIM2(DDDELE,-0.637045, 0.375245)

DELED=DFLE*RAD

DDDELED = DDDELE*RAD

STR = AFGEN(STRCKF,DELED)

XSTR = STR - 1.1760

XKS=FLOOT/KS

XFRK= XSTR + XKS

DXFEK=DERIV(0.0,XFRK)

*

* DELTA-PL TRANSDUCER AND ASA DEMODULATOR


```

*
*   PLHYS1= HSTRSS(n,0,-62.5,62.5,PL1)
*   VLDC1= PLHYS1*KBT
*   VL = REALPL(IC,TCT,VLDC1)
*   VWDCOT= WD*WD*VI
*   VW= CMPLXPL(IC,IC,LD,WD,VWDCOT)
*   VK1= LENLAG(TAUC*KC,TAUC,VW)
*   VK2=REALPL(IC,TAUC,VW)
*   VK = VK1 - VK2

```

```

*
*   POSITION TRANSDUCER AND ASA DEMODULATOR
*

```

```

*   LVDTD= KFB*XFHK
*   LVDT= REALPL(IC,TDT,LVDTD)
*   V7DCUT= WC*WD*LVDT
*   VZ = CMPLXPL(IC,IC,LD,WD,V7DCUT)

```

```

*
*   DELTA-PS TRANSDUCER AND ASA DEMODULATOR
*

```

```

*   VPSCOT= PCA*KPT*WL*WD
*   VPS = CMPLXPL(IC,IC,LD,WD,VPSCOT)

```

TERMINAL

METHOD RKSFx

```

TIMER      CELT=0.00005,CLTDEL=0.005,PRDEL=0.010,FINTIM=2.000

```

```

RANGE      VCA,IA,ILIMA,TCHSA,TCA,ETA,QNA,GXA,GZA,PNA,...
           FOPA,FORTA,XCAFSW,FORCEA,XSALIM,CXSALM,...
           FSP,PVS,GQA,CXSALM,QSA,XGA,FA,PCA,FHF,COULF,FTOT,XPS,IAPS,...
           XPSLIM,PS,PV,CL,XR,XTOT,FL,TR,ITCT,...
           CDELED,DELED,XSTR,XKS,XFRK,LVDT,VZ,PLHYS1,VL,VW,VK

```

```

LABEL 625/34-5479 ELFCVN ACTUATOR MCEL 1 11.25.75

```

```

LABEL      SMALL SIGNAL FREQUENCY RESPONSE

```

```

OUTPUT     DELED,CMDELE

```

```

OUTPUT     ILIMA,CXFRK

```

```

LABEL      ELEVON POSITION DEGREES

```

```

OUTPUT     CDELED

```

```

LABEL      ELEVON RATE DEGREES/SECOND

```

```

OUTPUT     ILIMA

```

```

LABEL      TORQUE MOTOR CURRENT MILLIAMPS

```

```

OUTPUT  XPSLIM
LABEL   POWER  SPOOL  POSITION  INCHES
OUTPUT  CL
LABEL   LOAD  FLOW    CURIC INCHES / SECOND
OUTPUT  FL
LABEL   LOAD  PRESSURE  LBS/SQUARE INCH
OUTPUT  FSL
LABEL   DROPPED SUPPLY PRESSURE  LBS/SQUARE INCH
OUTPUT  FOA
LABEL   SECONDARY DELTA PRESSURE FEEDBACK  LBS/SQUARE INCH
OUTPUT  STR,XFRK
LABEL   ACTUATOR POSITION  INCHES
LABEL   ACTUATOR POSITION FEEDBACK  INCHES
OUTPUT  XSALIM
LABEL   SECONDARY VALVE POSITION  INCHES
PRINT  VCA,XSALIM,XCA,DXPS,XPS,XX,ODELFC,DELECO,V7
END
STOP

```

APPENDIX B
MODEL 2 LISTING

```
TITLE 625/34-5479 ELEVON ACTUATOR MODEL 2 11.25.75
* IMPLEMENTATION MODEL FREQUENCY RESPONSE
* MACRO FOR FIRST ORDER LIMITED INTEGRATOR
* NOTE - THIS MACRO ONLY LIMITS THE INPUT OF THE INTEGRATOR.
* THE OUTPUT MUST BE LIMITED IN A NOSORT SECTION AT
* THE BEGINNING OF THE DYNAMIC SECTION.
*
MACRO Y = LINTG(IC,YDOT,LCLIM,HILIM)
PROCEDURE DYDT = LIM1(Y,YDOT,LCLIM,HILIM)
  DYDT = YDOT
  IF(Y .LE. LCLIM) DYDT = AMAX1(0.0,YDOT)
  IF(Y .GE. HILIM) DYDT = AMIN1(0.0,YDOT)
ENDPROCEDURE
Y = INTEGR1(IC,DYDT)
ENDMACRO
*
*
* MACRO FOR SECOND ORDER LIMITED INTEGRATOR
* NOTE - THIS MACRO ONLY LIMITS THE INPUT TO THE INTEGRATORS.
* THE OUTPUTS MUST BE LIMITED IN A NOSORT SECTION AT
* THE BEGINNING OF THE DYNAMIC SECTION.
*
CRD Y,YDOT = LINTG2(IC1,IC2,DYDOT,LCLIM,HILIM)
PROCEDURE DYDT,D2YDT2 = LIM2(Y,YDOT,YDDOT,LCLIM,HILIM)
  DYDT = YDOT
  D2YDT2 = YDDOT
  IF(Y .LE. LCLIM) GO TO 10
  IF(Y .GT. HILIM) GO TO 11
  DYDT = AMIN1(0.0,YDOT)
  D2YDT2 = AMIN1(0.0,YDDOT)
  GO TO 11
10 DYDT = AMAX1(0.0,YDOT)
  D2YDT2 = AMAX1(0.0,YDDOT)
11 CONTINUE
ENDPROCEDURE
YDOT = INTEGR1(IC1,D2YDT2)
Y = INTEGR1(IC1,DYDT)
ENDMACRO
```

```

*
*
INITIAL
*
*      ELEVON ACTUATOR - IMPLEMENTATION MODEL
*      ELEVON ACTUATOR - IMPLEMENTATION MODEL
*      ELEVON ACTUATOR - IMPLEMENTATION MODEL
*      ELEVON ACTUATOR - IMPLEMENTATION MODEL
*
*****
***** CONSTANTS AND FUNCTIONS ARE FOR THE OLIMCO AND ACTUATION SYSTEM *****
*****
CONST  AP=2.7A1F-2,      APS=1.93UE-1,      AN=1H.02
CONST  FL=1.500E4,      HETA=1.717E5,      NPS=1.3H6
CONST  CL=1.0H0E-9,      CUUJ=11.6
CONST  CU=4.550
CONST  CELETA=1.000
CONST  FRAM=1.55UE4,      FIAPP=1.452E3
CONST  MFL=1.258E2
CONST  IE=2.6A1E3,      IL=8.500E0
CONST  KA=1.500E1
CONST  KASA=0.352712,    KH=3.190E-3,      KC=1.710E-1
CONST  KDELE=1.0
CONST  KFR=1.173E0,      KPS=7.47917E-6,      KPT=1.670E-3
CONST  KJPS=51.811,      KQS=4.4769,      KS=1.540E-3
CONST  KT=1.2H5E5,      KIM=4.500E-2,      KXPS=6.22UE0
CONST  KO=4.240E0,      KJ=1.7476E-1,      KP=5.224E-2
CONST  KIP=1.494E-1,      K22=4.460E-2
CONST  L=1.0E-3,      LAP=1.100E-3
CONST  LN=1.653E-2,      LAPS=6.500E-2
CONST  MPS=3.070E-3,      PPS=3.000E3
CONST  P3=0.0,      P4=(15.0,21.0,25.0),      P5=0.0
CONST  RAD=57.2958
CONST  TAERN=0.00,      TAUC=1.00UF-1,      TIT=4.000E-3
CONST  VT2=6.200E-1,      WU=1.140E2
CONST  ZDS=1.000E-3,      ZLS=1.000E-3,      ZETAC=7.17UF-1
FUNCTION MCMENT = (-35.5,7.767), (-35.0,7.884), ...

```

```
(-30.0,4.223), (-25.0,4.411), (-20.0,4.661), ...
(-15.0,4.766), (-10.0,4.800), (-5.0,4.741), ...
(-5.0,4.767), ( 0.0,4.671), ( 5.0,4.517), ...
( 10.0,4.307), ( 15.0,4.007), ( 20.0,3.740), ( 21.5,3.639)
```

```
FUNCTION STROKE = (-36.5,-4.266),(-35.0,-4.061),...
(-30.0,-3.358),(-25.0,-2.628),(-20.0,-1.840),...
(-15.0,-1.116),(-10.0,-0.352),(-5.0,0.000),...
(-5.0, 0.416),( 0.0, 1.176),( 5.0, 1.927),...
( 10.0, 2.661),( 15.0, 3.375),( 20.0, 4.064),( 21.5, 4.266)
```

DYNAMIC
NOSORT

```
XQ= LIMIT(-LXPS,LXPS,XQ)
XPS= LIMIT(-LXPS,LXPS,XPS)
XR= LIMIT(-5.445, 3.090, XR)
DELE= LIMIT(-0.437045,0.375245,DELE)
```

SORT

- * ASA INPUT AND TORQUE MOTOR
- * THE ASA OUTPUT IS RATE LIMITED TO 20 DEGREES/SECOND

```
CMDELE= CELEIN*INF(P3,P4,P5)
DELINC= AFGEN(STROKE,CMDELE)
VC1 = (DELINC-1.176)*KFB
```

```
IA= (VC1-VZ)*KA - VK
I = LIMIT(-IL,IL,IA)
P = 0.02 + (0.16/7.6)*AFS(I)
ILIM= HSTRSS(0.0,-P,P,I)
TC = KTM*ILIM
```

- * FLAPPER AND SECOND STAGE VALVE

```
ETA = TC - KPS*P1 - KXP5*XPS
AQL = AHS(QL)
PSP = PSS = (K11+K12*AQL+K22*AQL*AQL)
PVS = PSP - P1*SIGN(1.0, XS)
XFA= LIMIT(-0.0016,0.0016,ETA*LN)
XSE= XFA*KQS
```

```

ASP= XPS*F1
XS= LIMIT(-0.01,0.015,XSF-XSF)
QSA= XS*CC*SIGN(SQRT(ABS(PVS)),PVS)
XCD= QSA/AR
XQ= LINTG(0.0,VGL,-LXPS,LXPS)
F1= 4.0*HFTA*APC*APS*(XQ-XPS)/VT2
V1= F1/APS

*
* MCD PISTON DYNAMICS
*
PS= PSS= (K0+K1*AGL+K2*AGL*AGL)
PV= PS- PL*SIGN(1.0,XPS)
XPSCSP= DEACSP(-L,L,XPS)
FRH= PV*KP*XPSCSP
* THE FOLLOWING TWO STATEMENTS COMPLETE THE STICKION FORCE
FCOLL1= DEACSP(-COLL,COLL,(4.0*F1)-FRH)
FCOLL= FCNSW(XPSC,-COLL,FCOLL1,COLL)
FTOT= (4.0*F1)-FRH-FCOLL
XPSCD= (FTOT-APC*XPSC)/MPS
XPS,XPSC= LINTG2(0.0,0.0,APSC,-LXPS,1,XPS)

*
* LOAD FLOW EQUATIONS
*
ZD= DEACSP(-ZPS,ZPS,XPS)
ZL= LIMIT(-ZLS,ZLS,XPS)
* IMPLICIT LOOP FOR Q
Q1= IMPL(0.0,0.01,FCOLL)
AGL1= ABS(Q1)
PS1= PSS=(K0+(K1+K2*AGL1)*AGL1)
PV1= PS1- PL*SIGN(1.0,XPS)
JPAF1= KQPS*ZD*SIGN(SQRT(ABS(PV1)),PV1)
QLAM1= 2.0*CL*(PS1*ZL-LAP*FL)/(LAP**2-ZL**2)
FOUL= CLAM1*CPAR1
JPAF= KQPS*ZD*SIGN(SQRT(ABS(PV1)),PV1)
QLAM= 2.0*CL*(PS1*ZL-LAP*FL)/(LAP**2-ZL**2)

*
* HAM PISTON
*
XRD= QL/AR

```

```

XR= LINTG1(0.0,VR,-5.442,3.090)
FL = KT*(XR-XSTR)
PL = FL/AR
MR= AFGEN(MOMENT,DELE)
TR = MR*FL
TDELE = DELE*MR*FL

```

- * THE FOLLOWING PROCEDURE COMPUTES THE TOTAL TORQUE INCLUDING
- * THE EFFECTS OF ACTUATOR STICKION AND ELEVON STICKION.

```

FFXFH= FTFH*MR
PROCEDURE TTOT= STICTN(DELED,XFH,FRAM,FFXFH,TR,TAERO,DDELE)
TLIMIT= FRAM+FFYFH
T1= TR+TAERO-TDELE
T2= FCNSW(XFH,-FFXFH,0.0,FFXFH)
T3= FCNSW(DELED,-FRAM,0.0,FRAM)
IF(XFH.NE.0.0) GO TO 20
  IF(DELED.NE.0.0) GO TO 12
  TC HERE IF XFH.EQ.0 AND DELED.EQ.0
  TTOT= DEADSP(-TLIMIT,TLIMIT,T1)
  GO TO 40
* 12 TC HERE IF XFH.EQ.0 AND DELED.NE.0
  TTOT= DEADSP(-FFXFH,FFXFH,T1-T3)
  GO TO 40
20 CONTINUE
  IF(DELED.NE.0.0) GO TO 30
  TC HERE IF XFH.NE.0 AND DELED.EQ.0
  TTOT= DEADSP(-FRAM,FRAM,T1-T2)
  GO TO 40
* 30 TC HERE IF XFH.NE.0 AND DELED.NE.0
  TTOT= T1-T2-T3
40 CONTINUE
ENDPROCEDURE

```

```

DELED = REALP(0.0,IE/4E,TTOT/PE)
DDELED= DDELE*FRAM
DDELE= LINTG1(0.0,DELED,-0.637045,0.375245)
DDELE = DDELE*RAI
STR = AFGEN(STROKE,DDELE)
XSTR = STR - 1.1760

```



```

XFH = XSTH + F1*KS
XFHD = DFHIV(0.0,XFH)

```

```

*
* TRANSDUCERS AND DEMODULATORS OUTPUTS
*

```

```

VX = REALPL(0.0,TOT,XFH*KFF)
VZ = CMPLPL(0.0,0.0,ETAD,WID,VX*WD*WD)
PLF = HSTRSS(0.0,-HPL,HPL,PL)
VPL = REALPL(0.0,TOT,PLH*KFI)
VM = CMPLPL(0.0,0.0,2*ETAD,WID,VPL*WD*WD)
VK1 = LECLAG(TALC*KC,TAUC,VM)
VK2 = REALPL(0.0,TAUC,VM)
VK = VK1 - VK2

```

TERMINAL

METHOD RKSF4

TIMER DELT=0.00005,CLTDEL=0.005,PRDEL=0.0)10*FINTIM=2.000

RANGE XQ,XPSF,XPS,XR,DELE,VCI,VK1,VK,TAU,ILIM,TC,F1,PI,...
ETA,DEPLE,STC,XSTH,FL,PL,ZL,ZL,GL,AGL,PSR,PVS,XQ,...
XQ,FCCL,PS,CV,FTOT,XPSF,XPS,XHD,AK,KK,TR,XFH,XFHD,XFH,...
IDELF,TFDELE,TTOT,DELED,DELE,VX,VZ,PLH,VPL,VM,VK2,QPAR,QLAY

LABEL 625/34-5479 ELEVON ACTUATOR MODEL 2 11.25.75

LABEL SMALL SIGNAL FREQUENCY RESPONSE

OUTPUT DELE,CMDELE

LABEL ELEVON POSITION DEGREES

OUTPUT DELED

OUTPUT I,XFRD

LABEL ELEVON RATE DEGREES/SECOND

OUTPUT I

LABEL TORQUE MOTOR CURRENT MILLIAMPS

OUTPUT XPS

LABEL POWER SPOOL POSITION INCHES

OUTPUT GL

LABEL LOAD FLOW CUBIC INCHES / SECOND

OUTPUT PL

LABEL LOAD PRESSURE LBS/SQ INCH

OUTPUT PS

ORIGINAL PAGE IS
OF POOR QUALITY

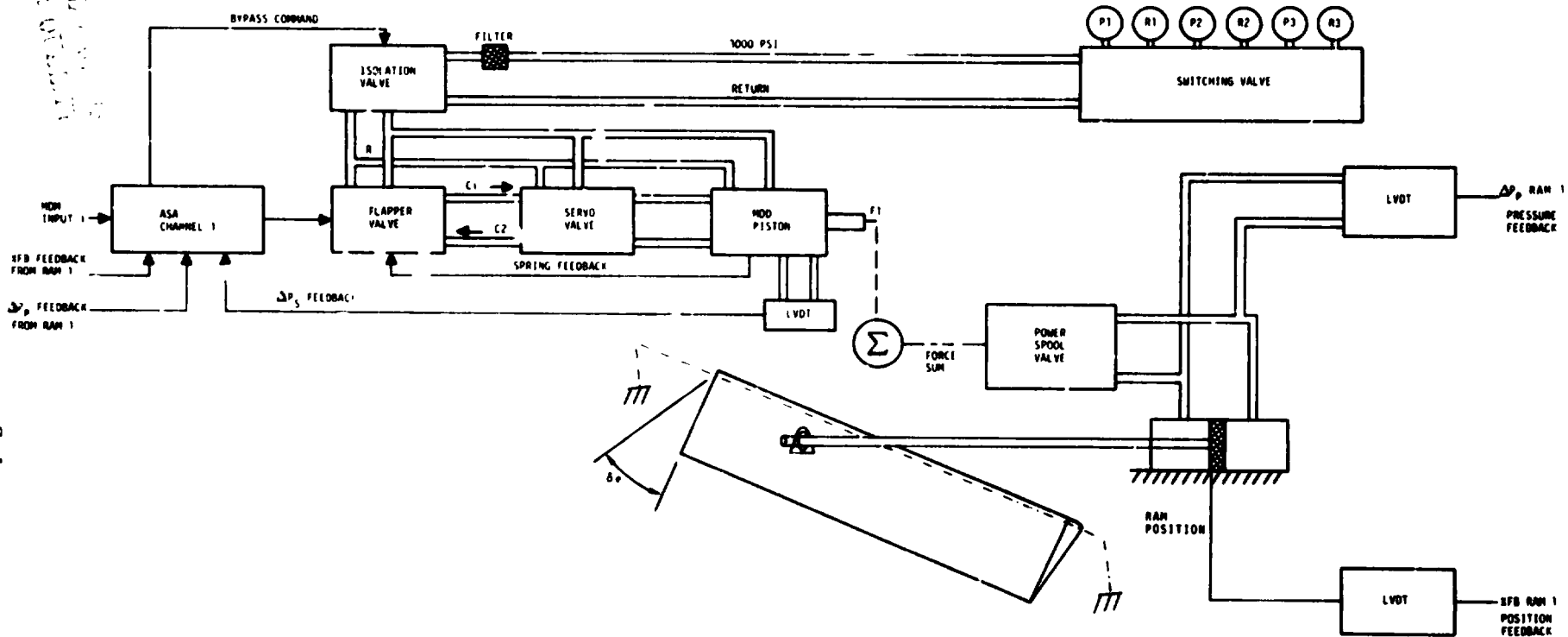
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LABEL  DROPPED SUPPLY PRESSURE    LBS/SQUARE INCH
OUTPUT  FI
LABEL  SECONDARY DELTA PRESSURE FEEDBACK    LBS/SQUARE INCH
OUTPUT  STR,XFR
LABEL  ACTUATOR POSITION    INCHES
LABEL  ACTUATOR POSITION FEEDBACK    INCHES
OUTPUT  XS
LABEL  SECONDARY VALVE POSITION    INCHES
PRINT  VC1,XS,XG,XPCD,XPS,XR,OCFLEO,CCELE,VZ
END
STOP
```

APPENDIX C

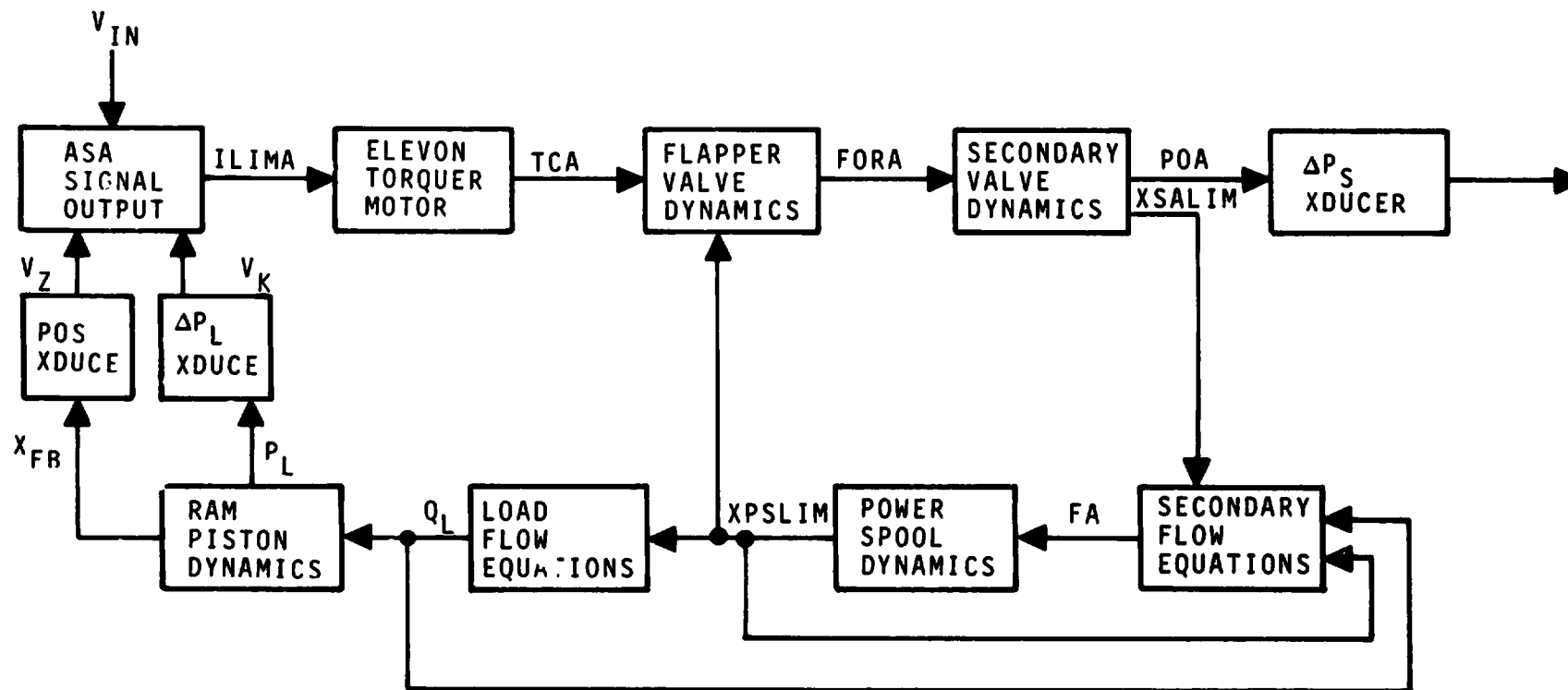
VIEWGRAPHS

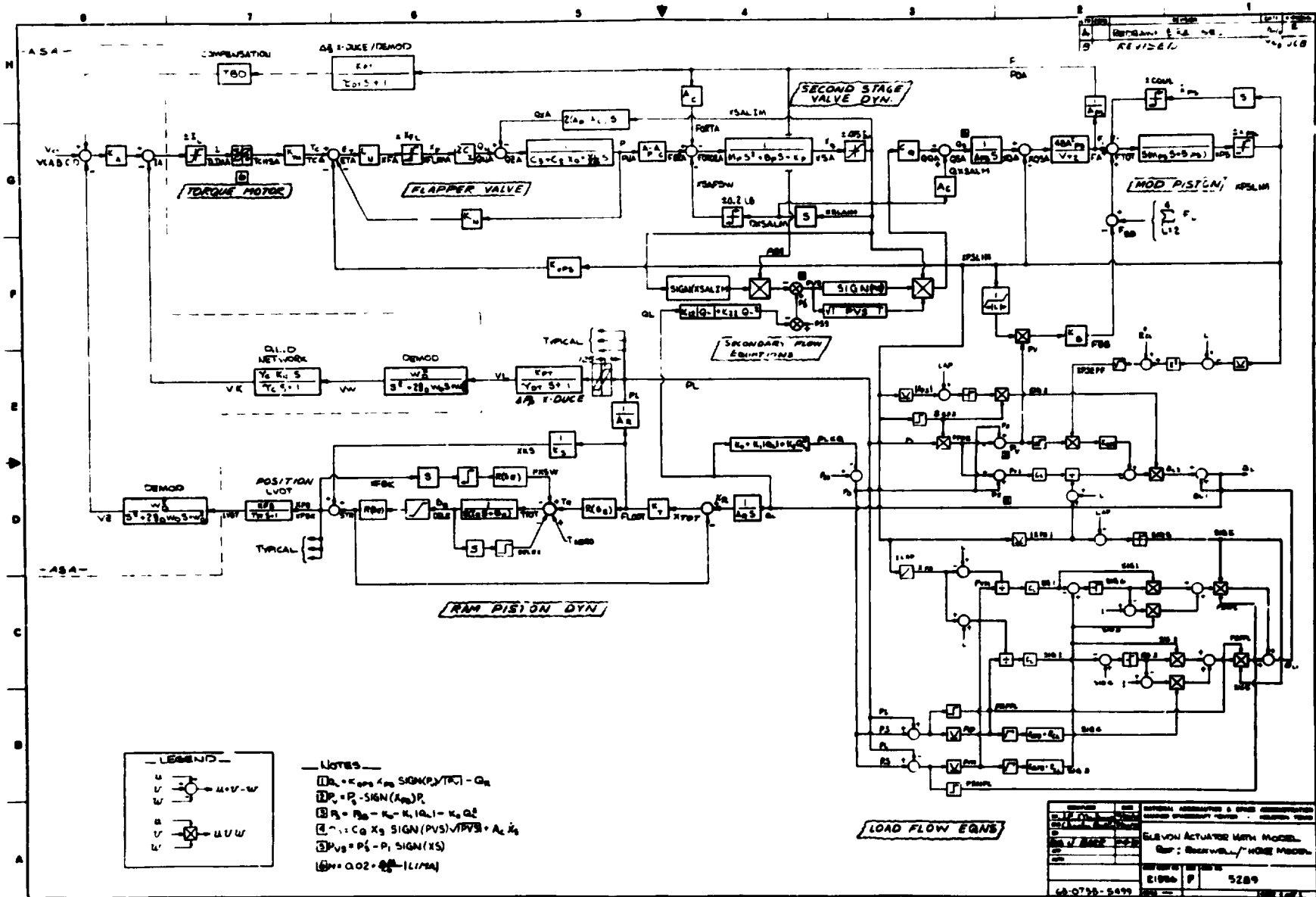
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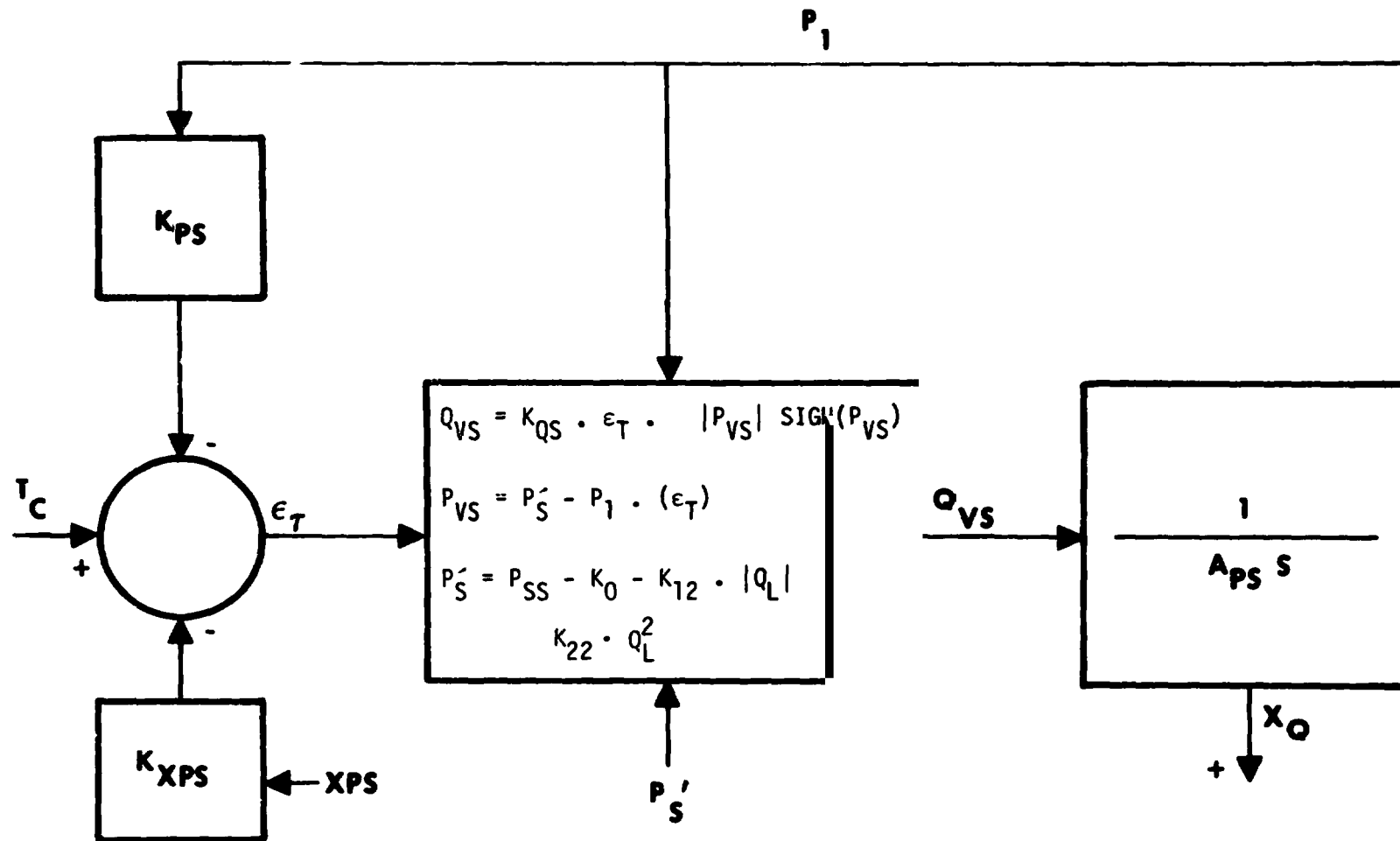
FUNCTIONAL BLOCK DIAGRAM - ELEVON

C-2



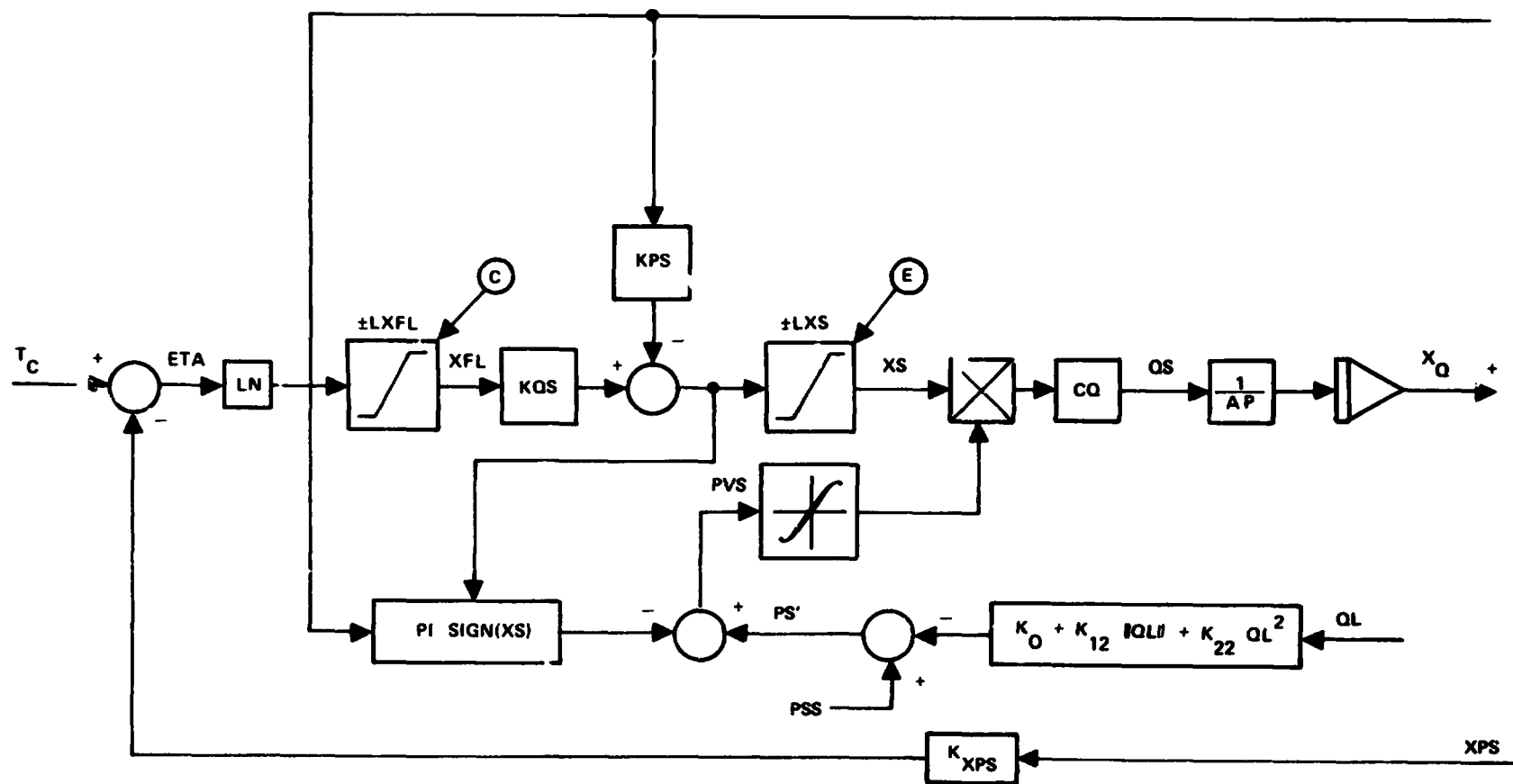


ROCKWELL MODEL 2



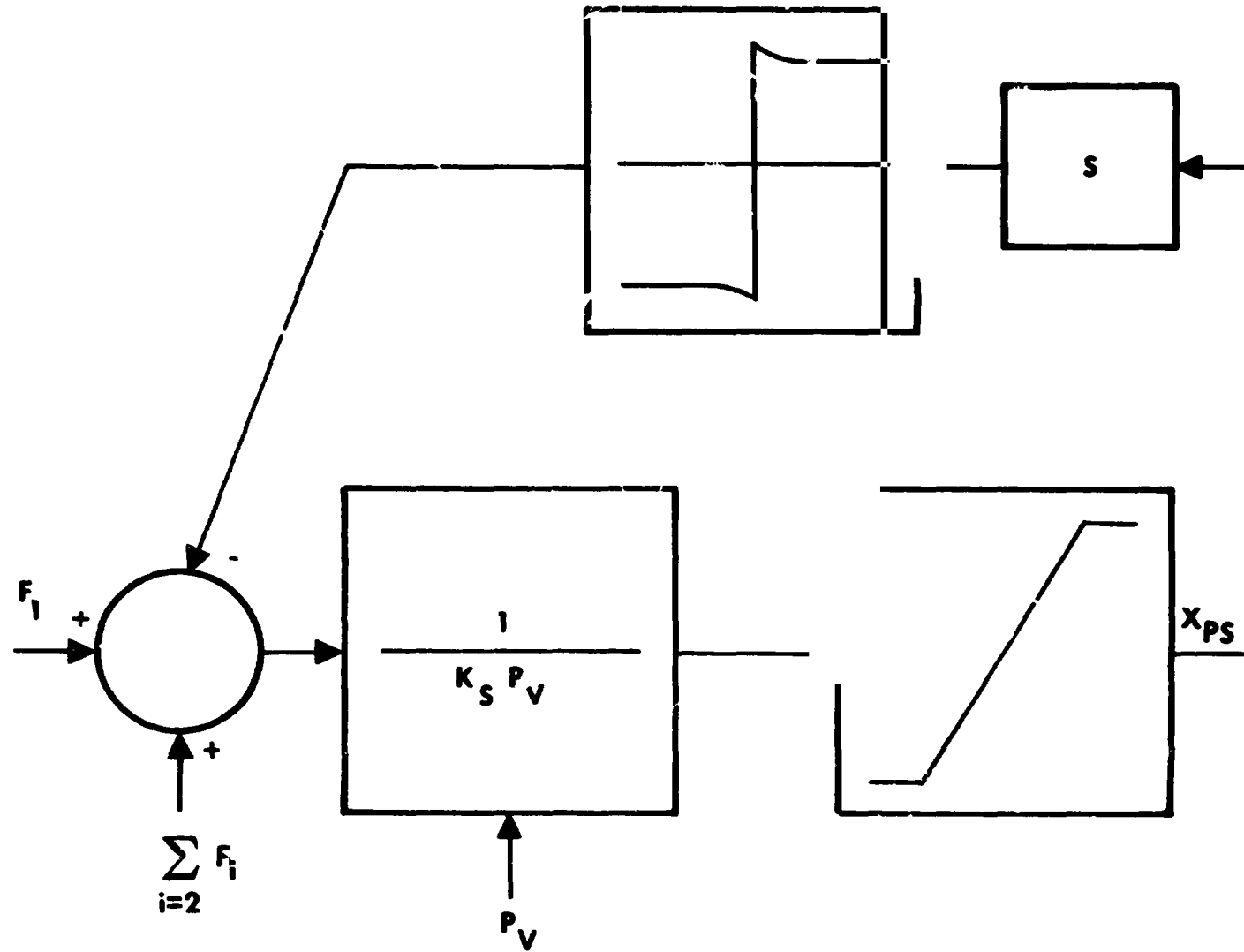
SECOND STAGE VALVE

IMPLEMENTATION MODEL



SECOND STAGE VALVE

ROCKWELL MODEL 12

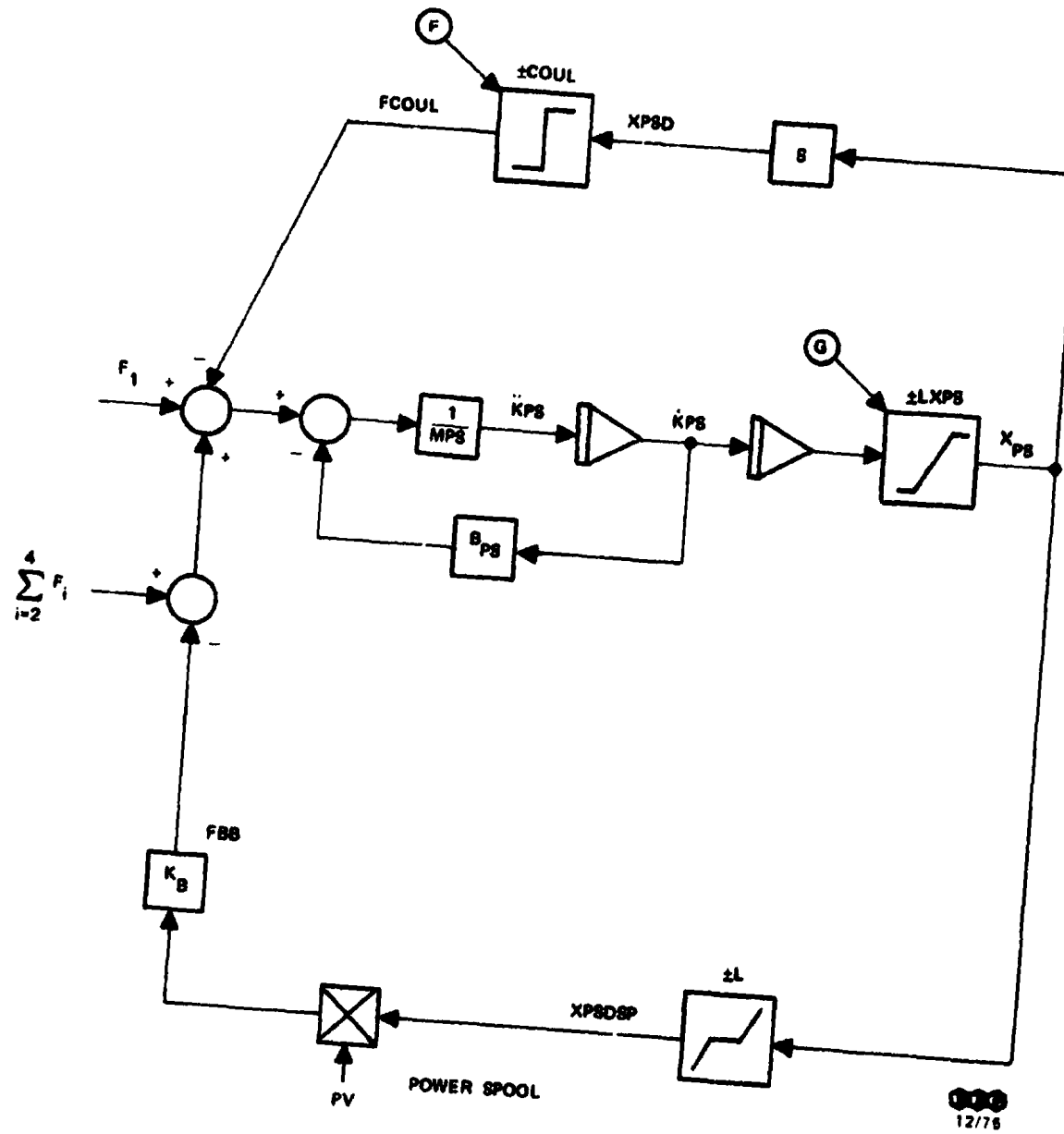


POWER SPOOL

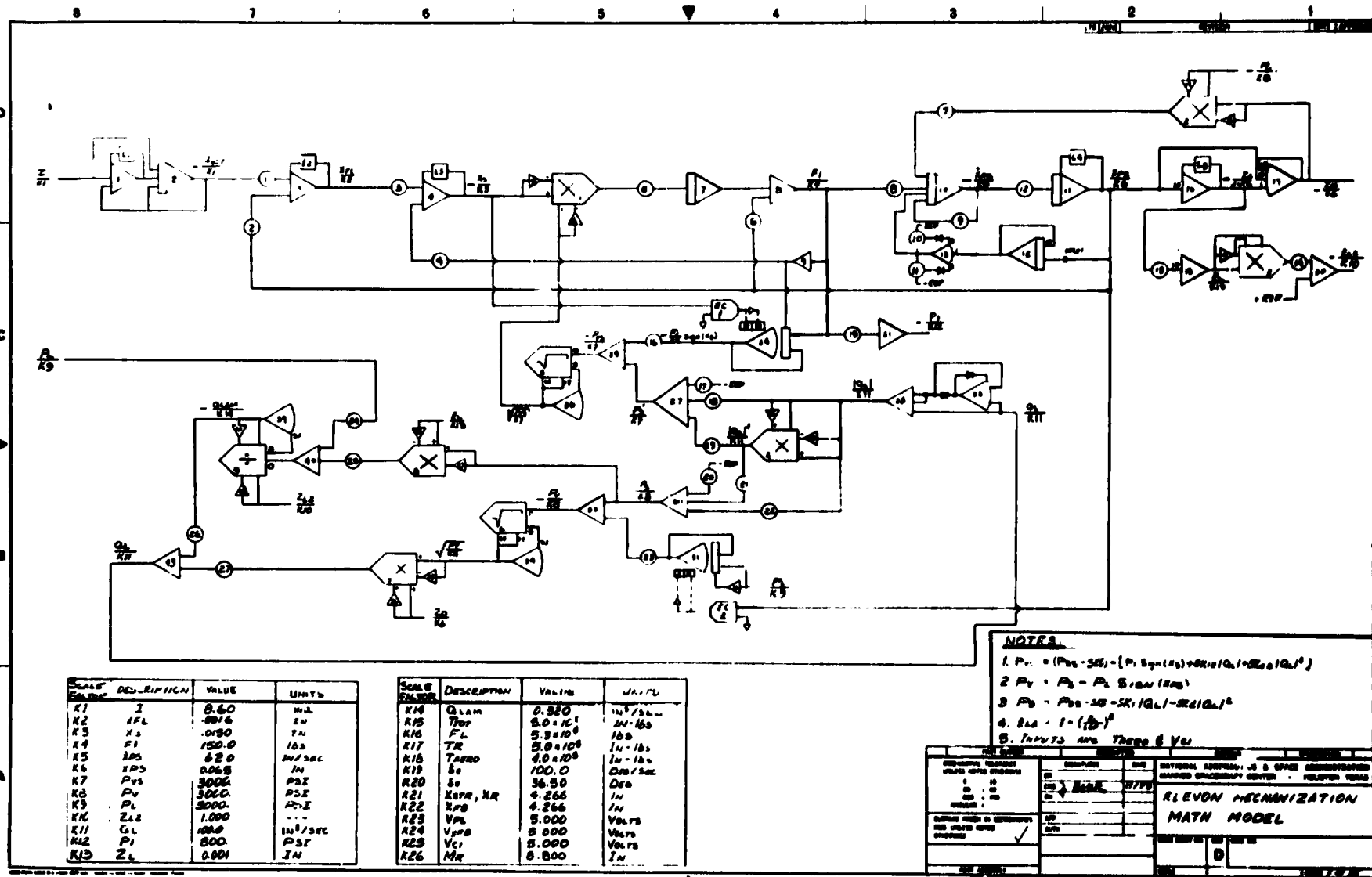


12/75

IMPLEMENTATION MODEL



C-8

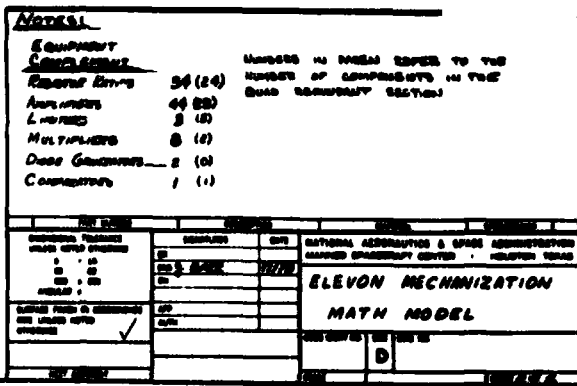


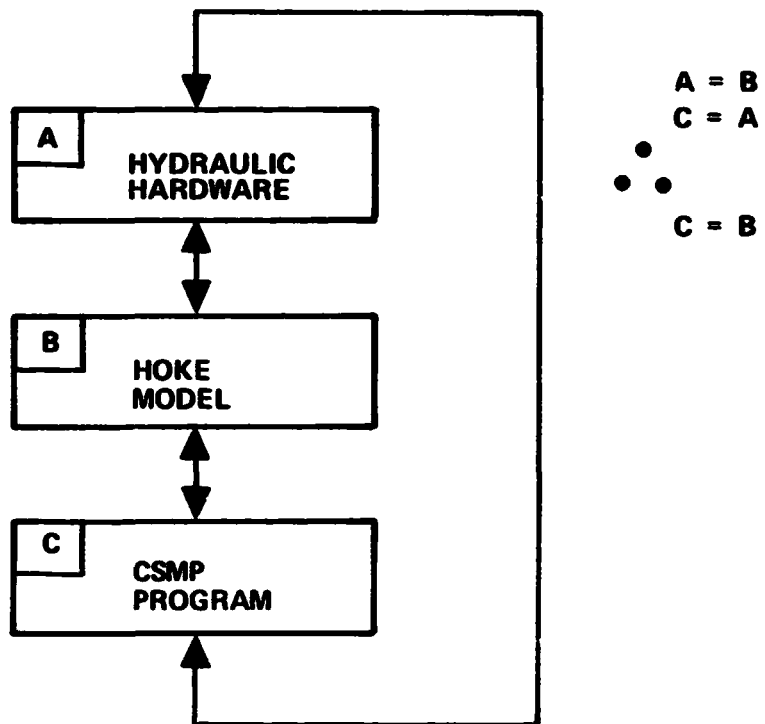
- NOTES**
1. $P_{x1} = (P_{x0} - S_{x1}) - (P_{x1} S_{y1} + S_{y1} S_{x1} + S_{x1} S_{y1} + S_{x1} S_{y1})$
 2. $P_{y1} = P_{y0} - P_{x1} S_{y1} (S_{x1})$
 3. $P_{y1} = P_{y0} - S_{x1} S_{y1} - S_{x1} S_{y1} + S_{x1} S_{y1}$
 4. $S_{x1} = 1 - (S_{x1})^2$
 5. Inputs are those of V₀

COMPONENT	DESCRIPTION	VALUE	UNITS
K1	I	8.60	IN
K2	IFL	8016	IN
K3	FL	1050	IN
K4	F1	150.0	IN
K5	SPS	62.0	IN/SEC
K6	SPS	0.065	IN
K7	PVS	3000	PSI
K8	PV	3000	PSI
K9	PL	3000	PSI
K10	ZL	1.000	---
K11	GL	100.0	IN/SEC
K12	P1	800	PSI
K13	ZL	0.001	IN

COMPONENT	DESCRIPTION	VALUE	UNITS
R14	Q _{LAM}	0.320	IN/SEC
R15	T _{TOP}	5.0 x 10 ¹	IN-16s
R16	FL	5.3 x 10 ⁴	16s
R17	TR	5.8 x 10 ⁵	IN-16s
R18	T _{ABO}	4.0 x 10 ⁵	IN-16s
R19	S ₁	100.0	Sec/Sec
R20	S ₂	36.50	Sec
R21	X _{SPR} , X _R	4.266	IN
R22	X _{SP}	4.266	IN
R23	V _{PL}	5.000	VOLTS
R24	V _{PR}	5.000	VOLTS
R25	V _C	5.000	VOLTS
R26	M _R	5.000	IN

REVISIONS	DATE	BY	APPROVED
1	10/1/68	WJH	
2	10/1/68	WJH	
3	10/1/68	WJH	
4	10/1/68	WJH	
5	10/1/68	WJH	
NATIONAL AERONAUTICS & SPACE ADMINISTRATION SPACE FLIGHT CENTER RESEARCH TOWER RLVON MECHANIZATION MATH MODEL			

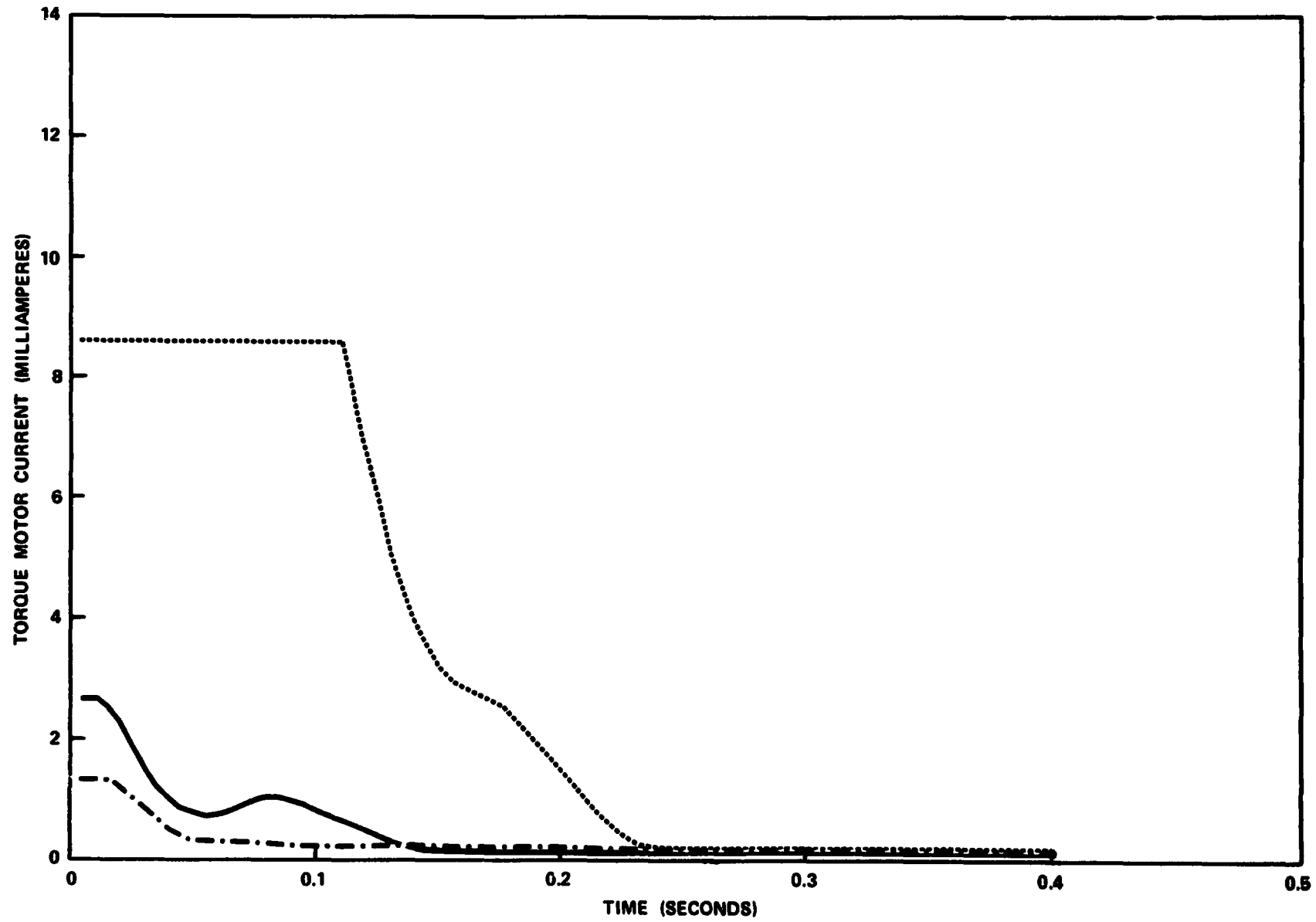




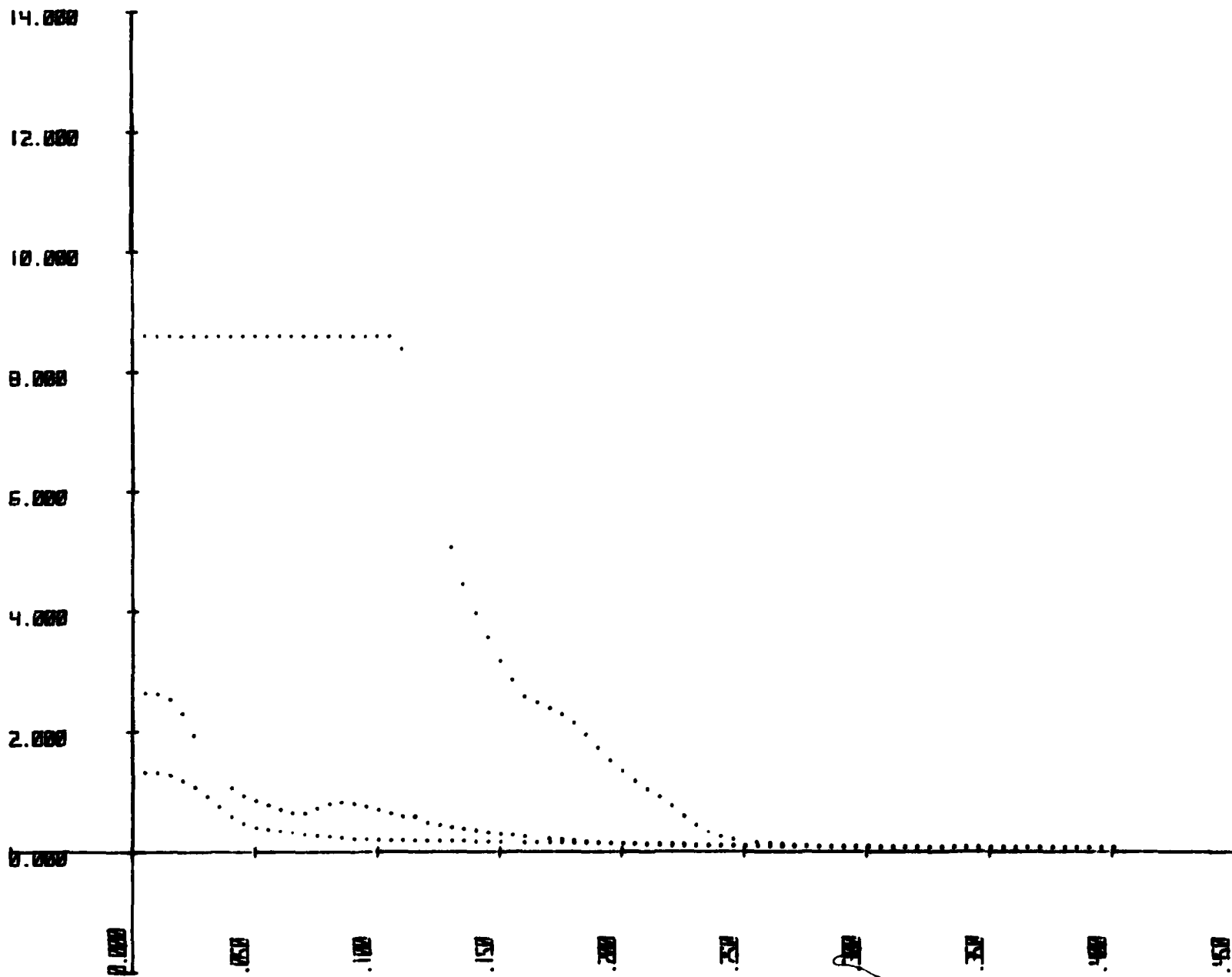
VERIFICATION METHOD

LEC
12/75

LEC
12/76

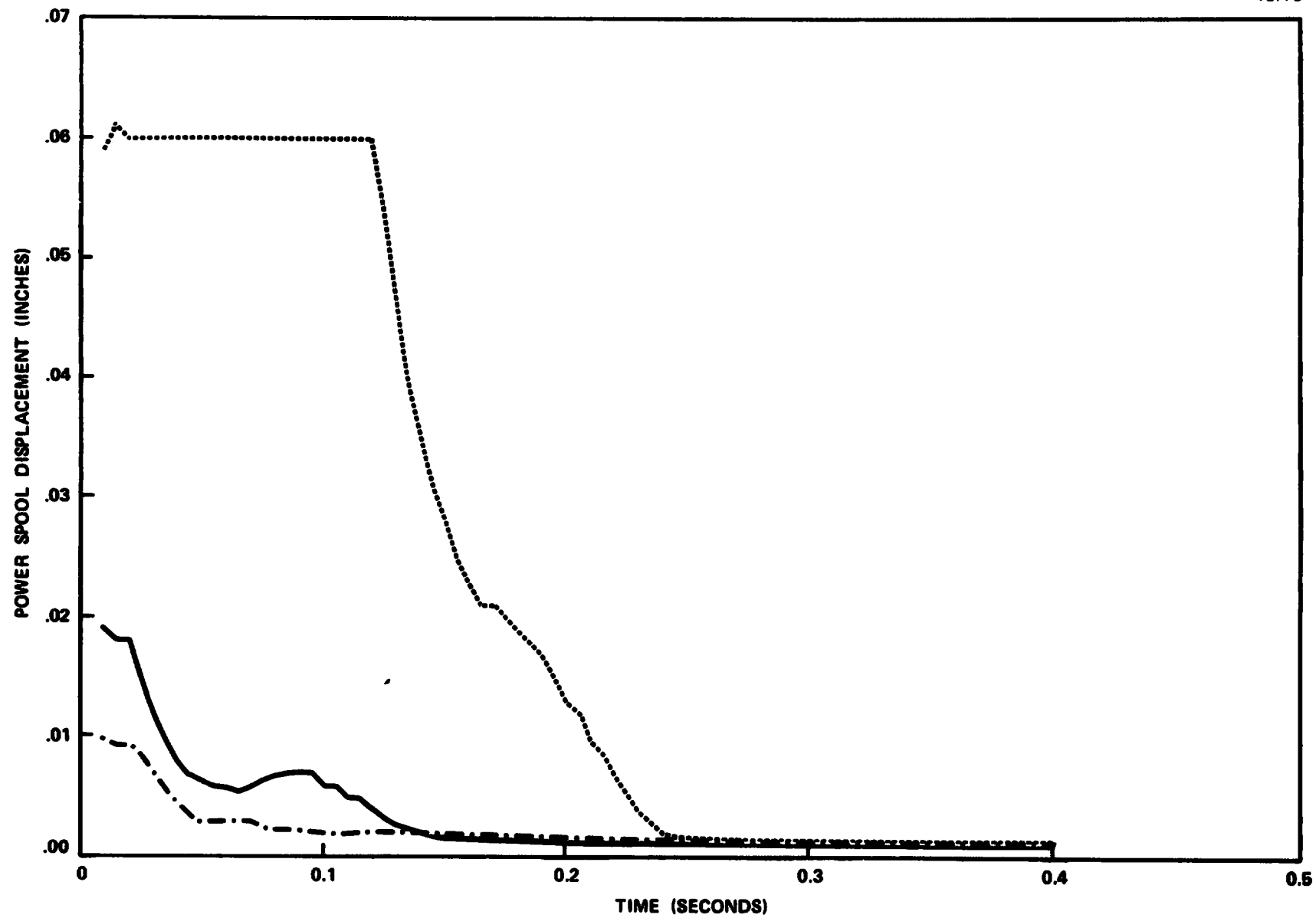


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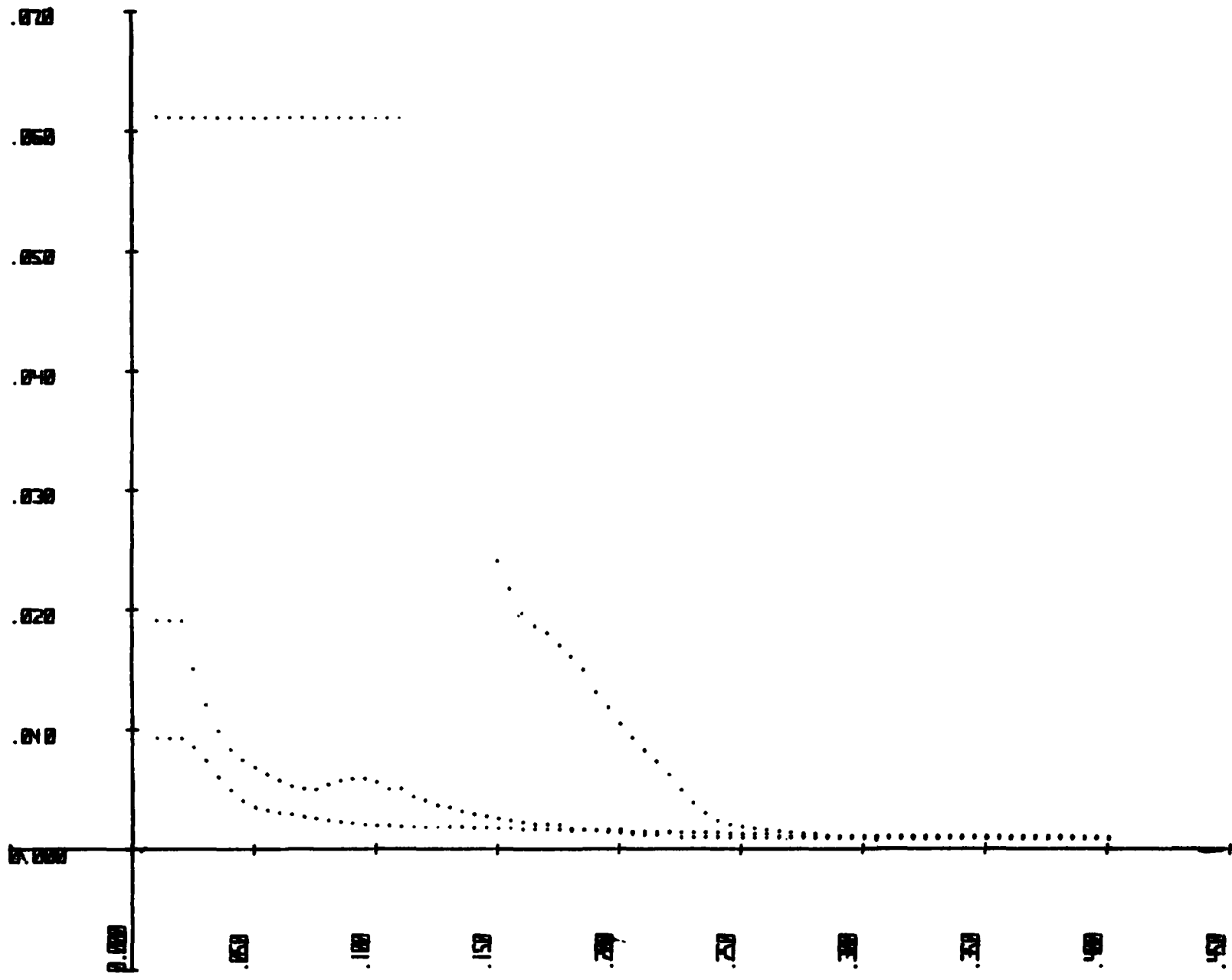


OUTBOARD ELEVON STEP RESPONSE

LEG
12/75

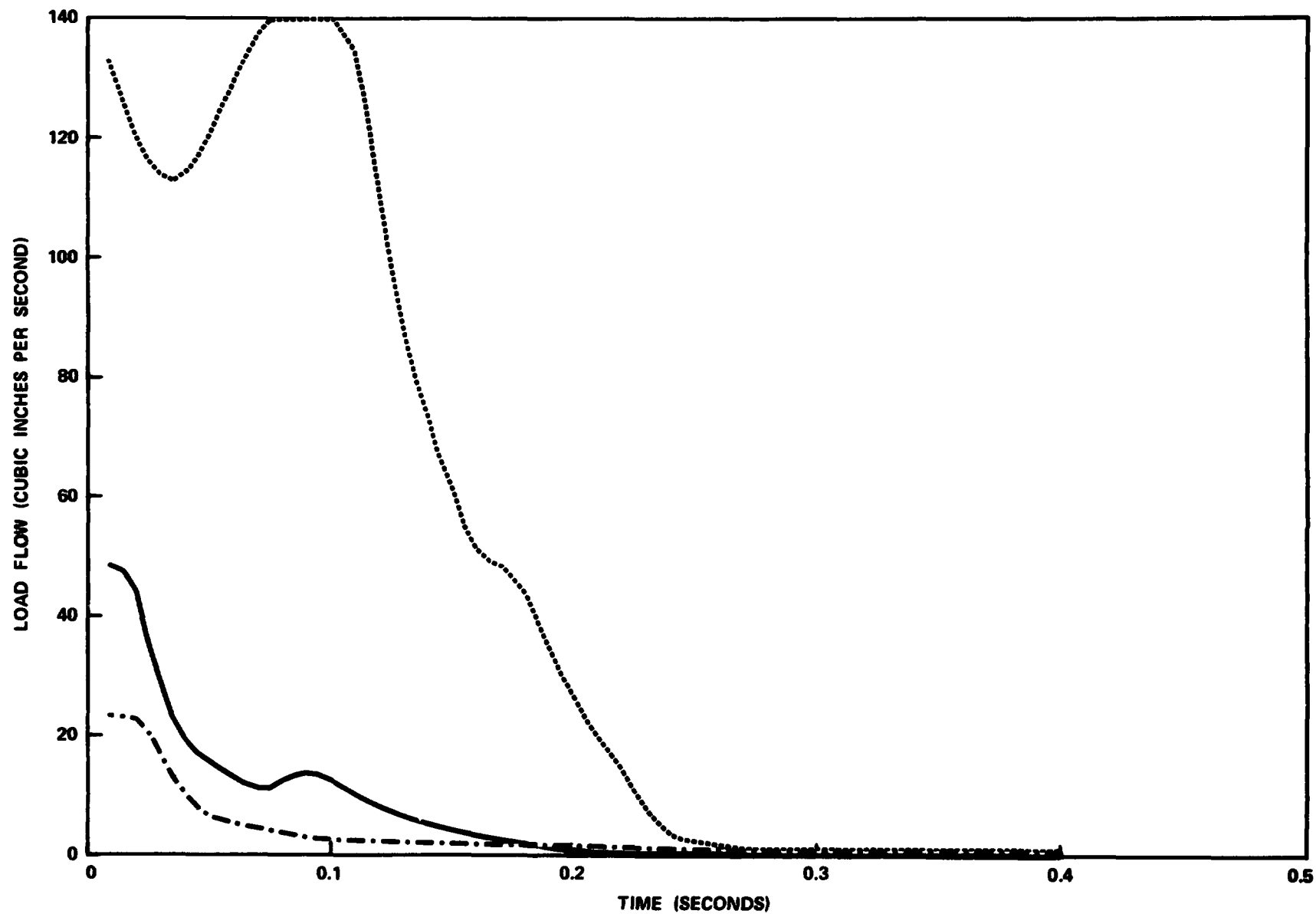


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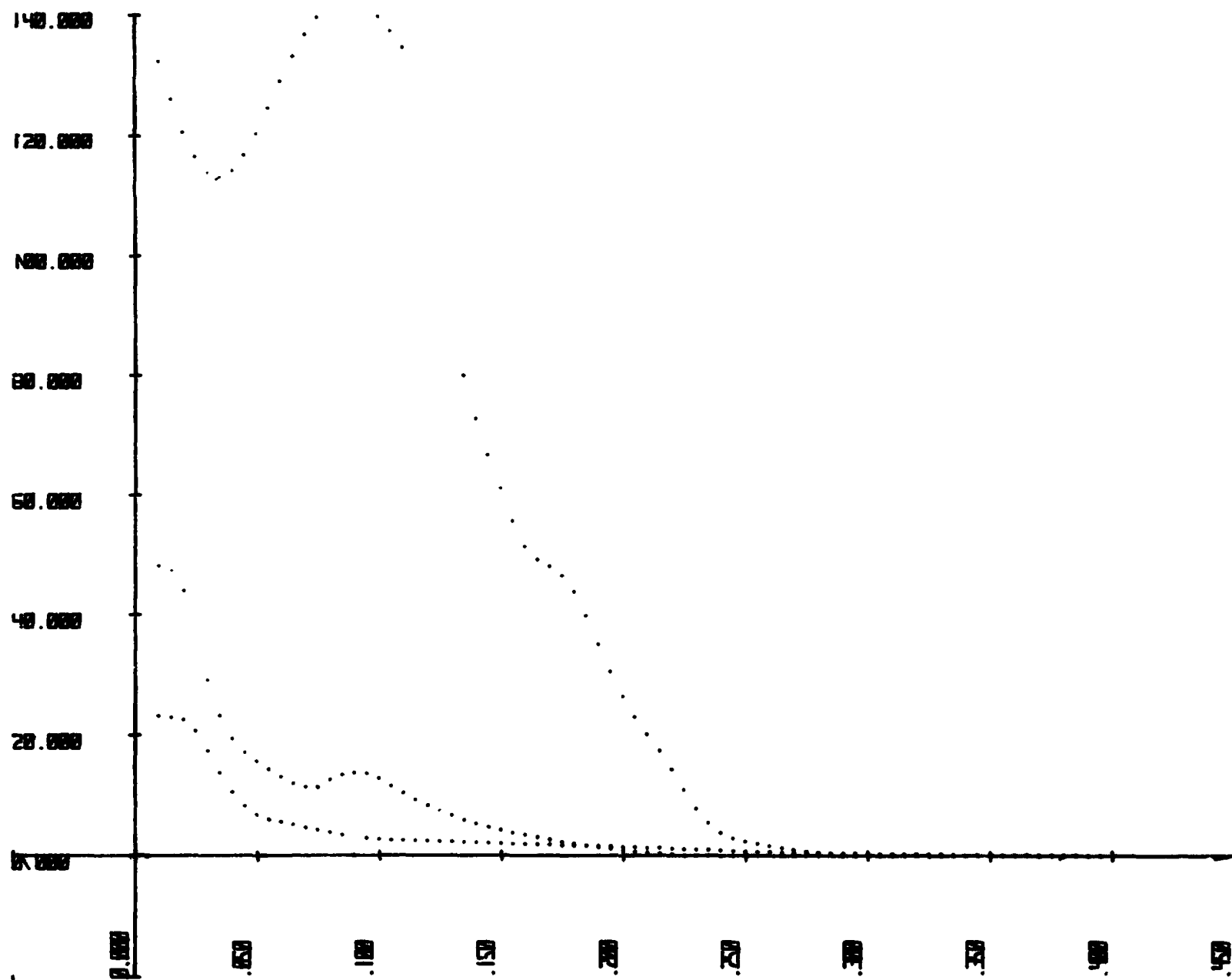


OUTBOARD ELEVON STEP RESPONSE

LEG
12/75

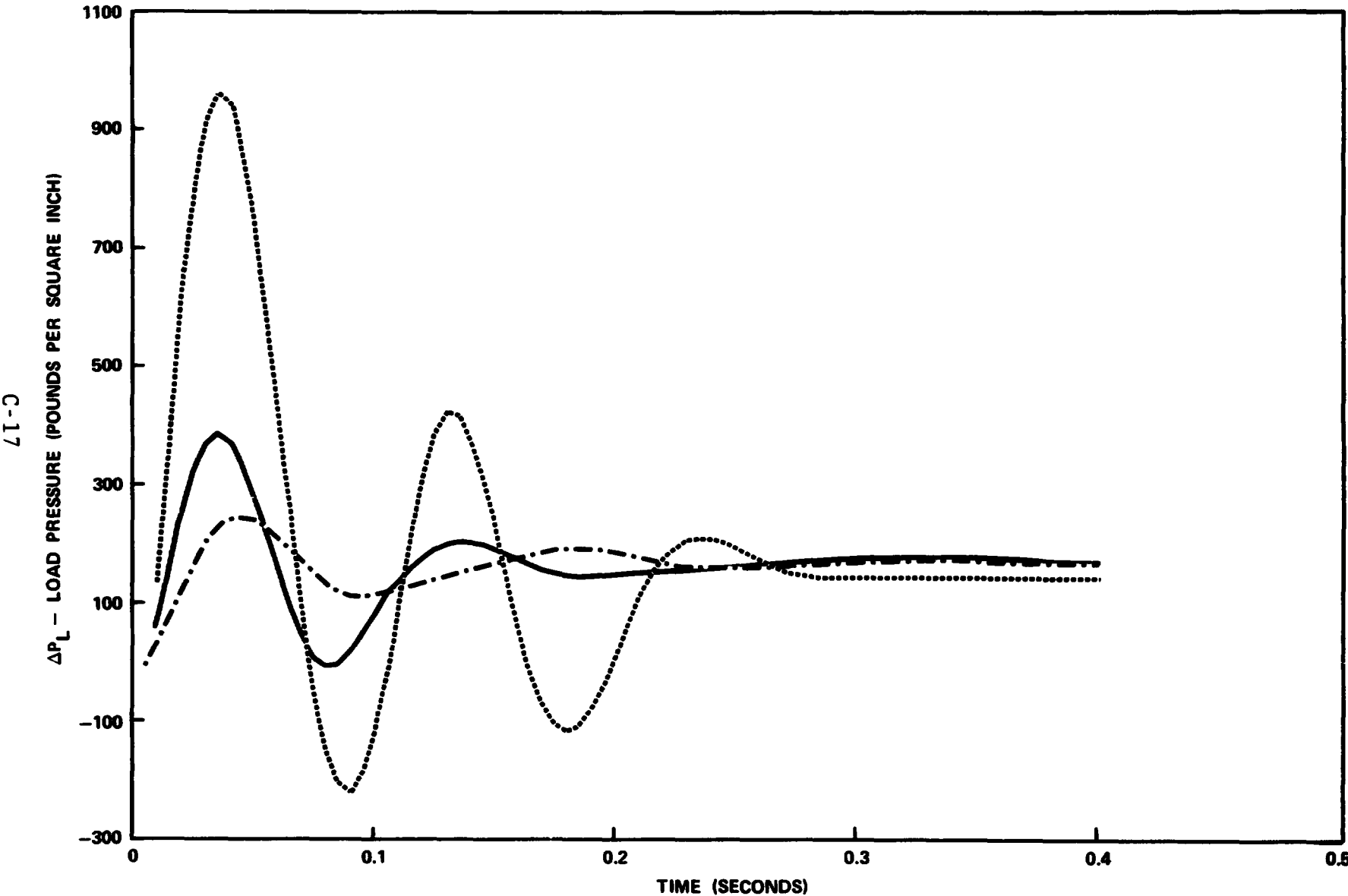


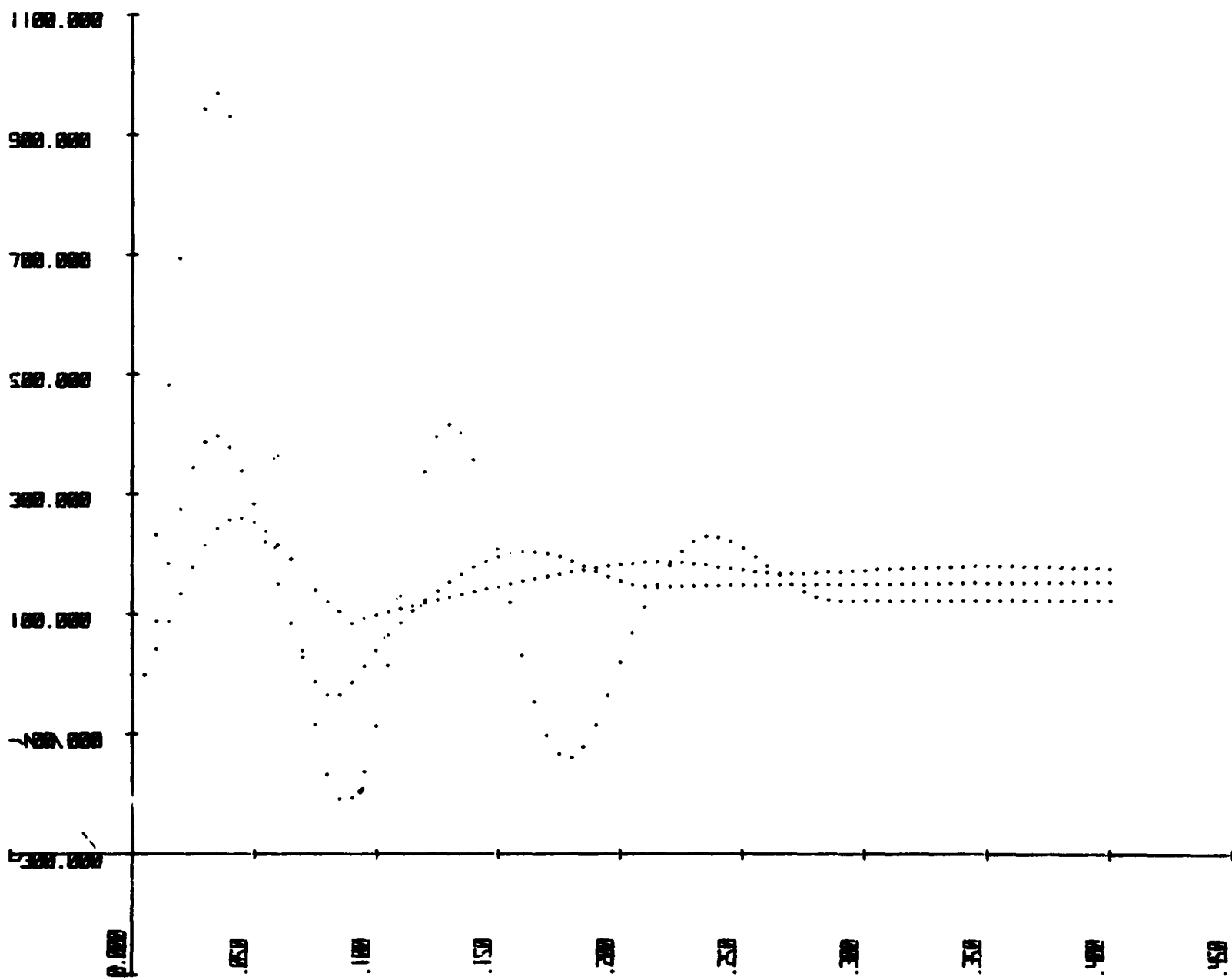
C-16



OUTBOARD ELEVON STEP RESPONSE

LEG
12/75

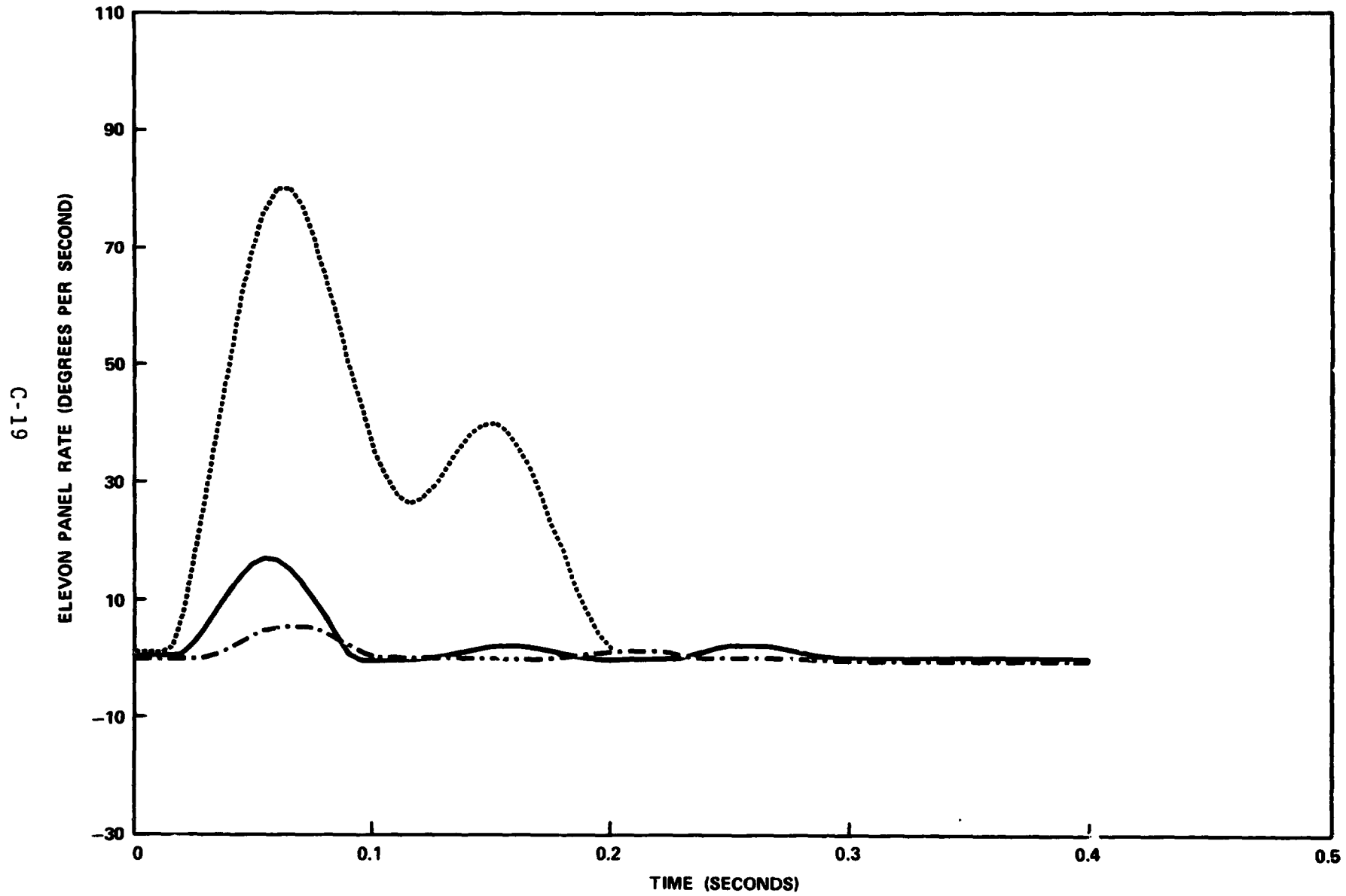




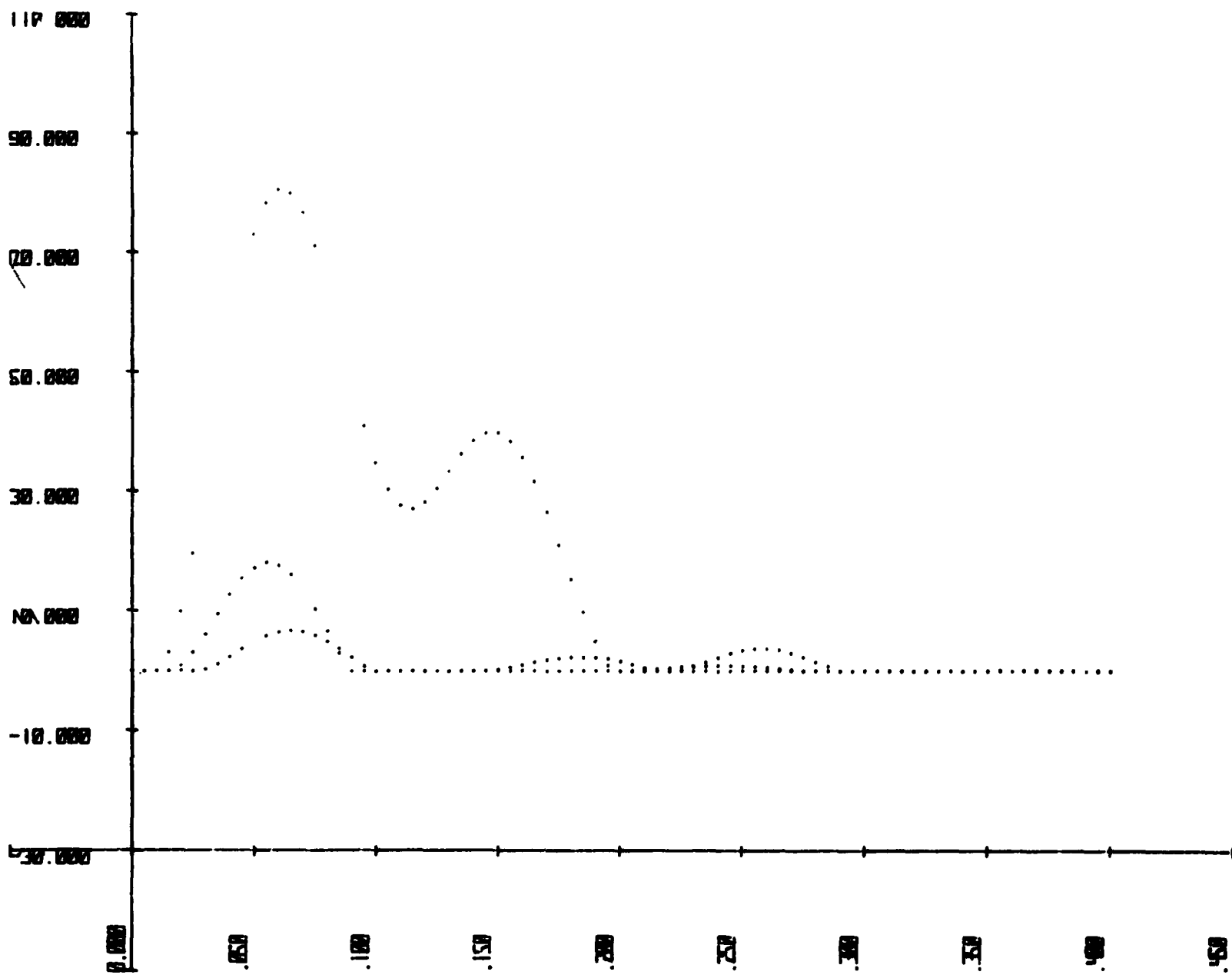
OUTBOARD ELEVON STEP RESPONSE

LEG

12/75

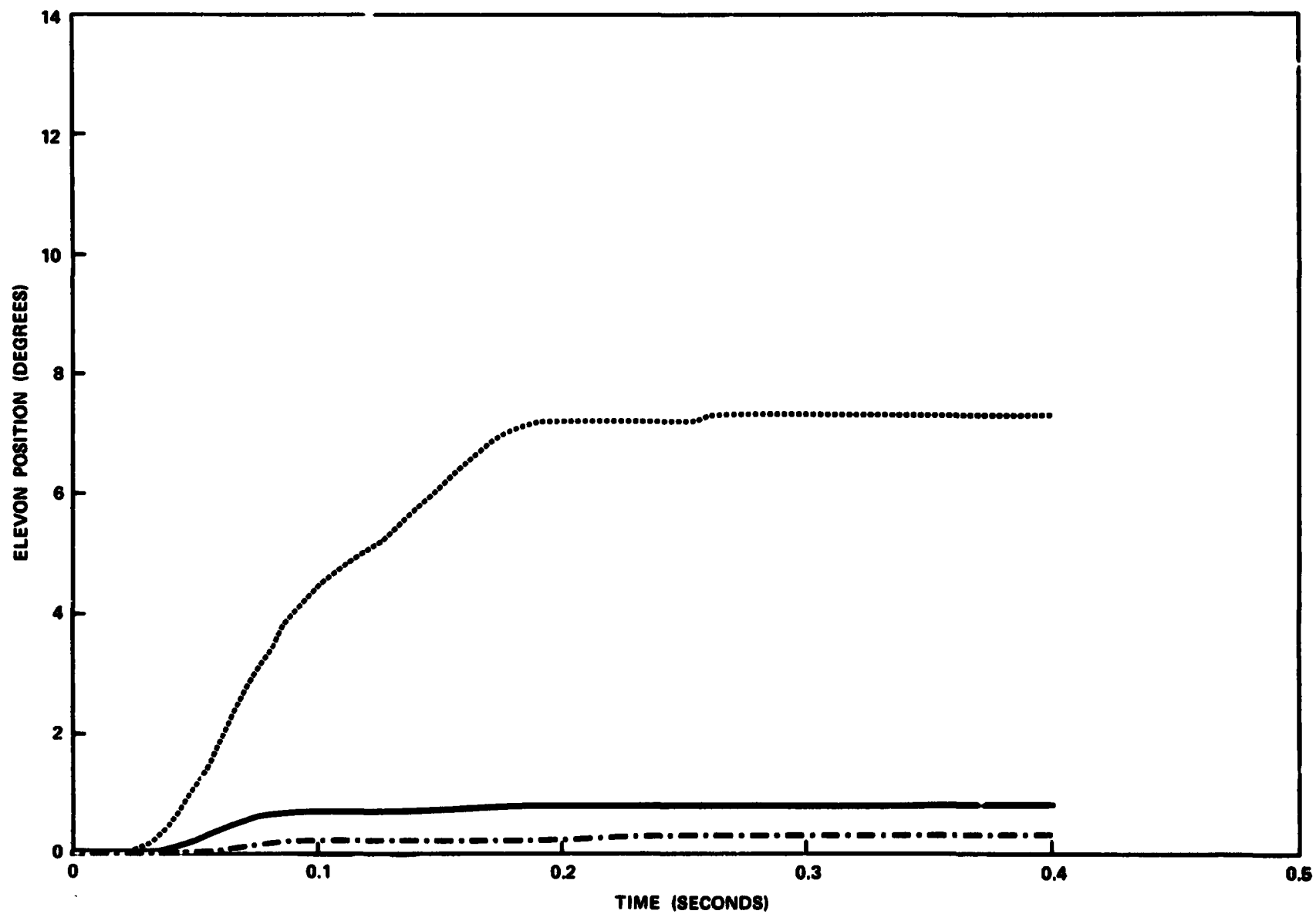


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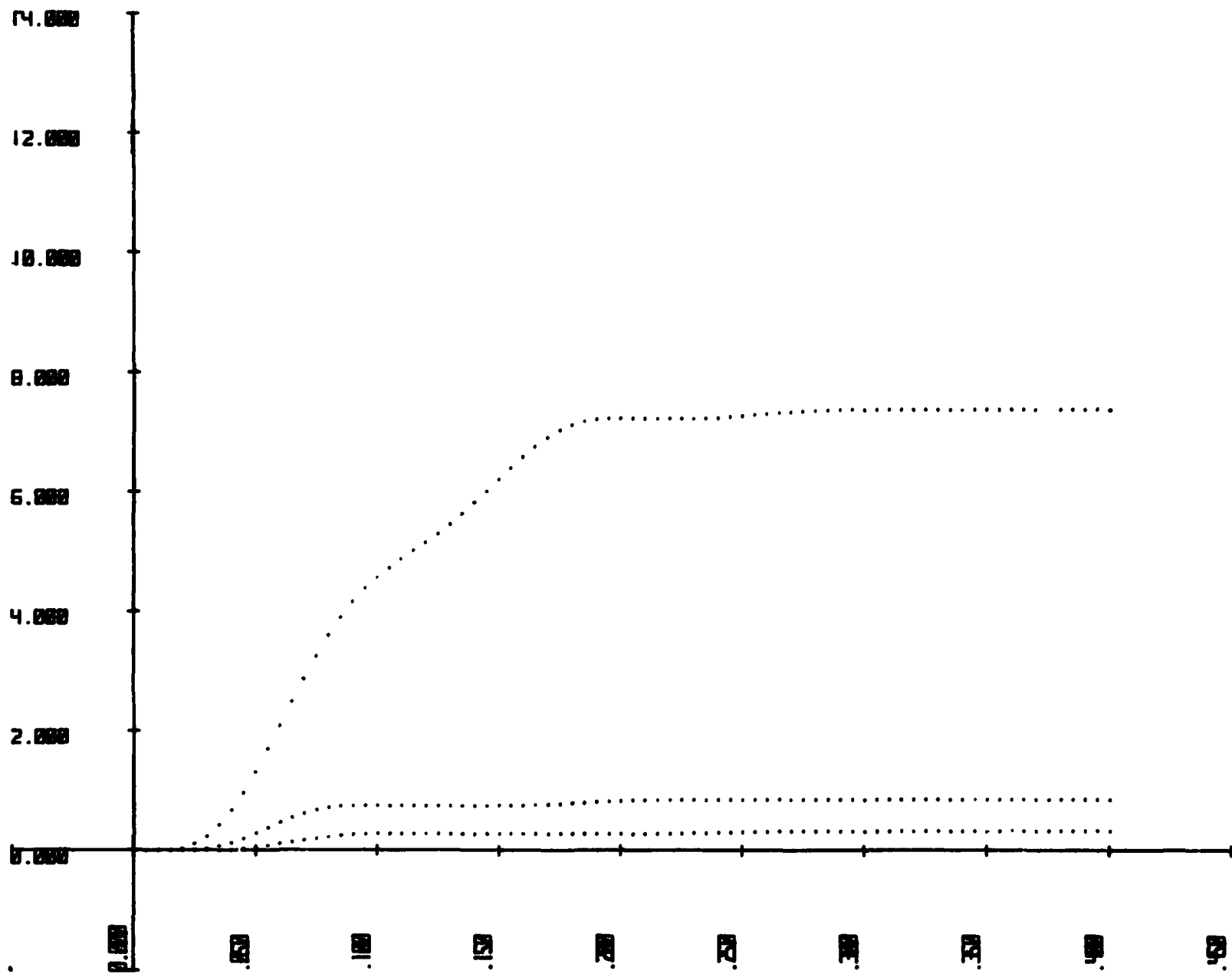


OUTBOARD ELEVON STEP RESPONSE

LEG
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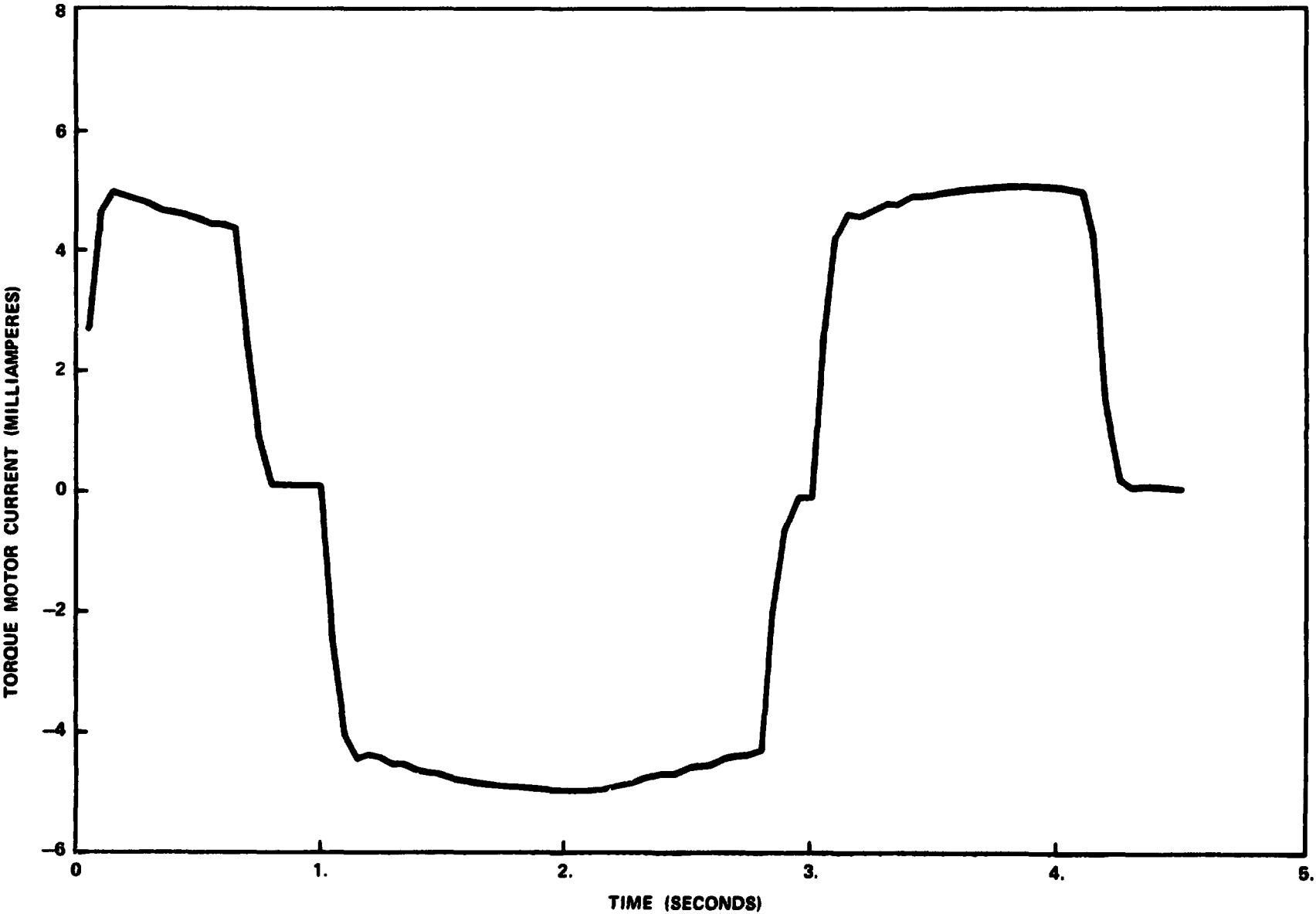
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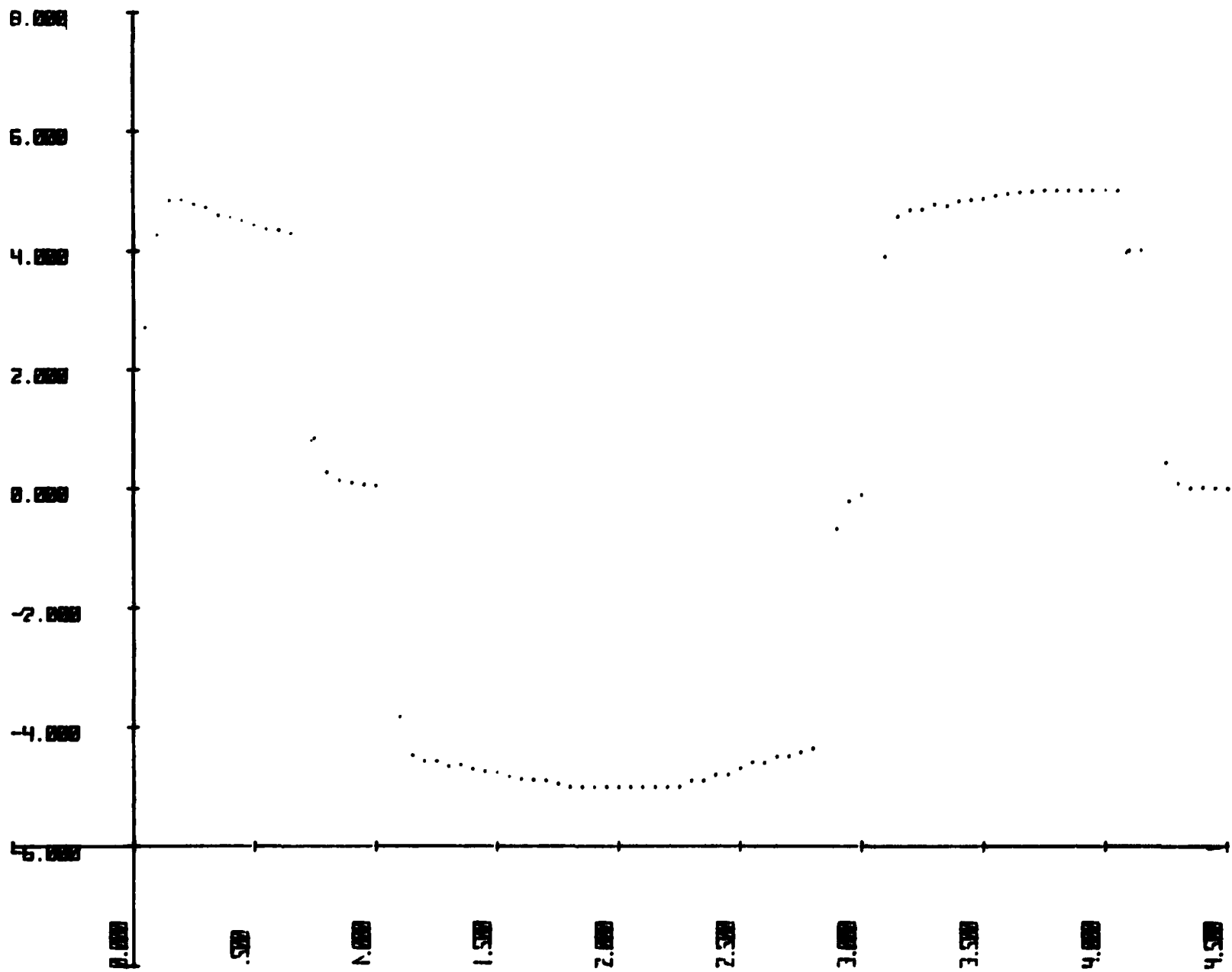
OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

12/75

C-23

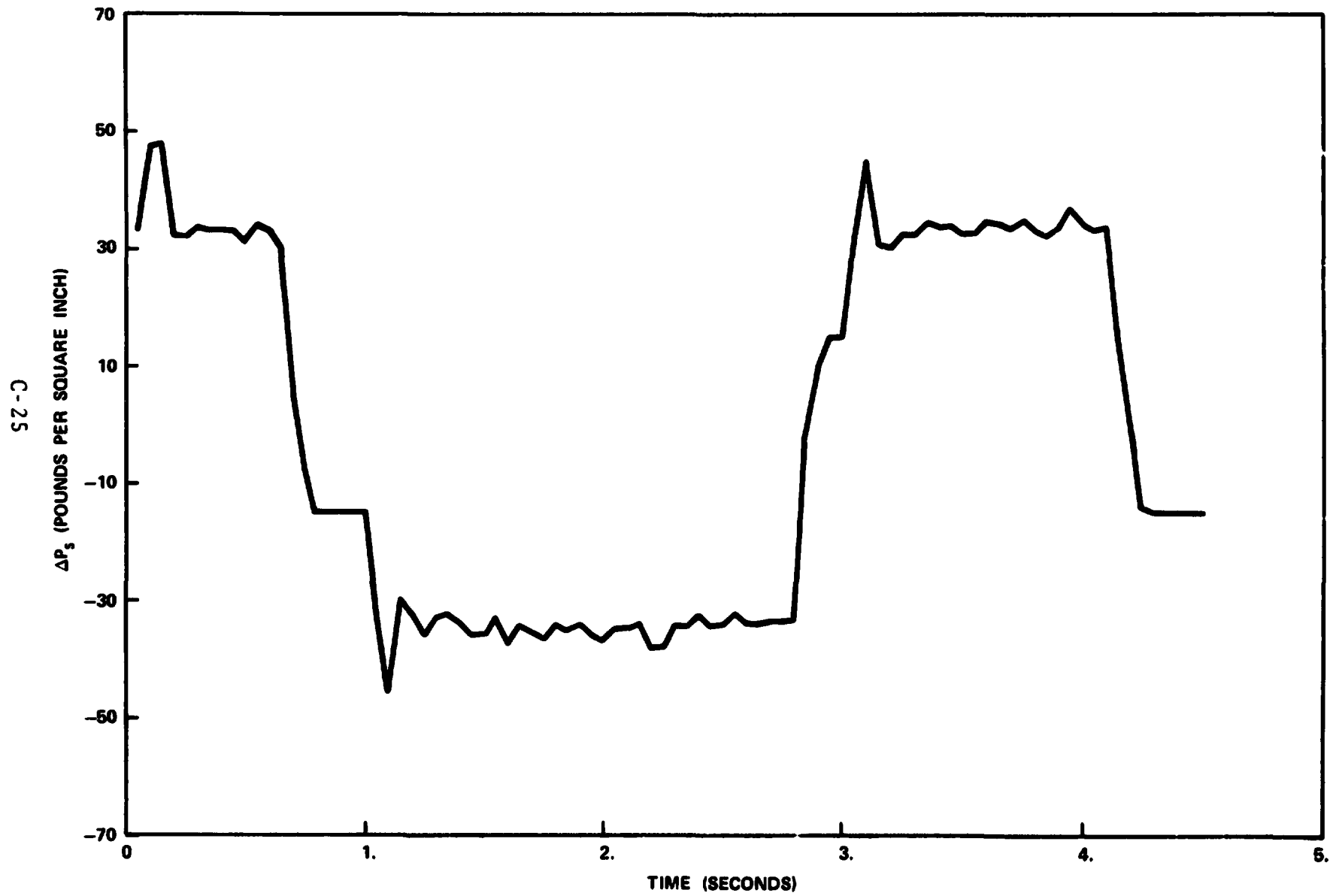


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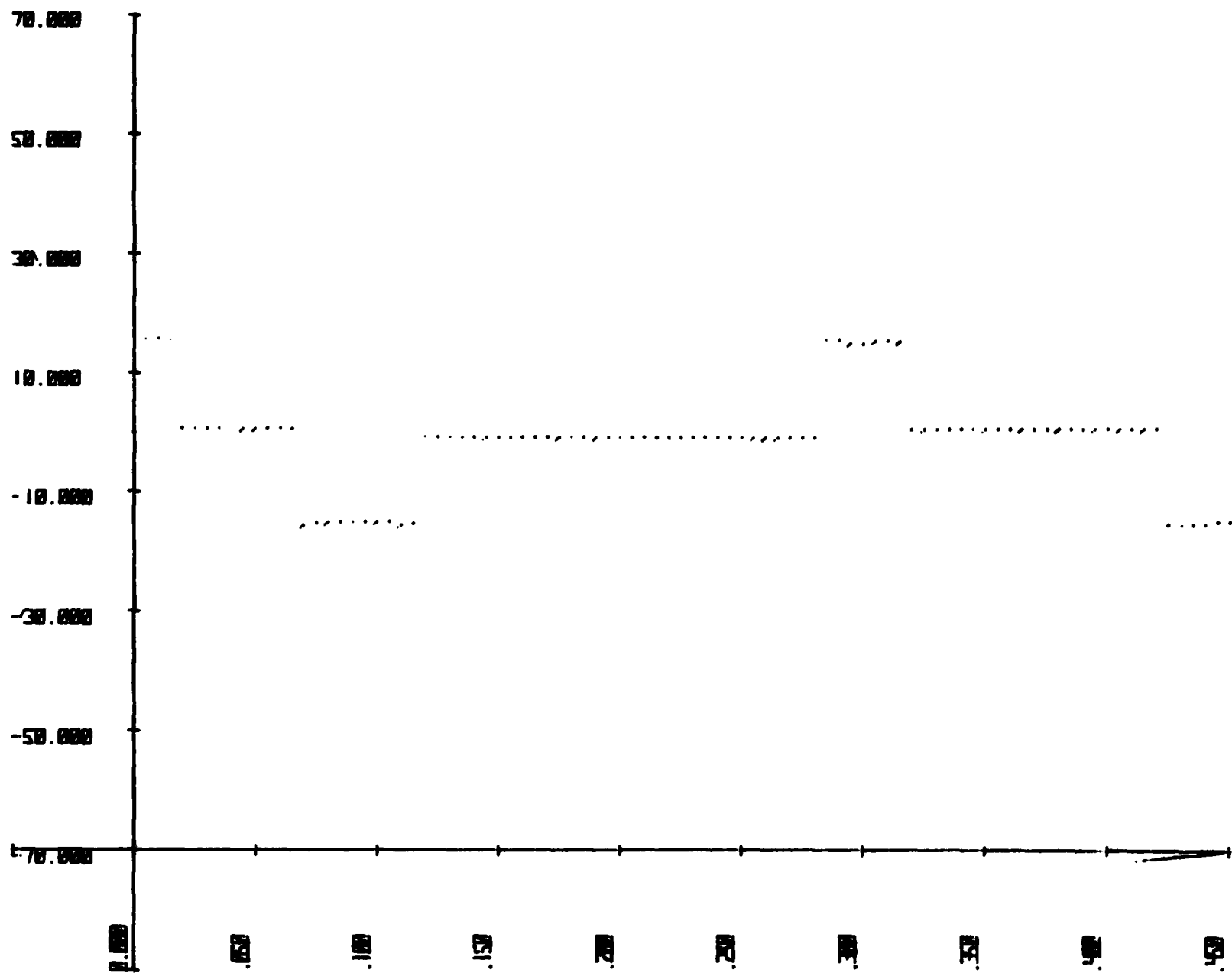


OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

12/75

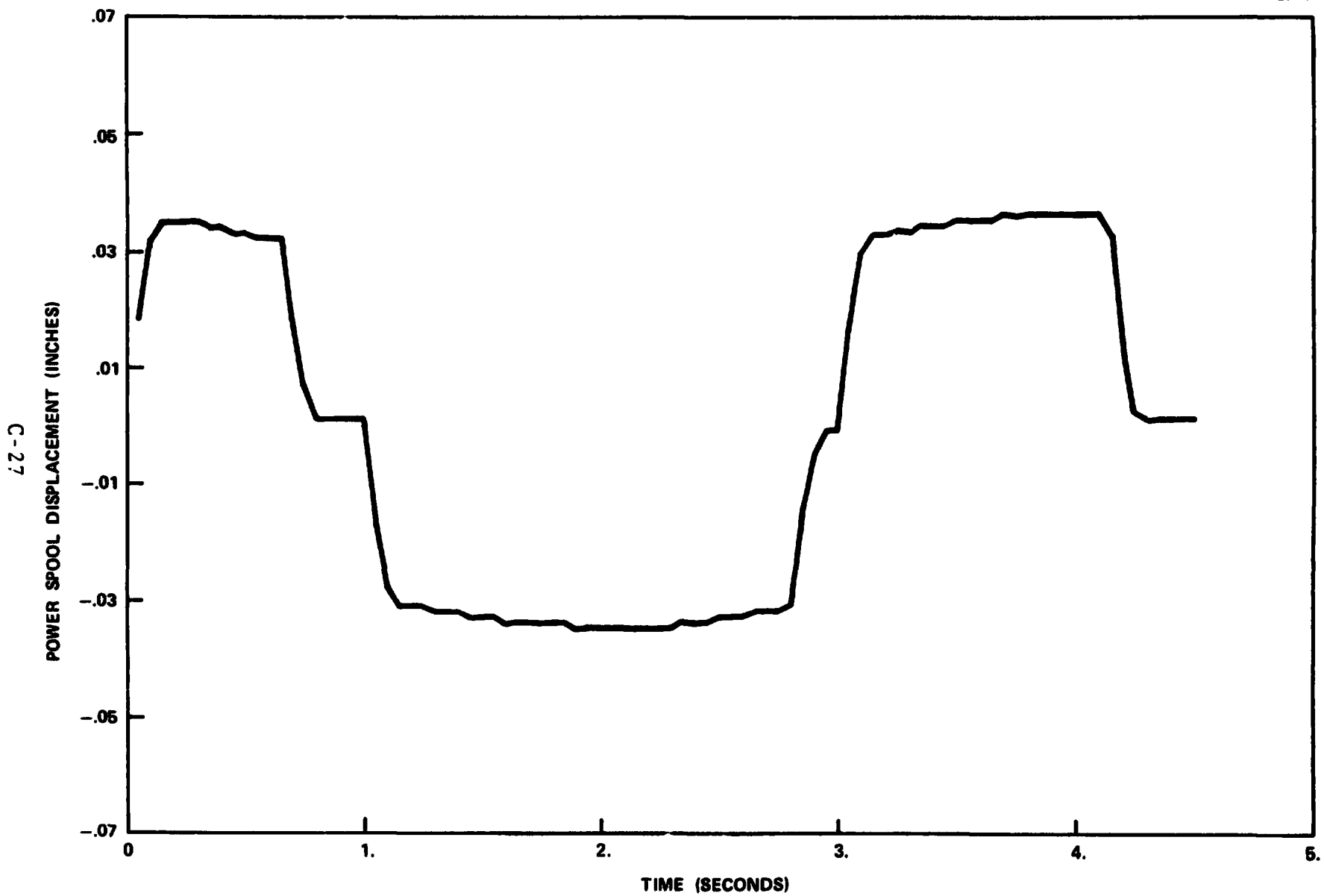


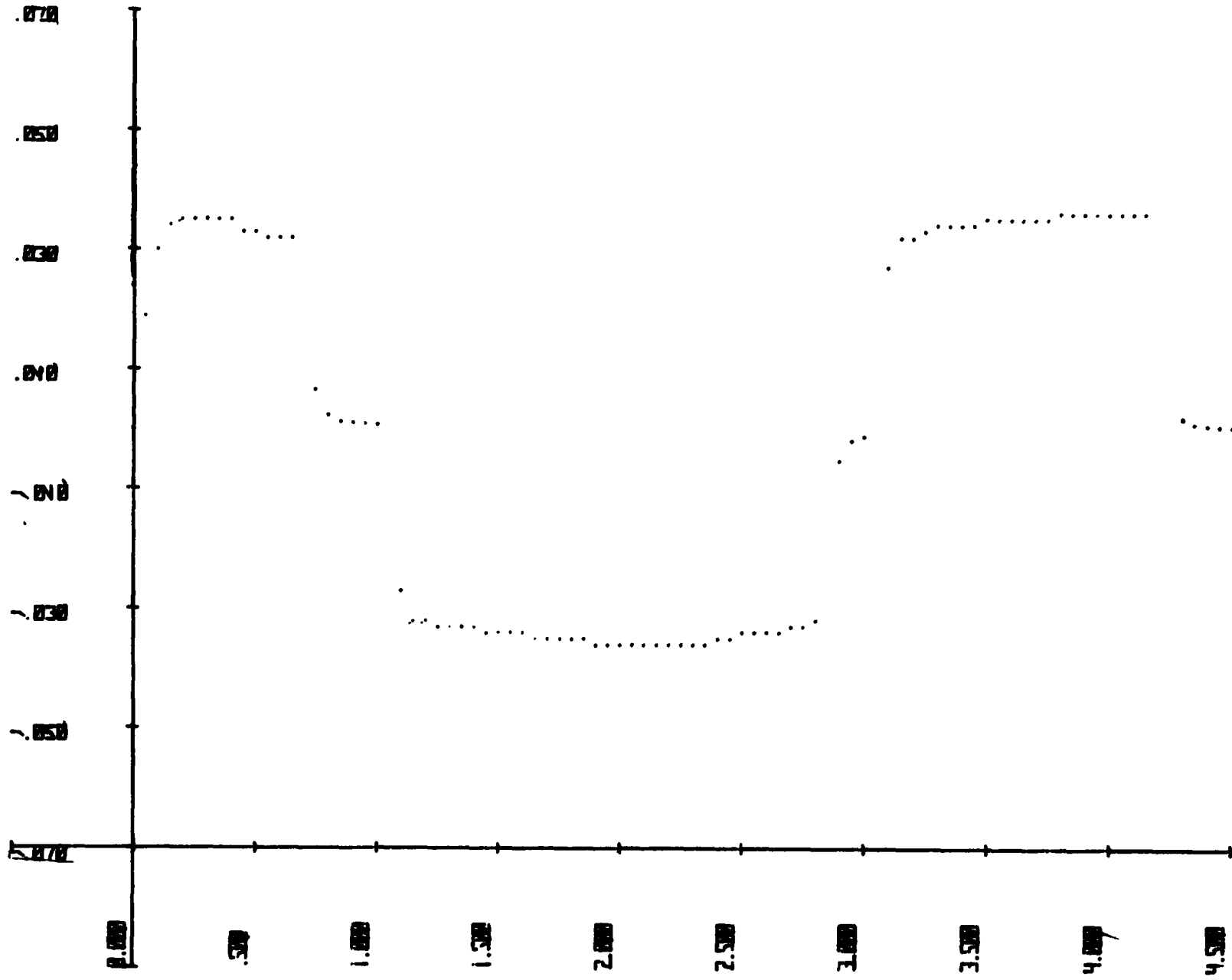
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OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

LEG
12/75

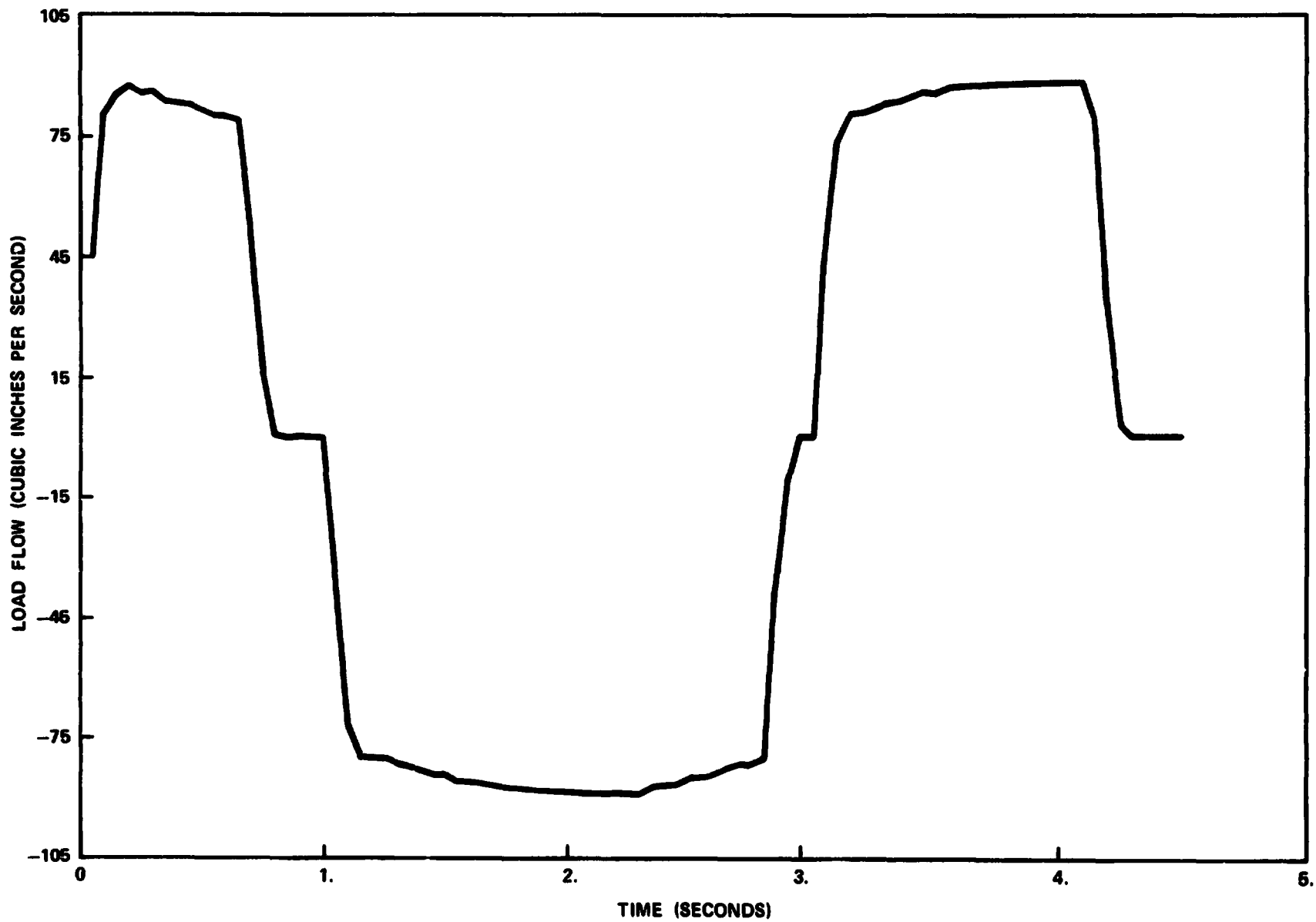




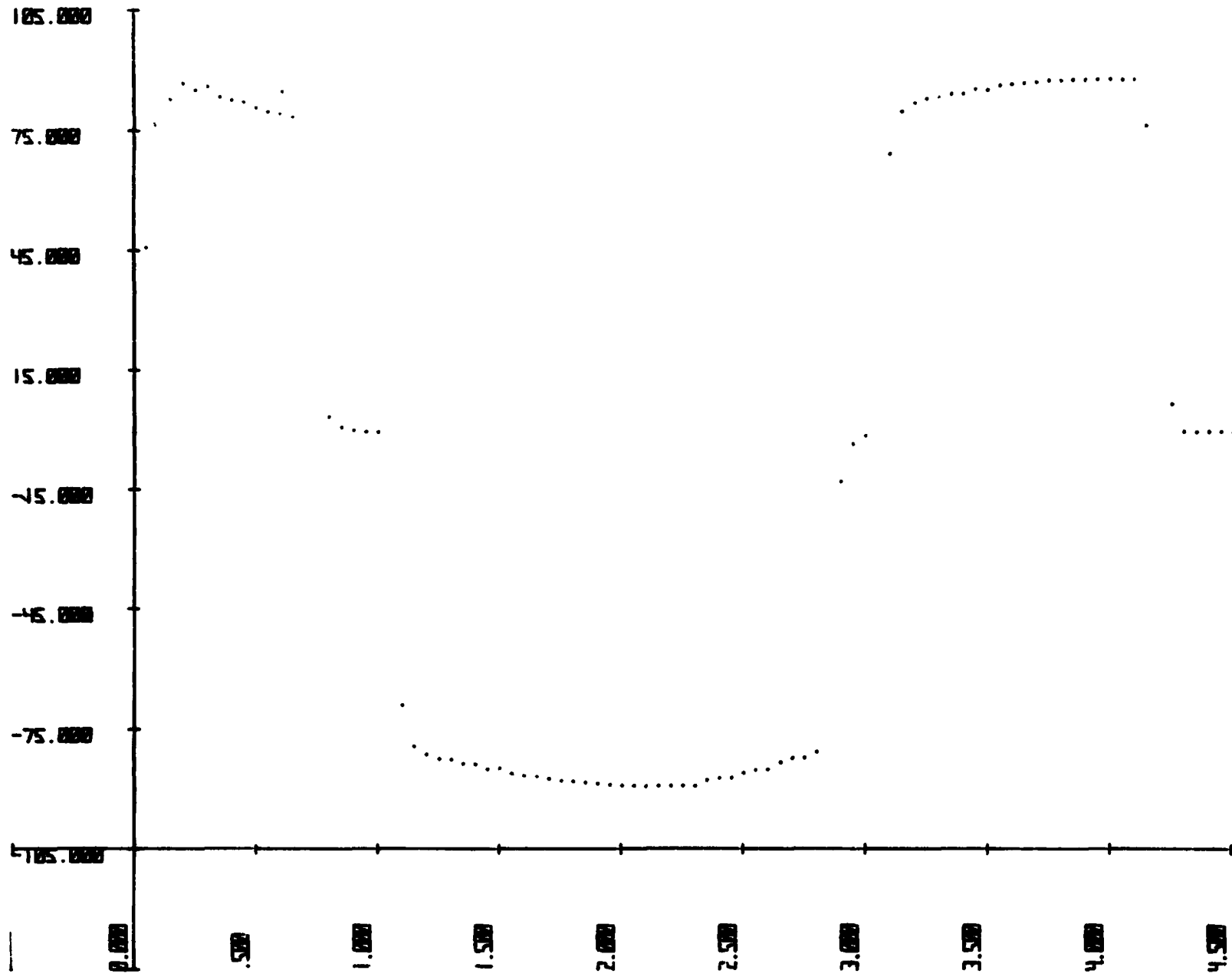
OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

LEG
12/75

C-29

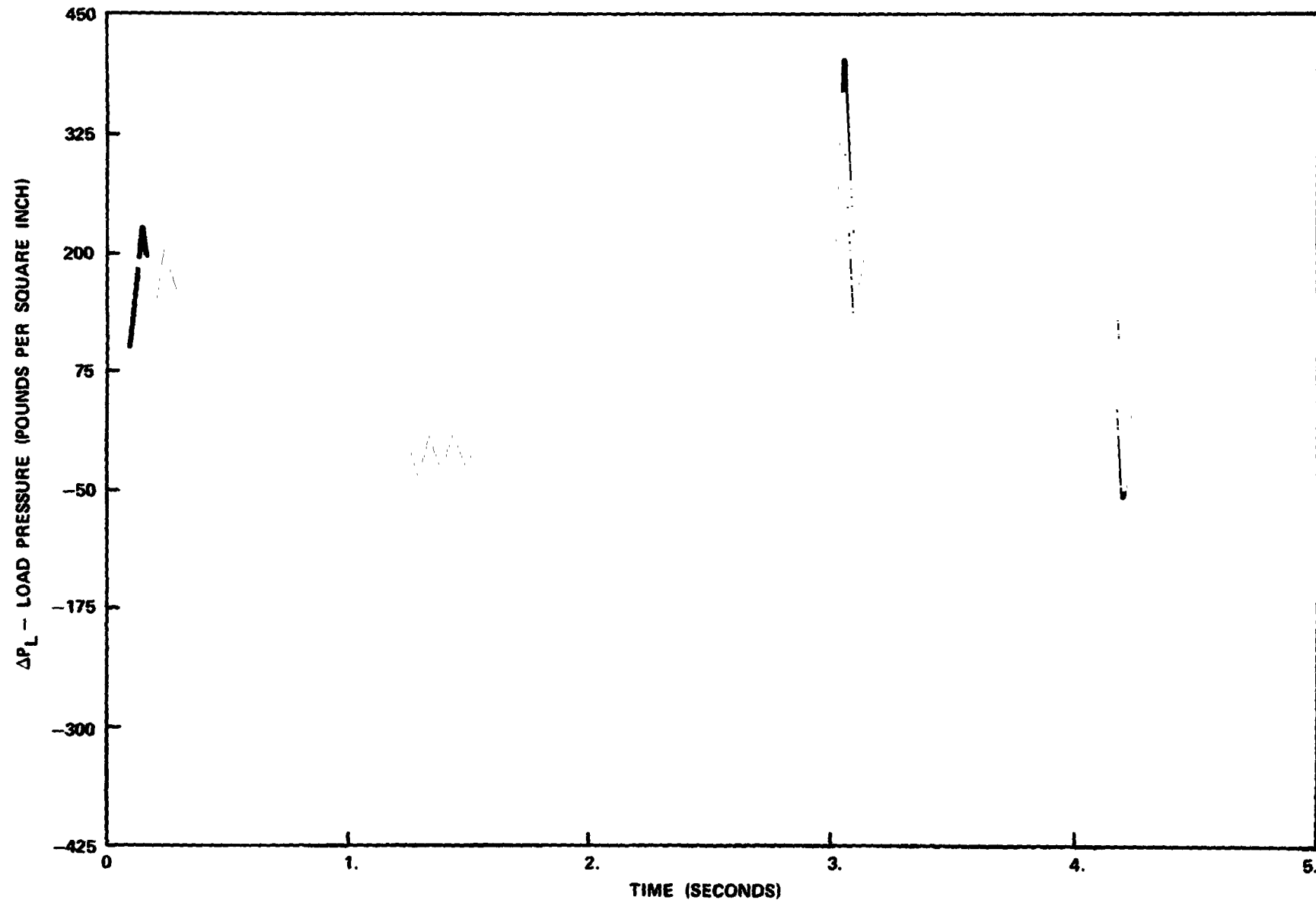


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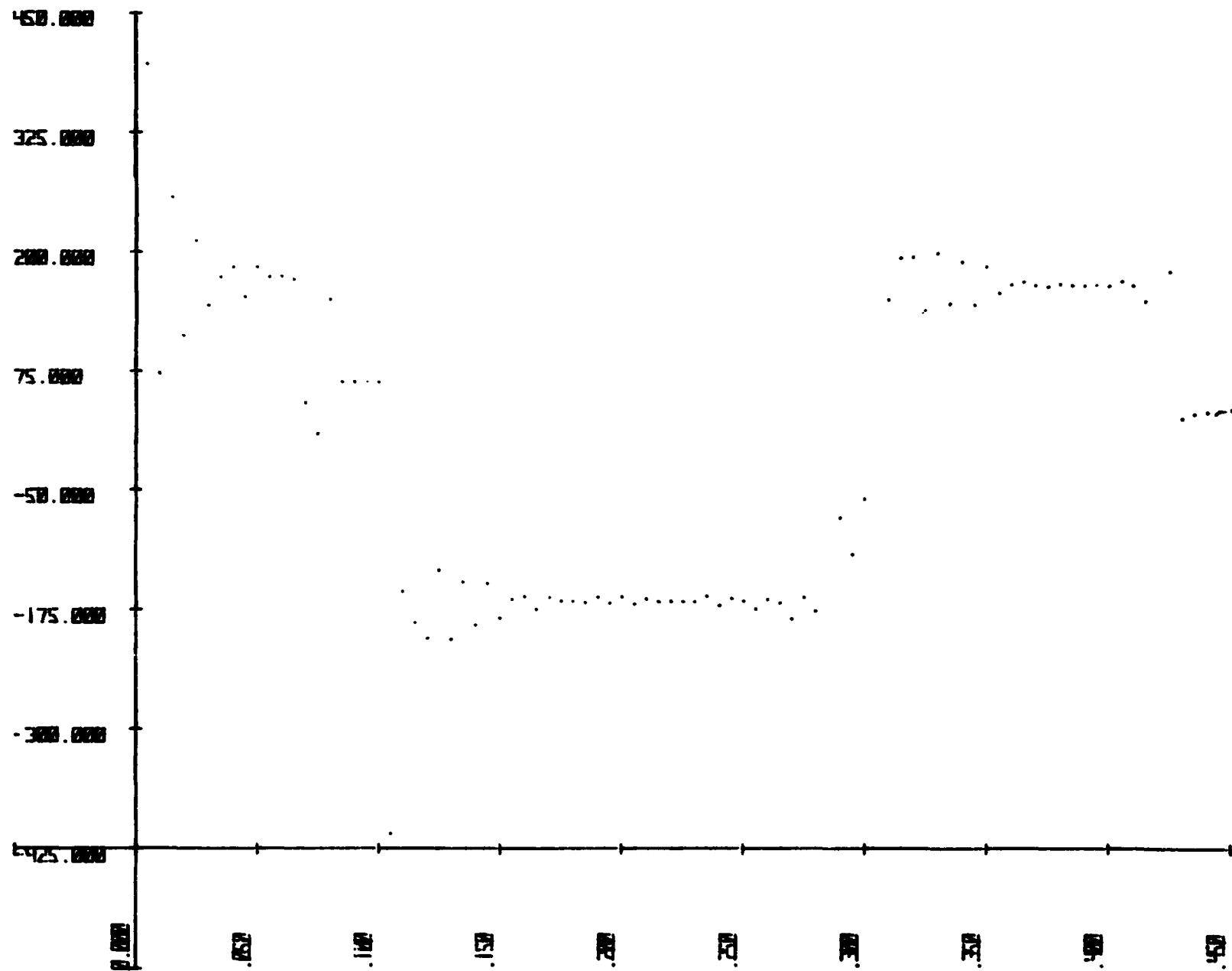


OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

LEG
12/75



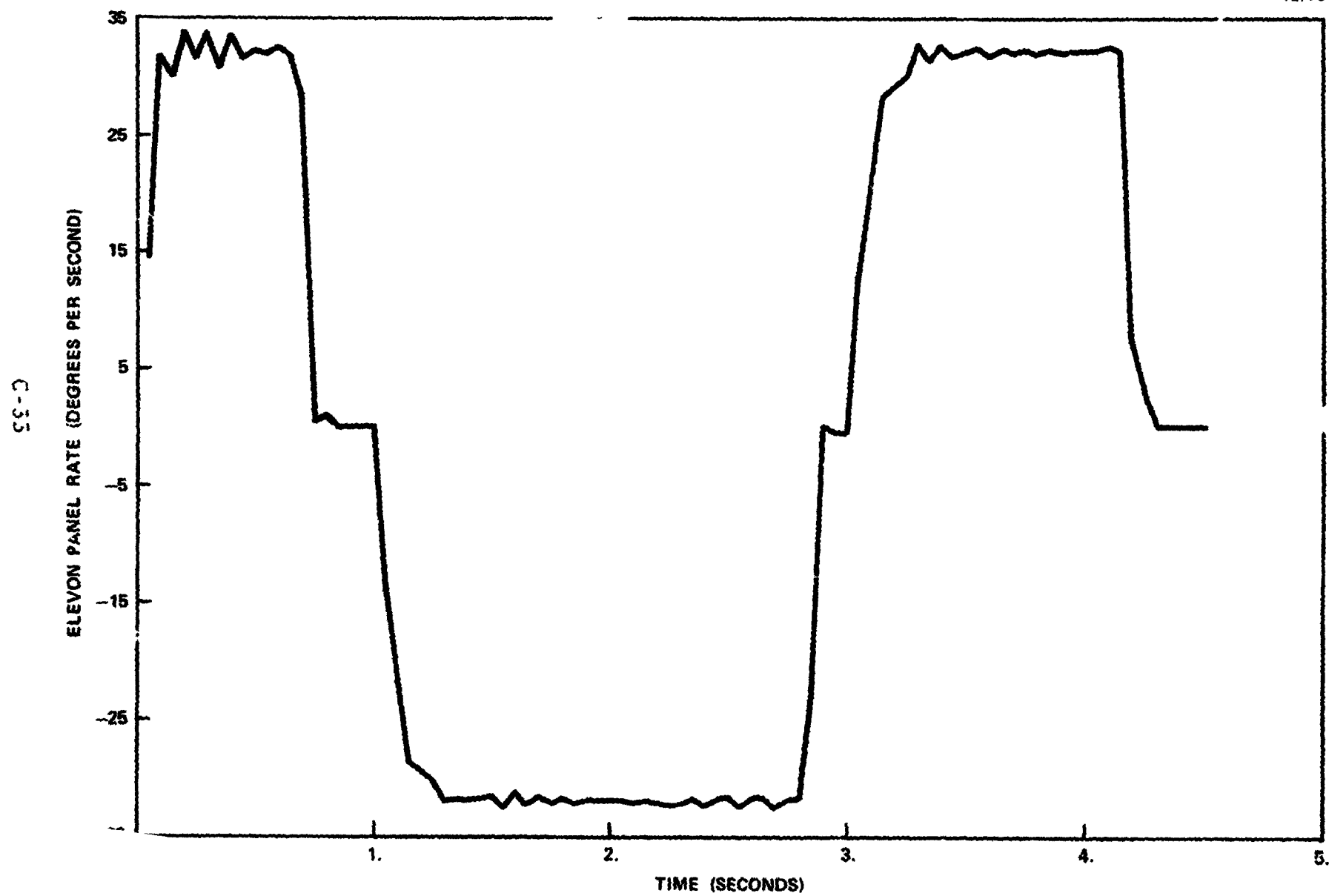
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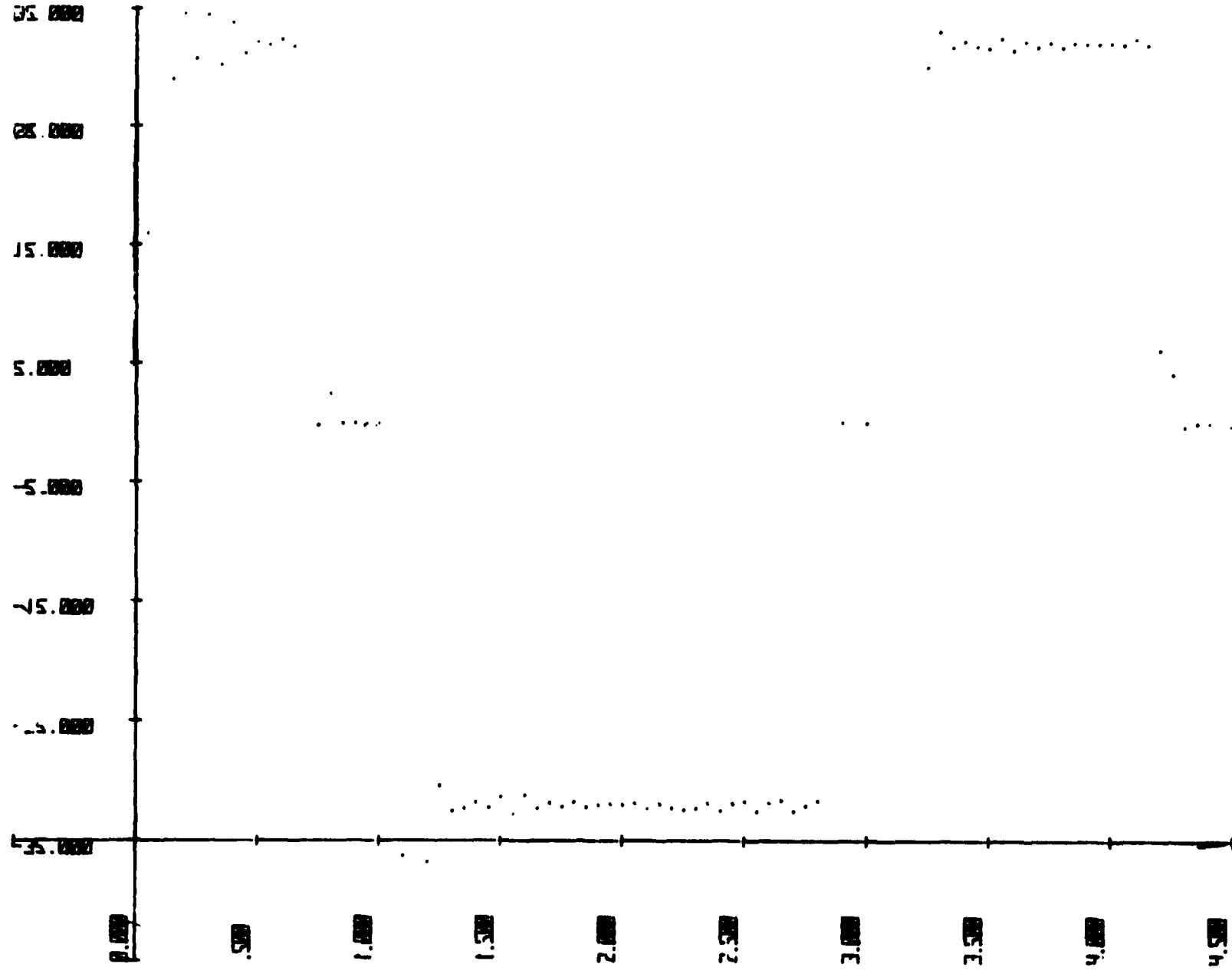
OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

LEG

12/75

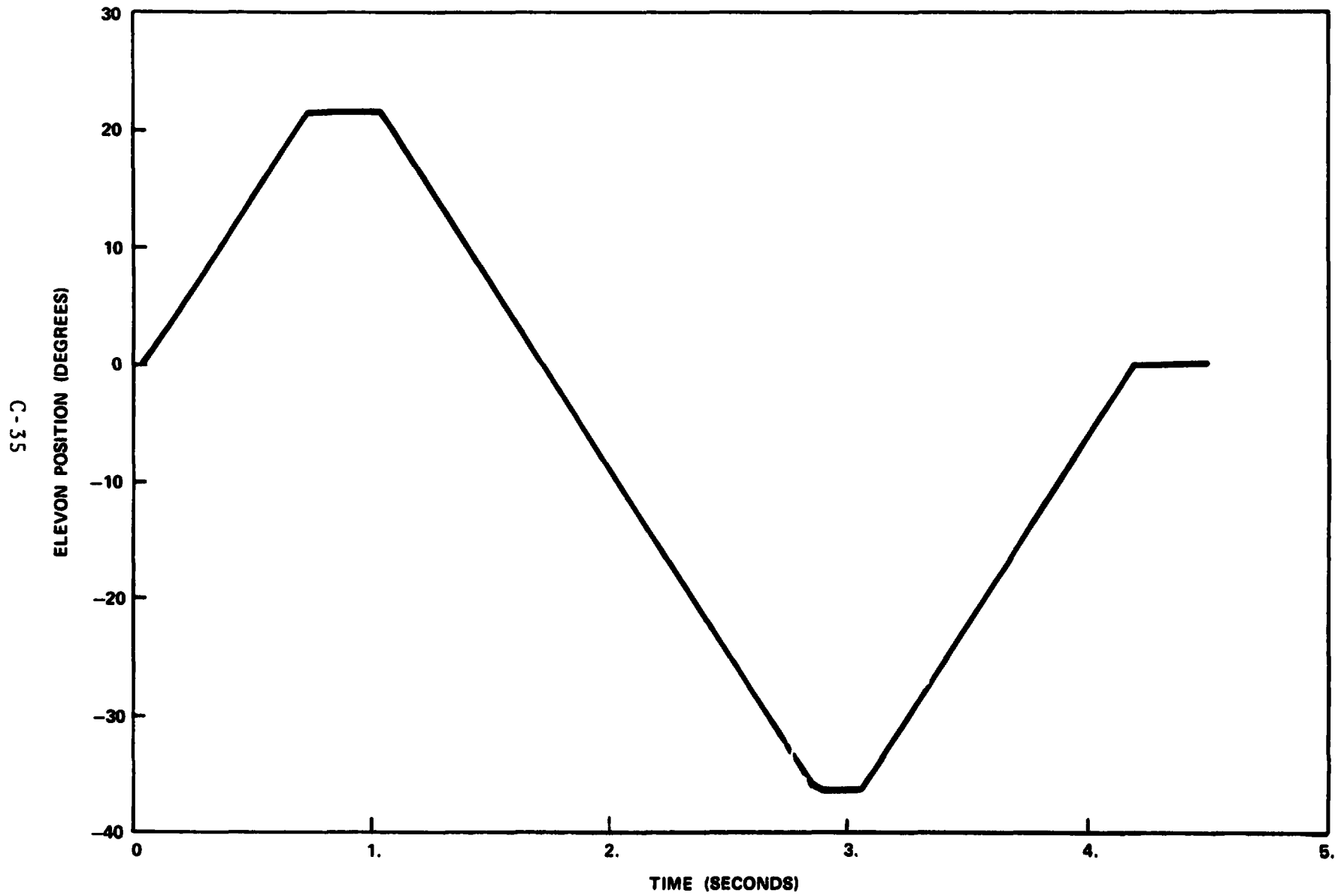


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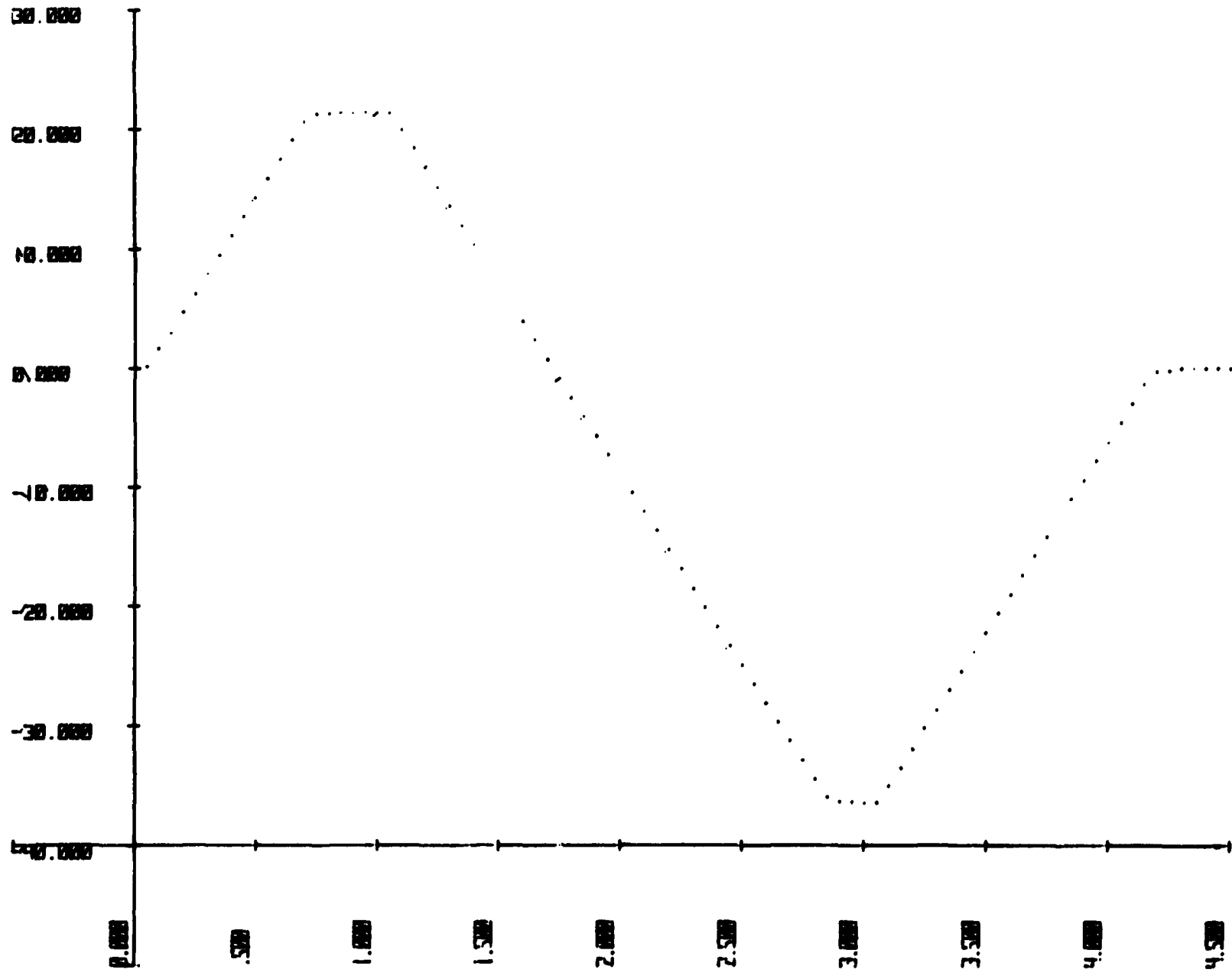


OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

LOG
12/75

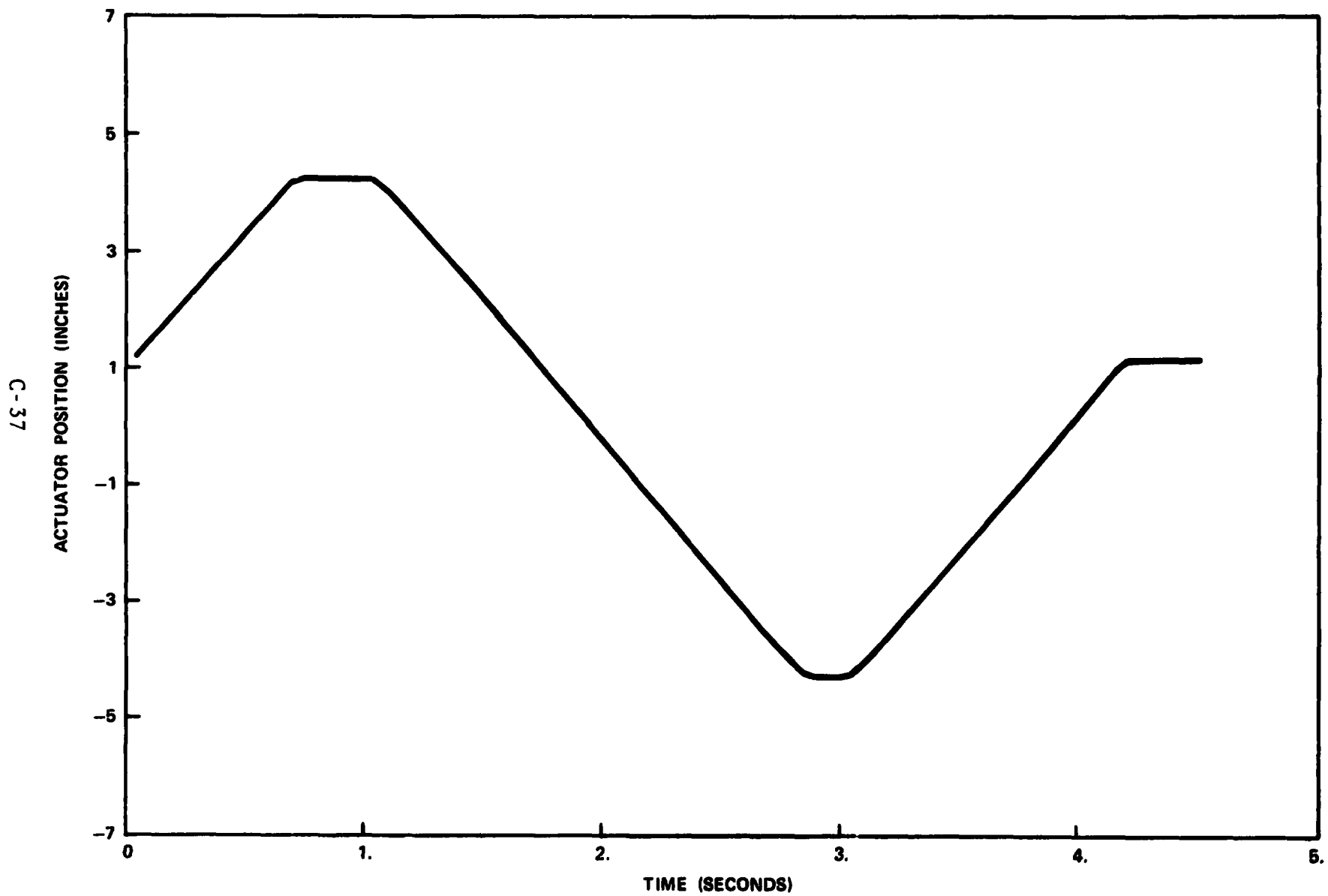


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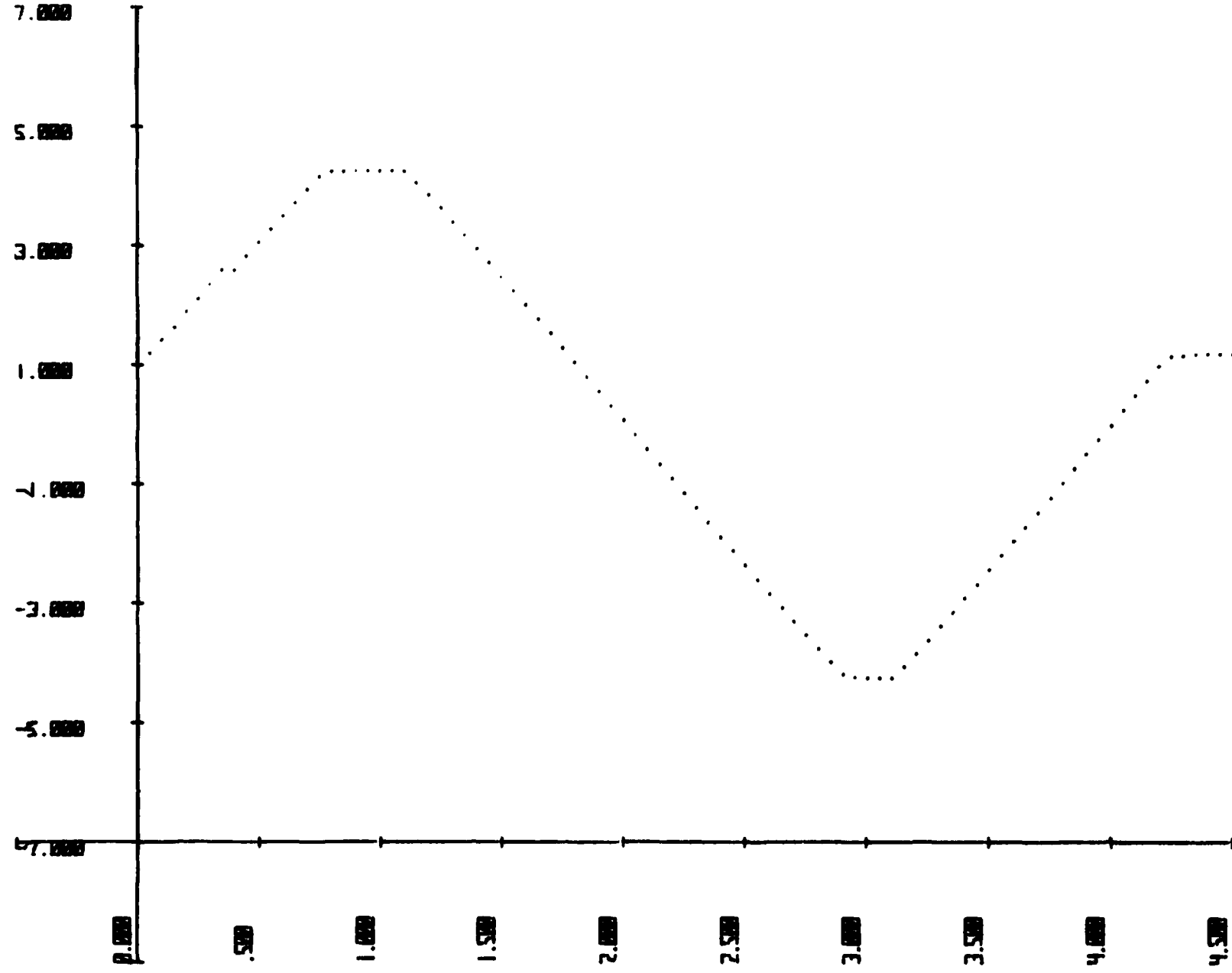


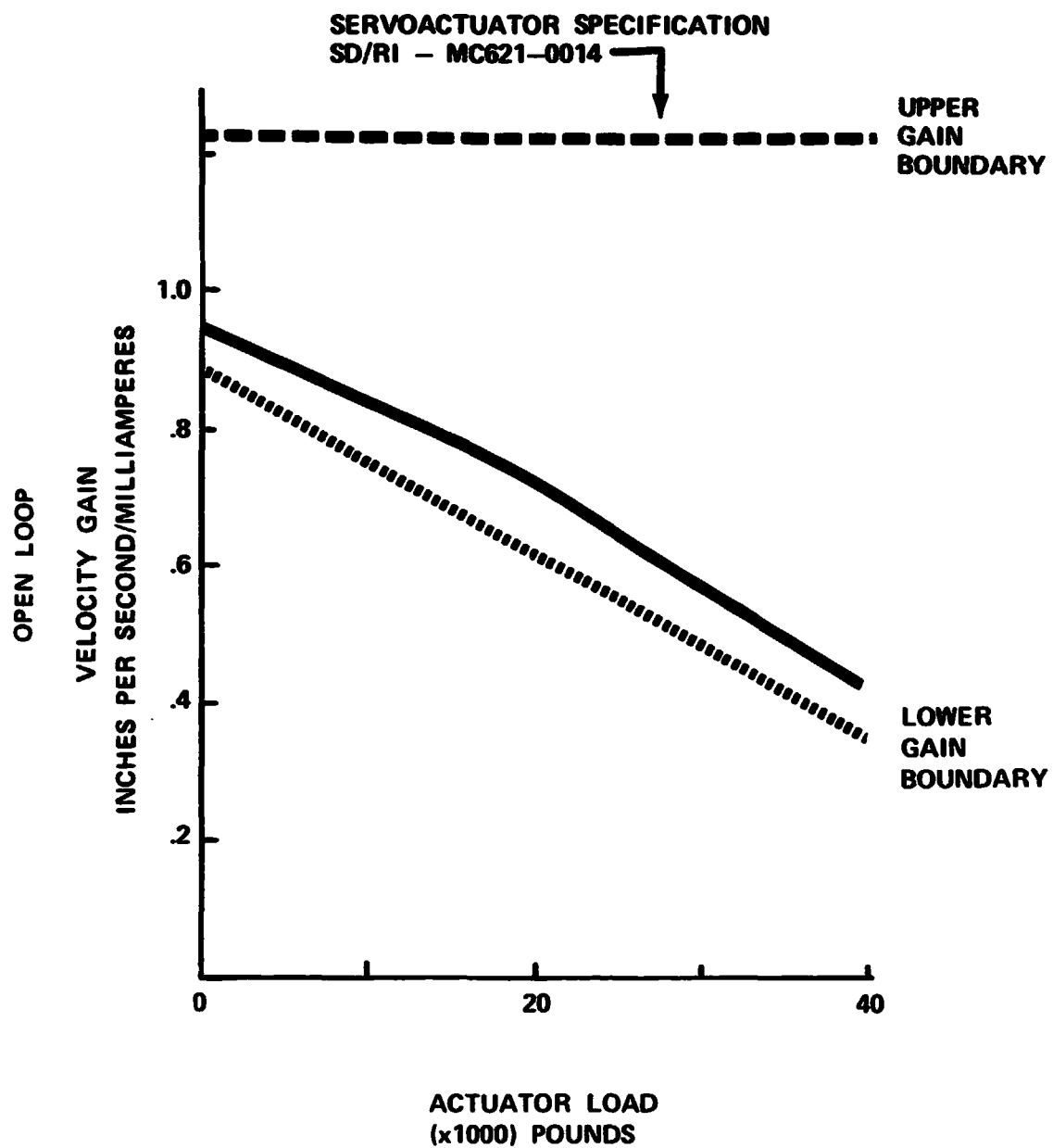
OUTBOARD ELEVON LARGE AMPLITUDE RESPONSE

1-50
12/75



C-38

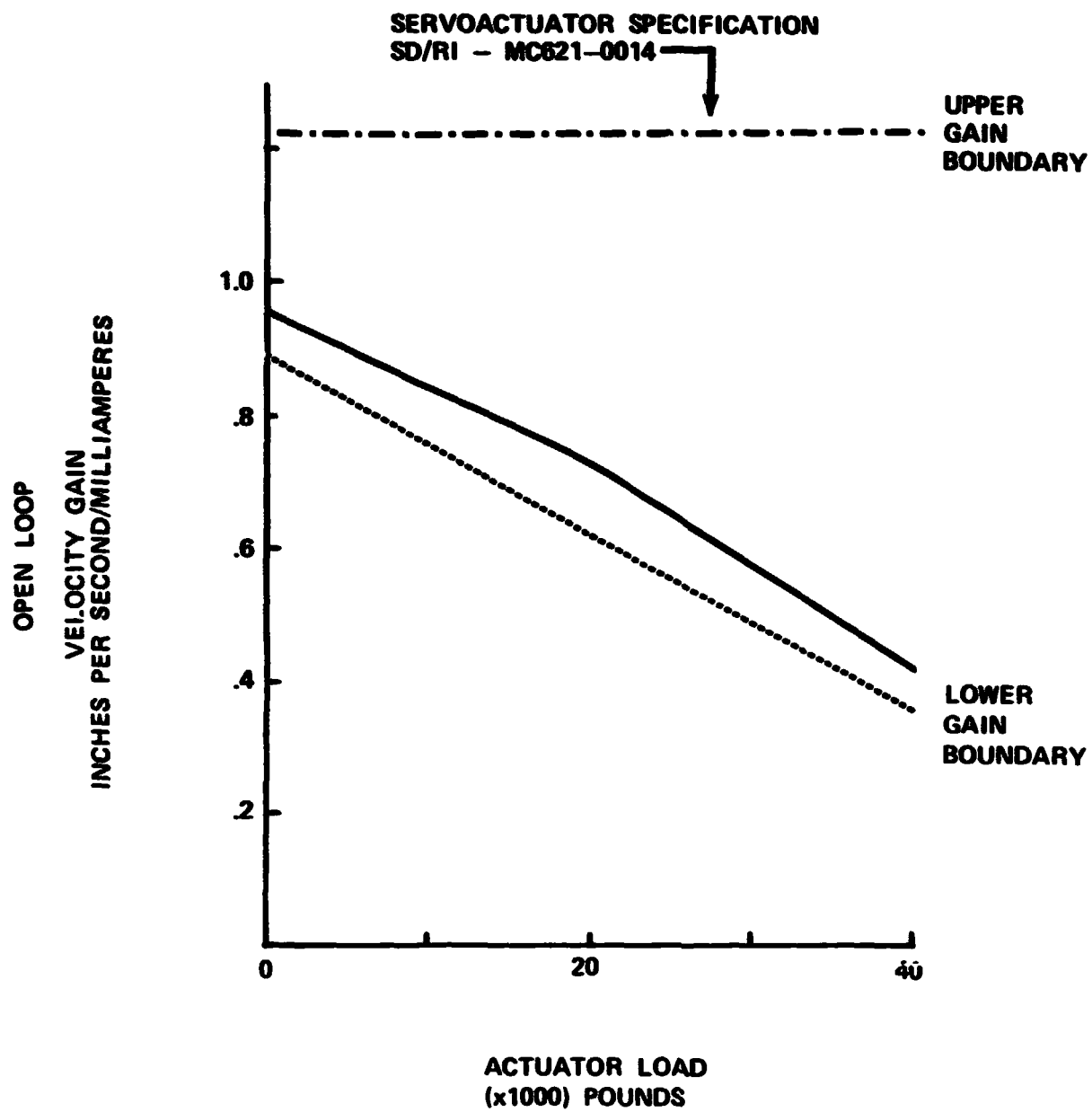




MODEL 1
OUTBOARD ELEVON



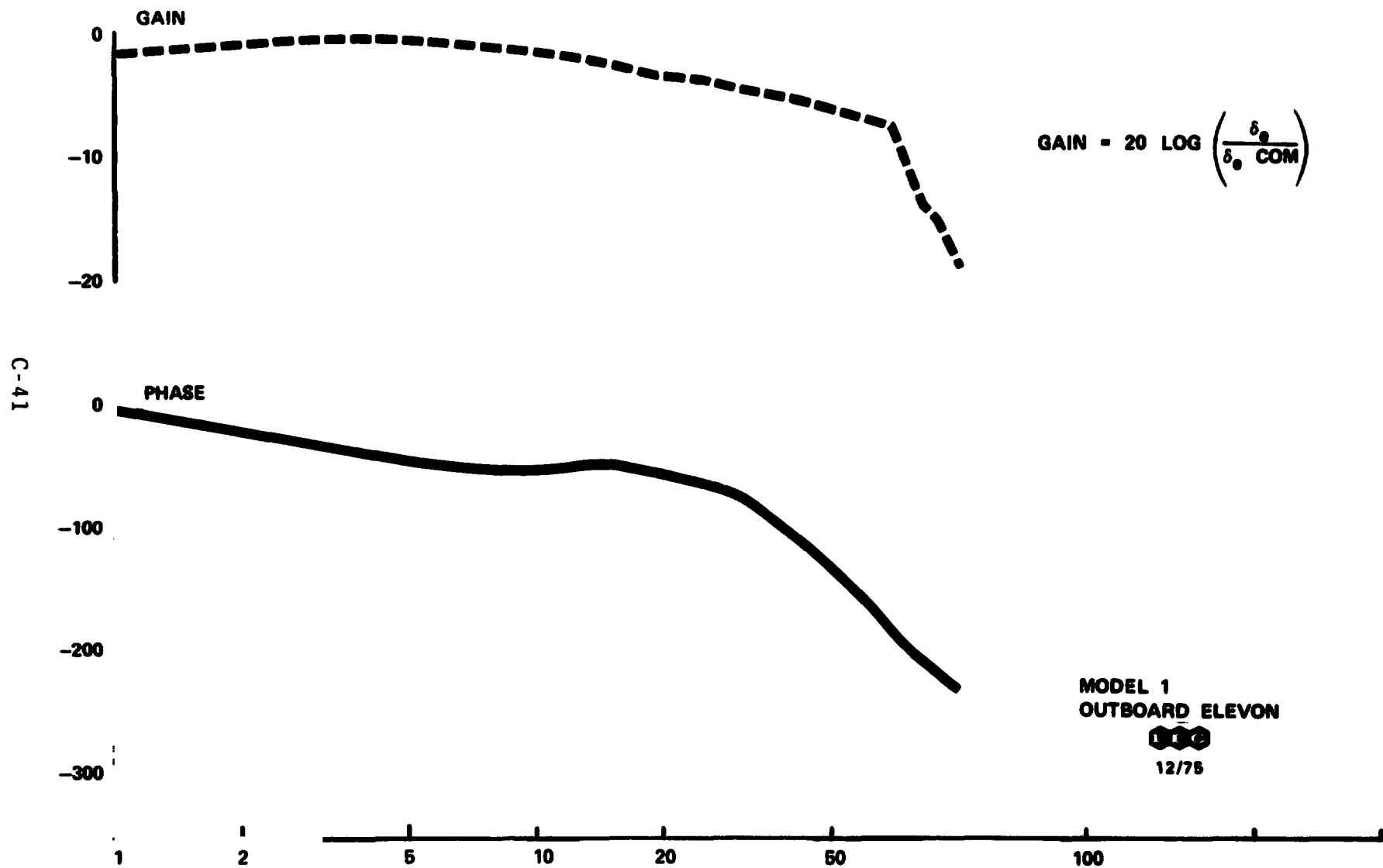
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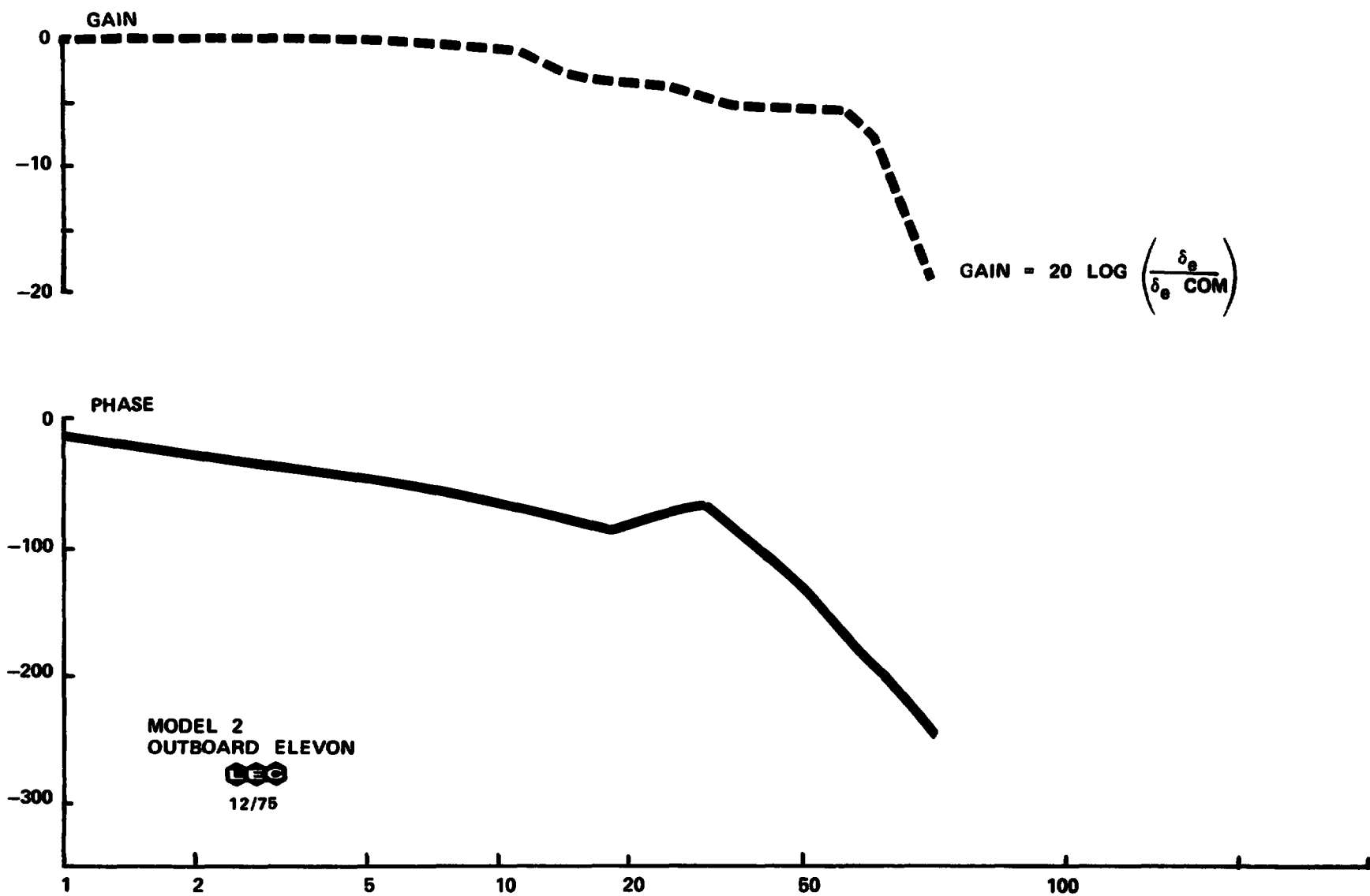
MODEL 2
OUTBOARD ELEVON



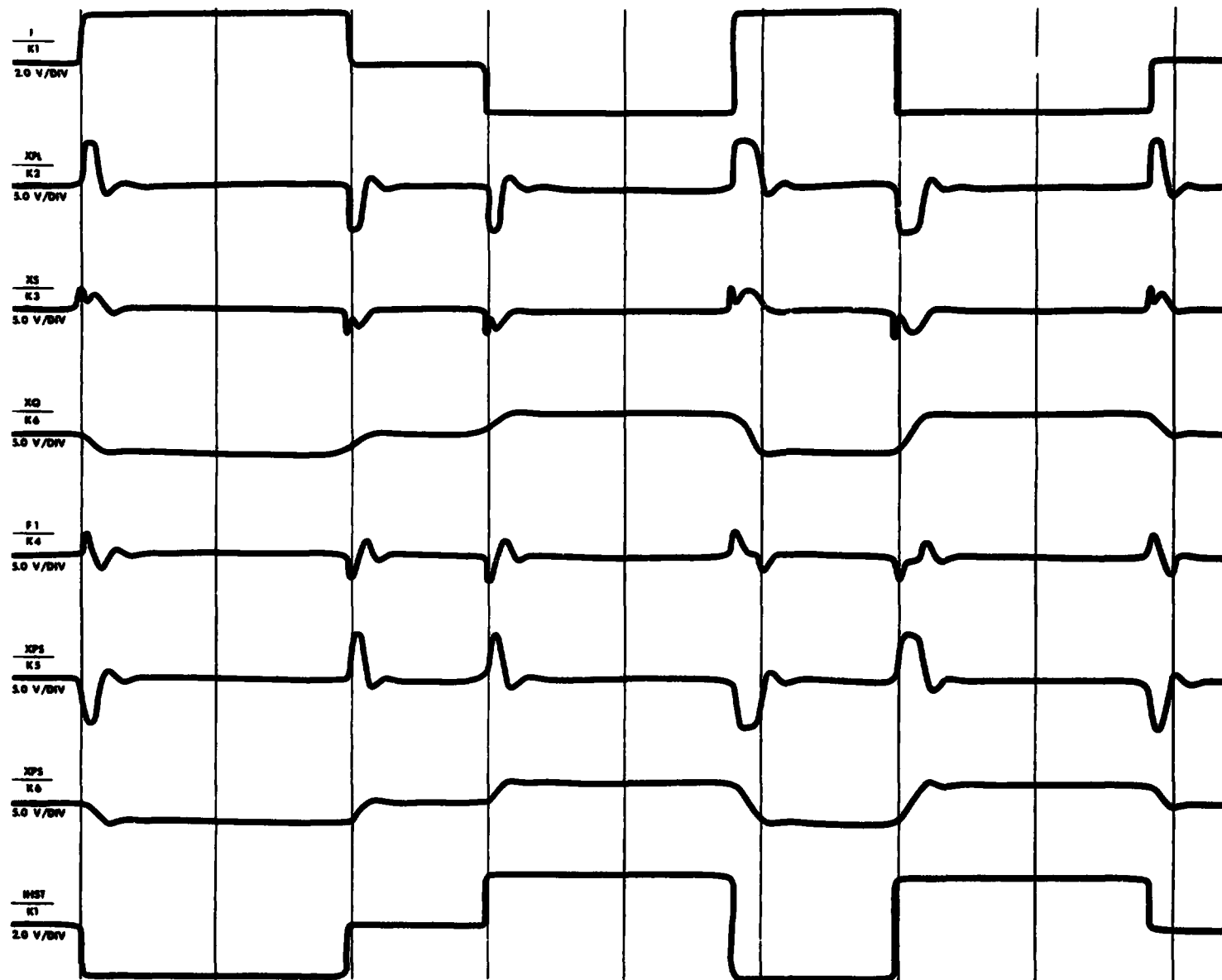
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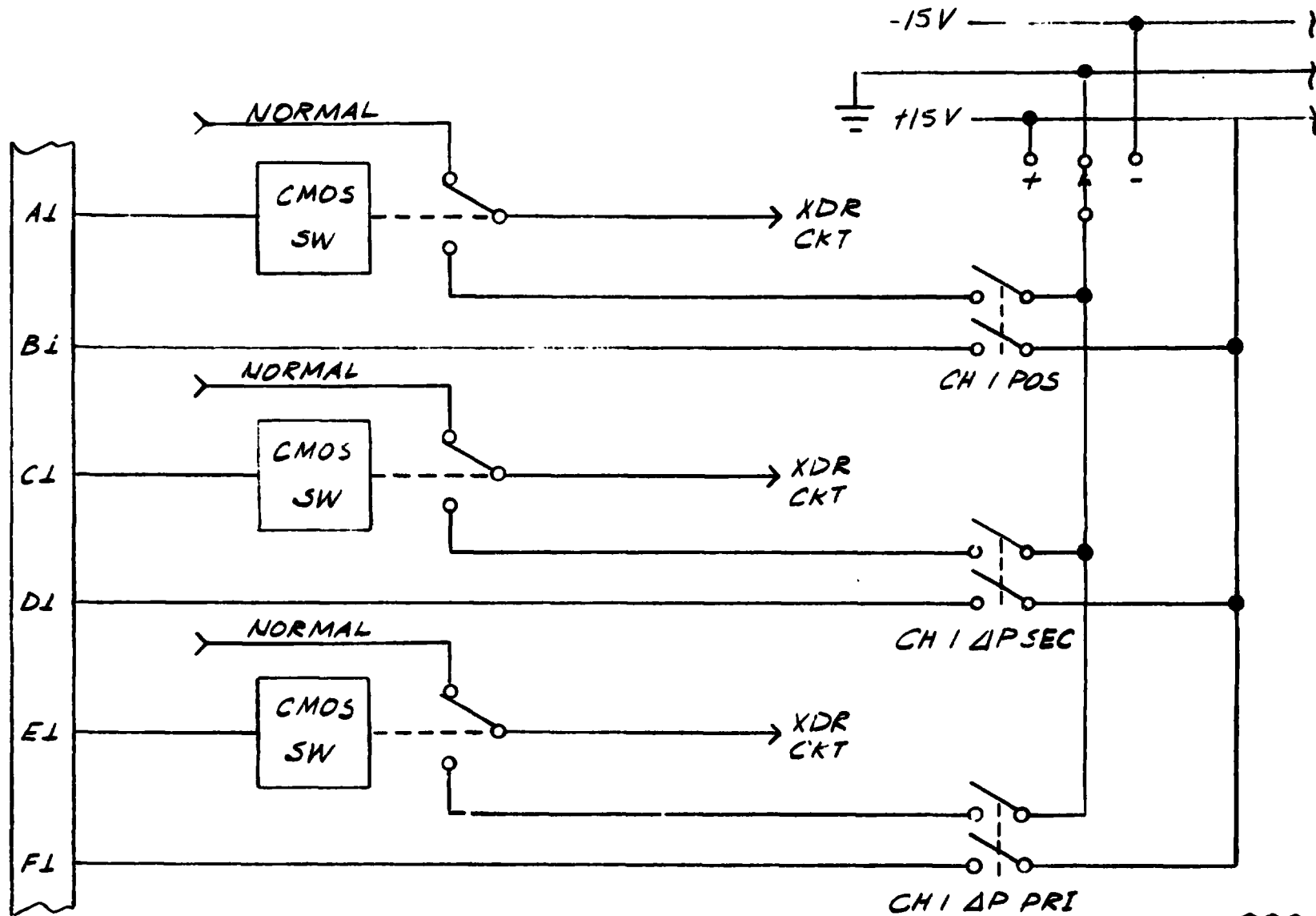
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C-43



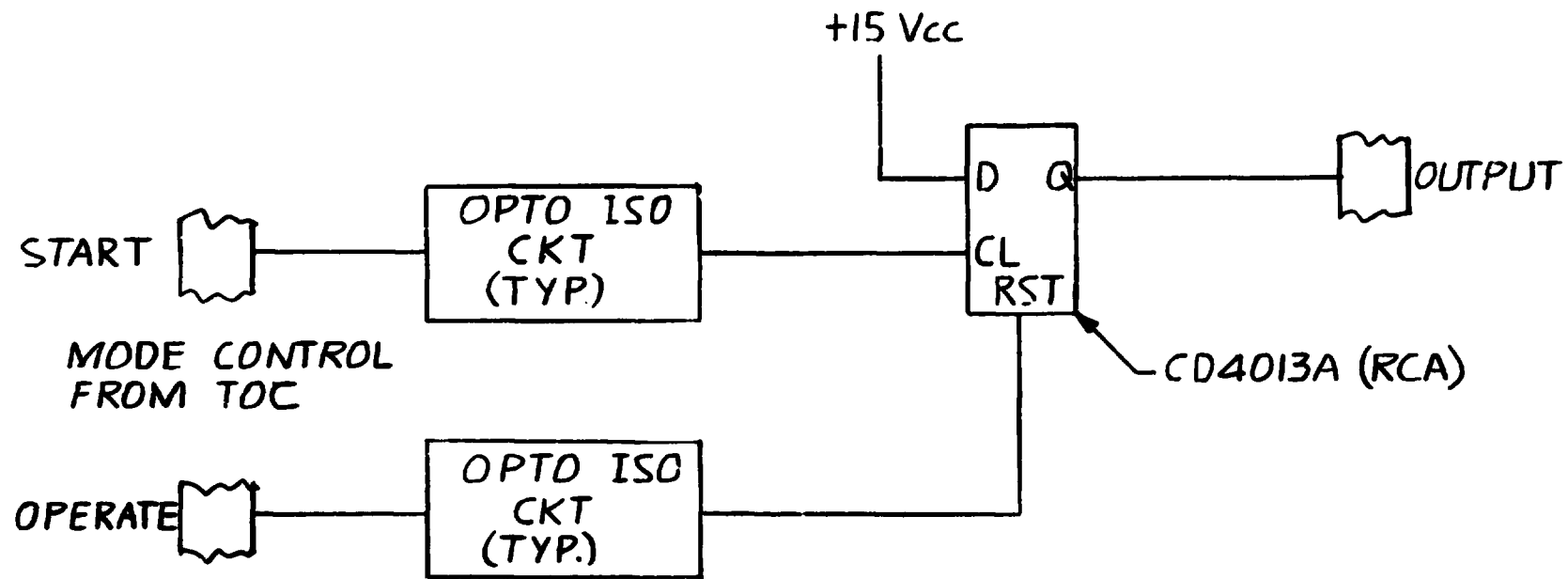
RUN ON C-173



SAS/TOC INTERFACE

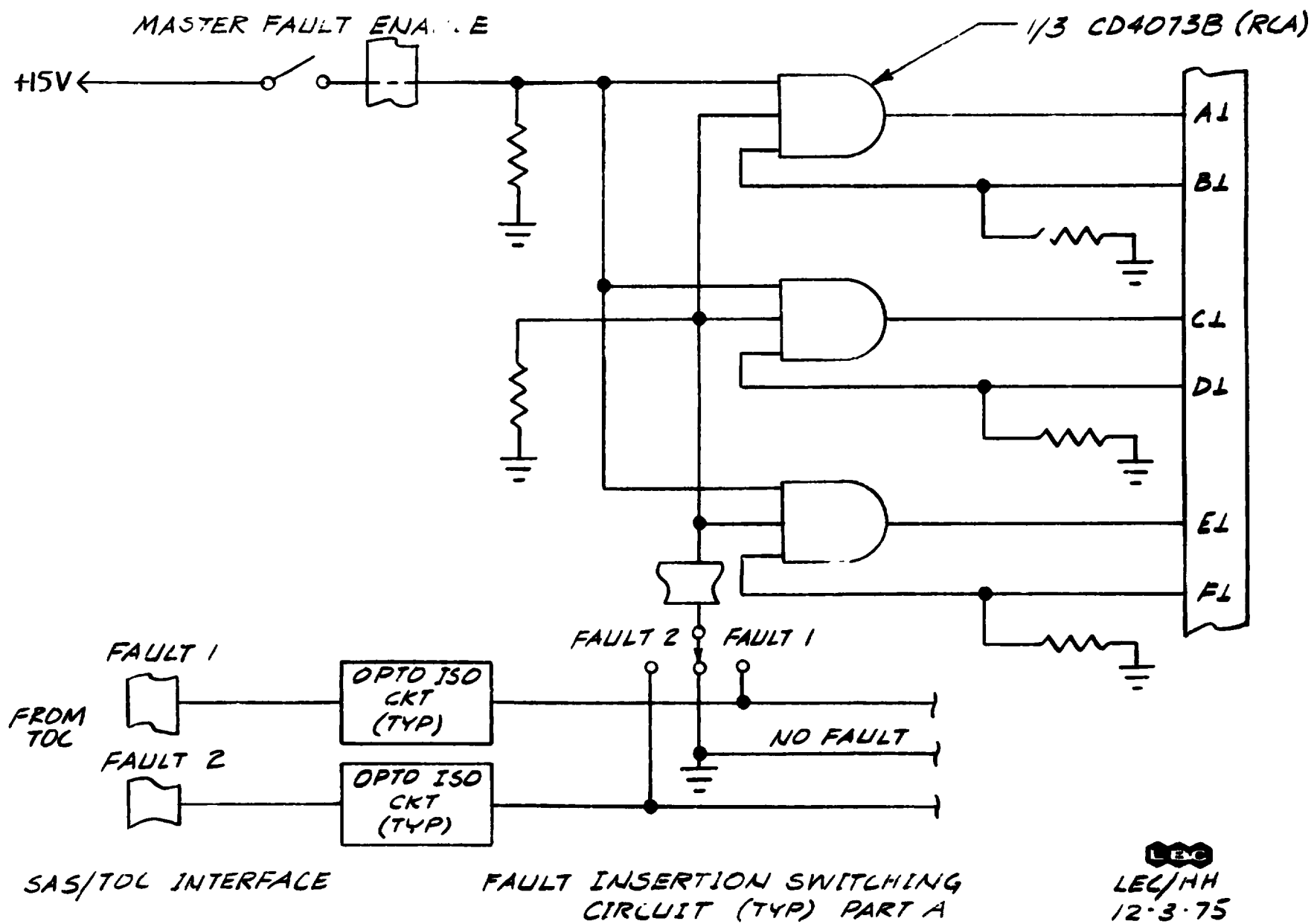
 FAULT INSERTION SWITCHING
 CIRCUIT (TYP) PART B

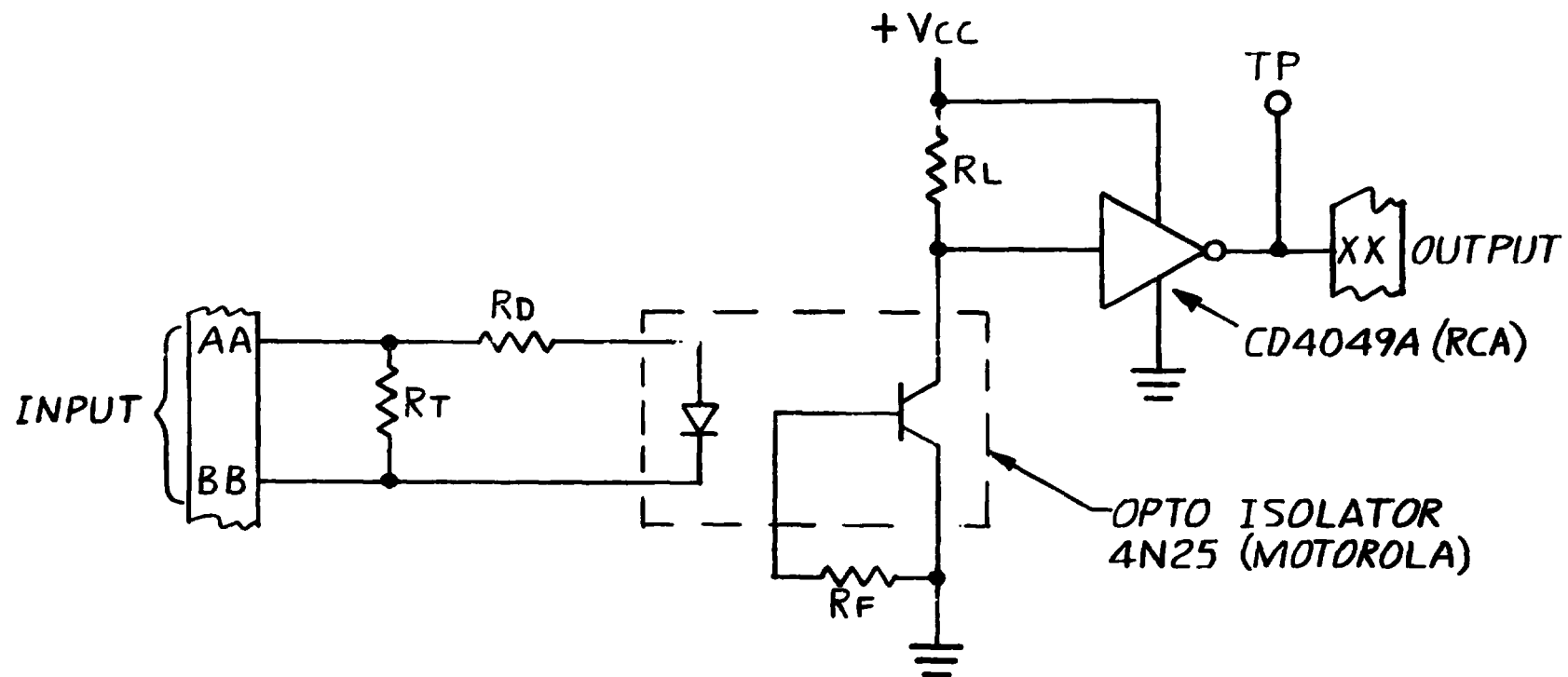
 LEC
 LEC/HH
 12-3-75



MODE CONTROL CIRCUIT (TYPICAL)

LEC
LEC/HH
12-3-75

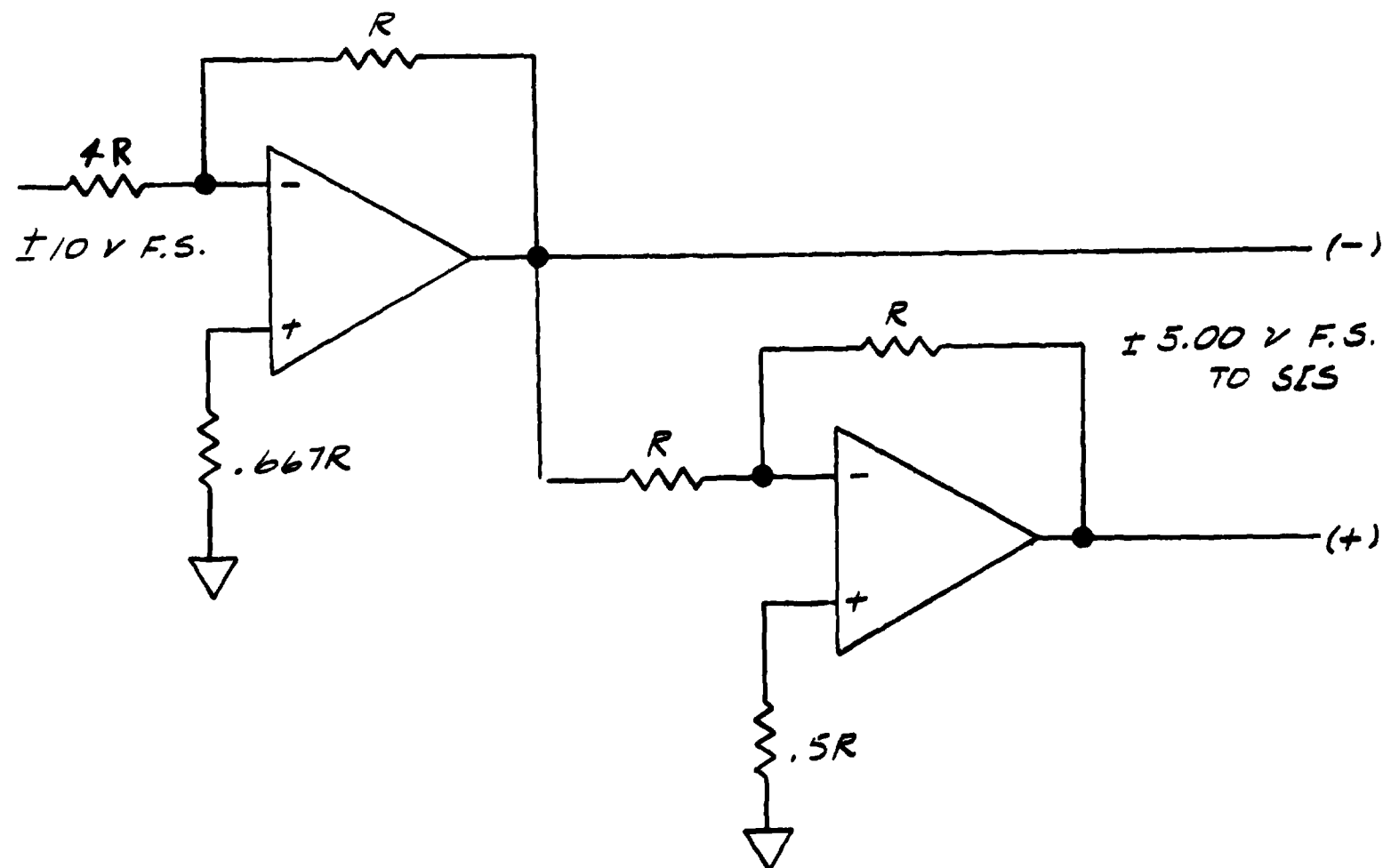




OPTO-ISOLATOR CIRCUIT (TYPICAL)
MODE / FAULT INSERTION


LEC
LEC/HH
12-3-75

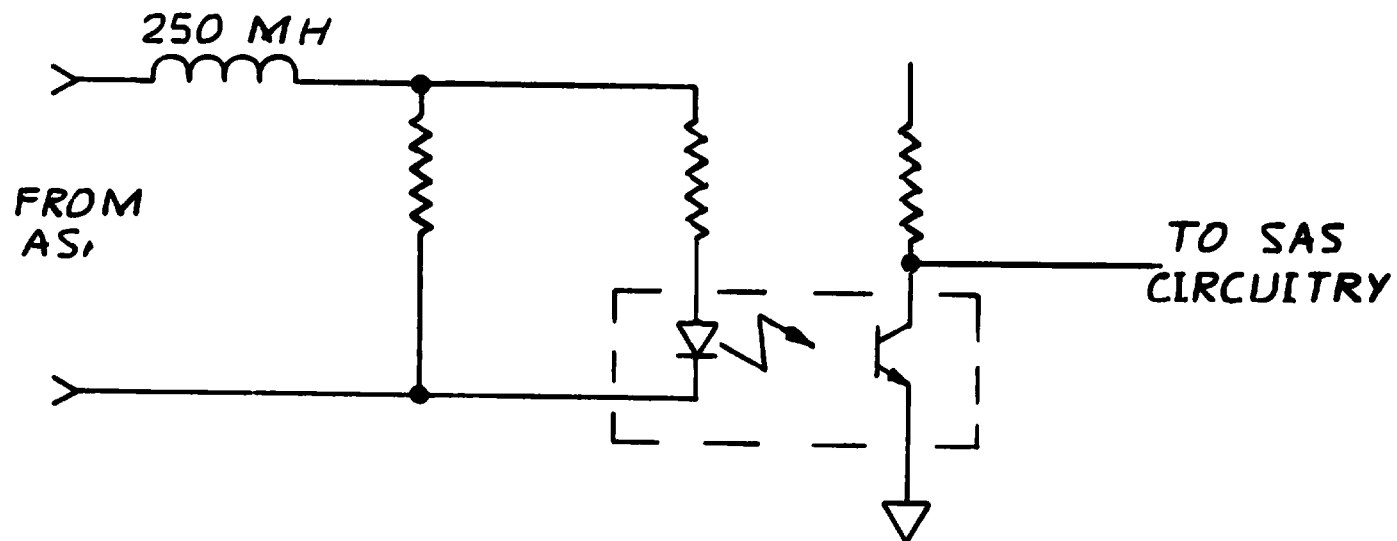
C-48



POSITION/ACCELERATION

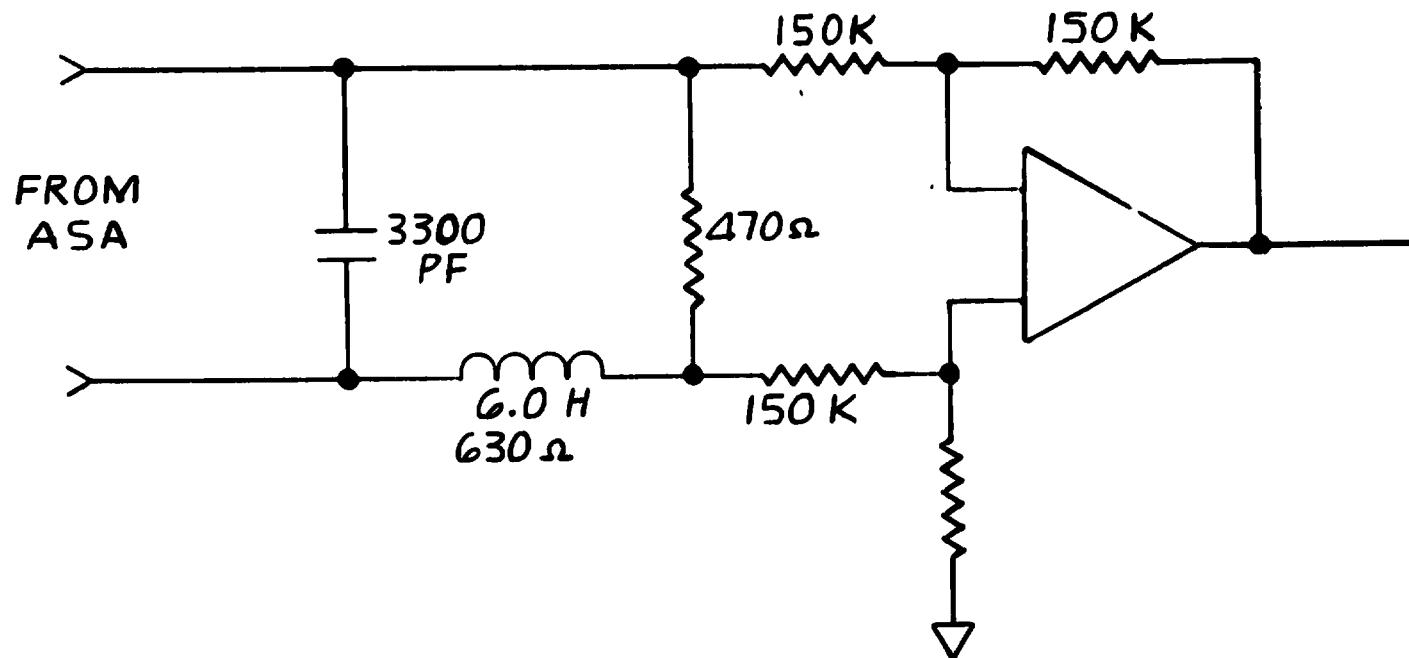
SAS/SIS INTERFACE


LEC/CLF
12/2/75



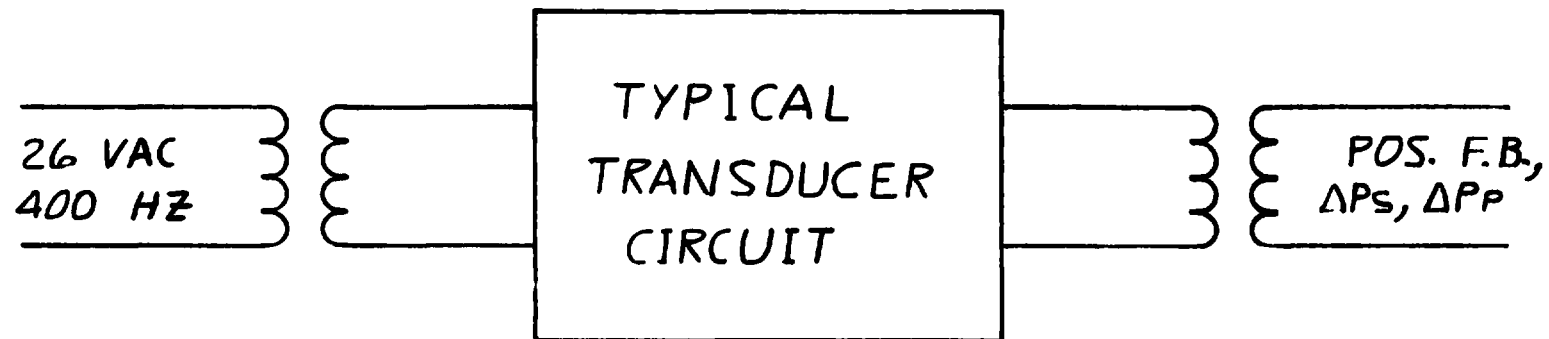
ELEVON ISOLATION VALVE
ASA/SAS INTERFACE

LEC
LEC/CLF
12-2-75



ELECON SERVO VALVE
ASA/SAS INTERFACE

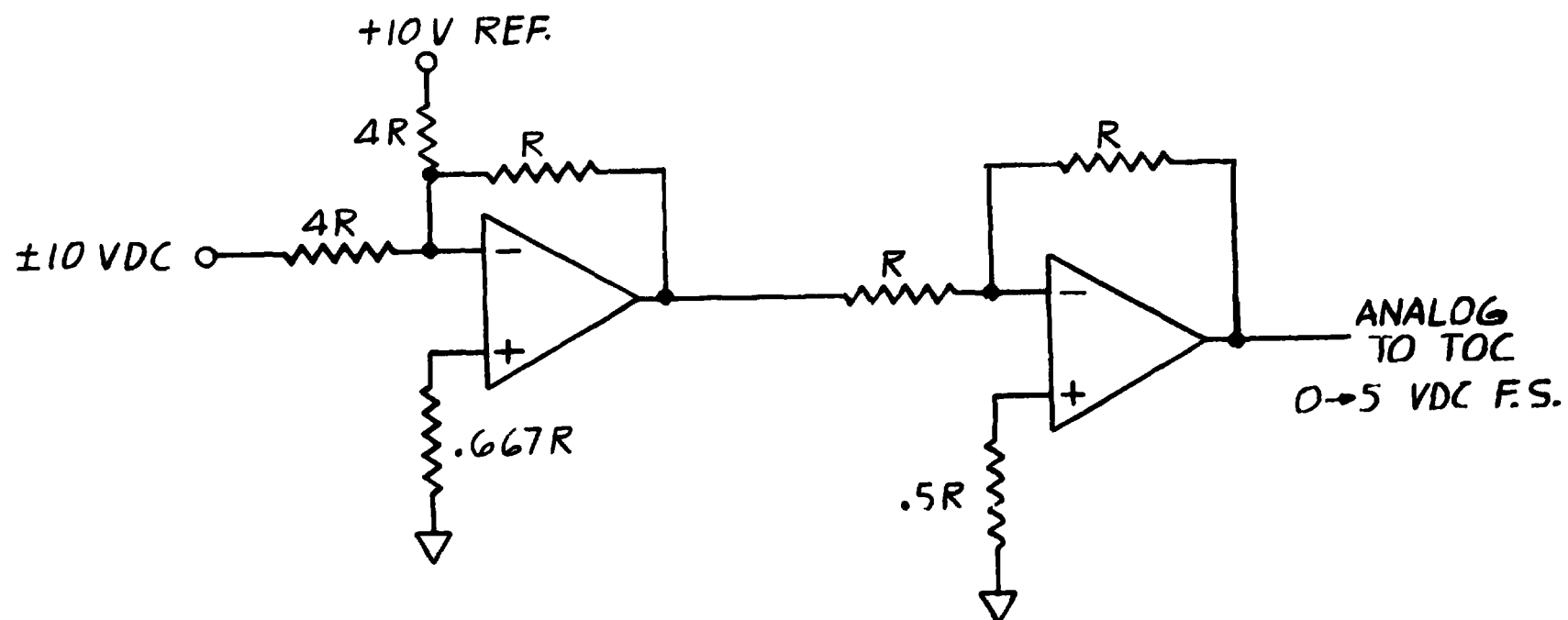
LEC
LEC/CLF
12-2-75



C-51

TYPICAL TRANSFORMER COUPLED
ISOLATION CIRCUIT ASA/SAS

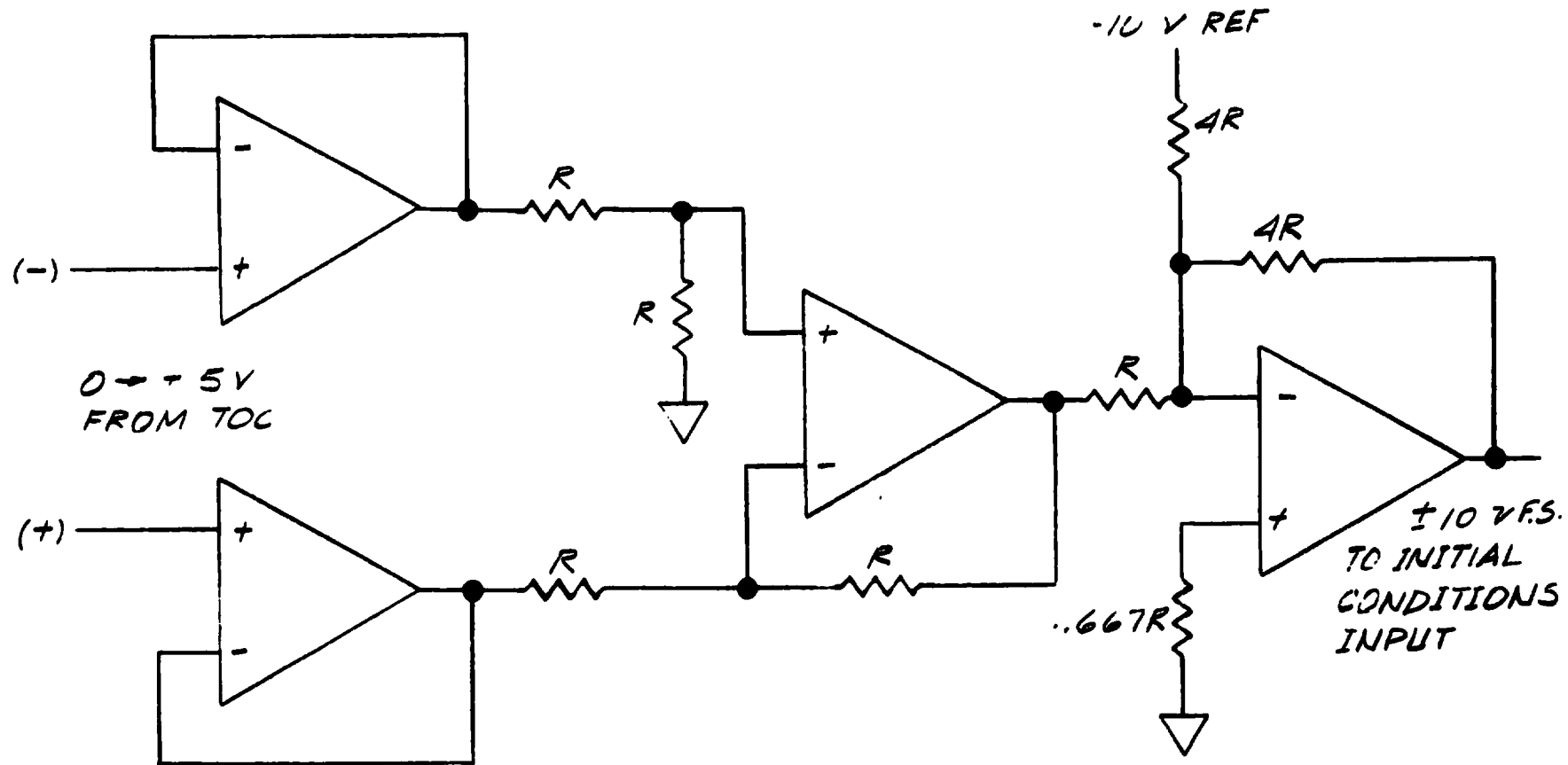
LEC
LEC/CLF
12-2-75



RATE/POSITION
SAS/TOC INTERFACE

LEC
LEC/CLF
12-2-75

C-53

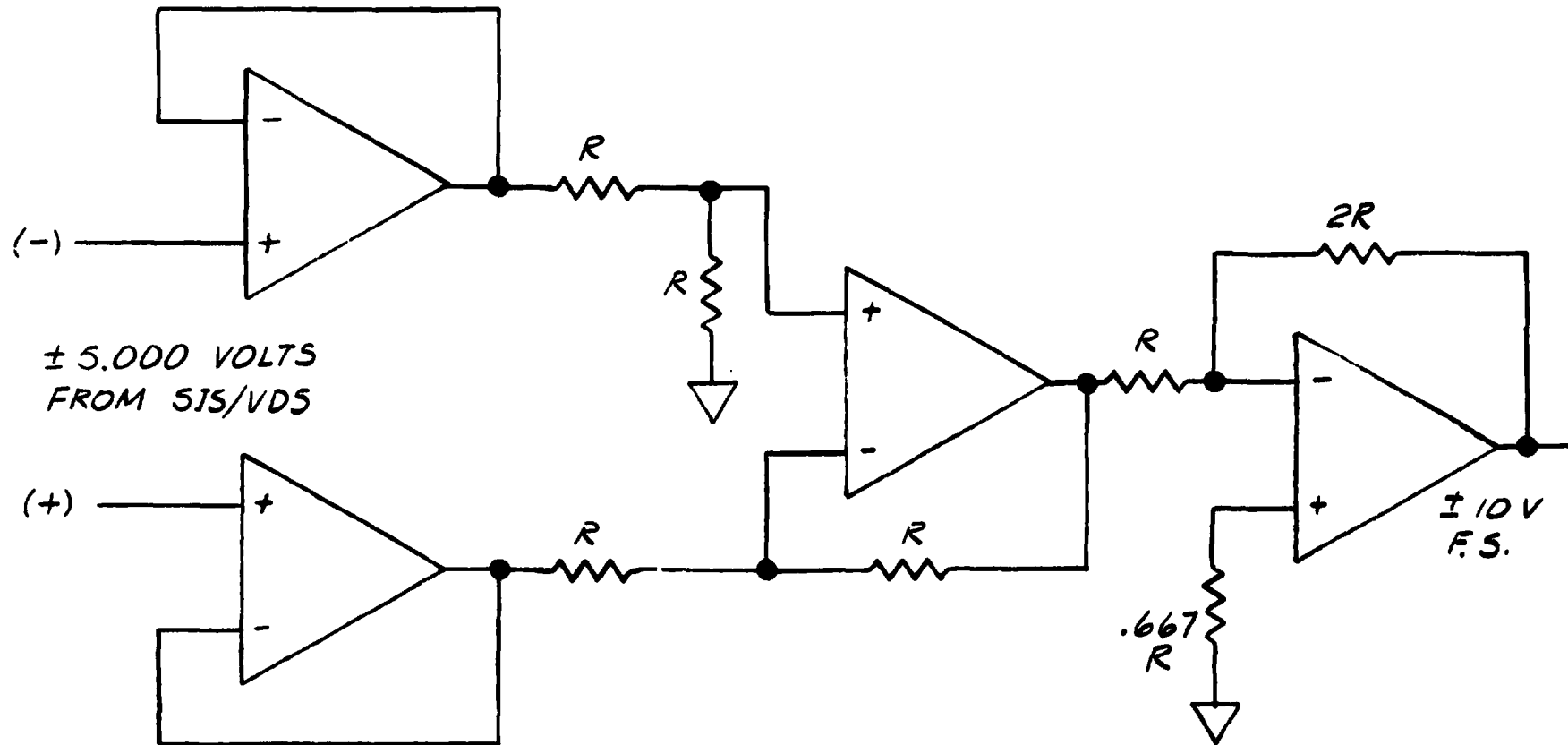


INITIALIZATION

TOC/ASA INTERFACE

LEG
LEG/CLF
12/2/75

C-54

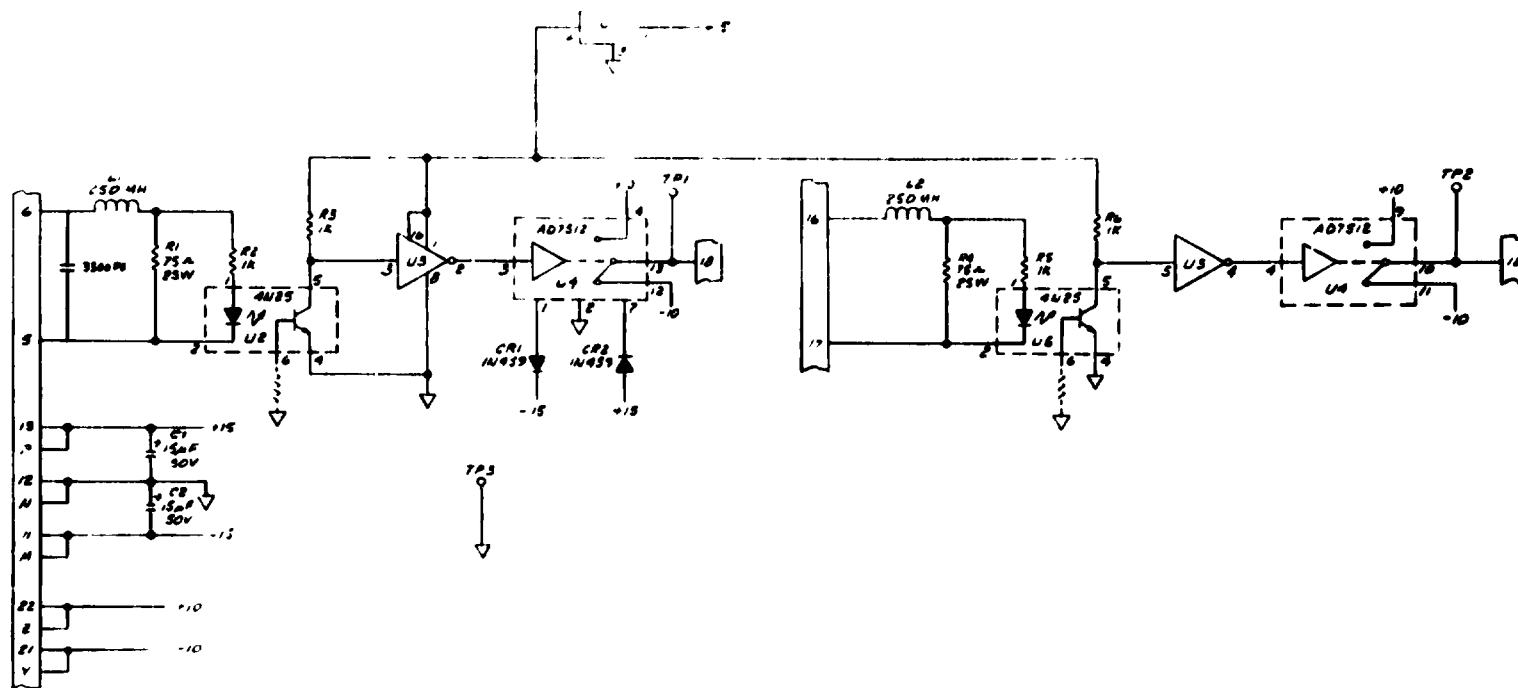


HINGE MOMENT

SIS/SAL INTERFACE

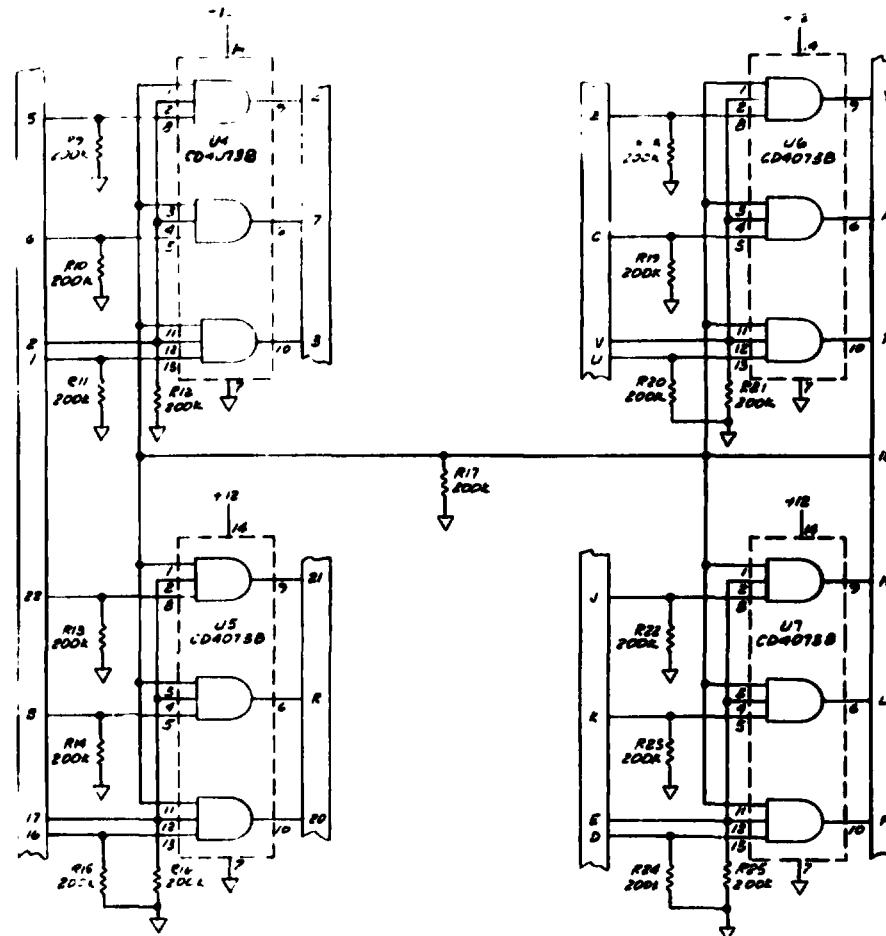
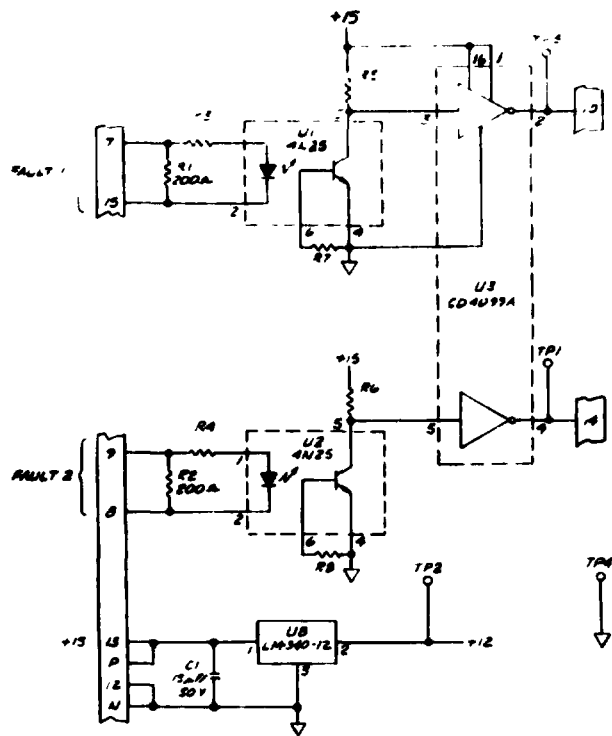
LEC
LEC/CLF
12/2/75

ISOLATION VALVE, INTERFACE, SIMULATOR ACTUATOR SUBSYSTEM



C-55

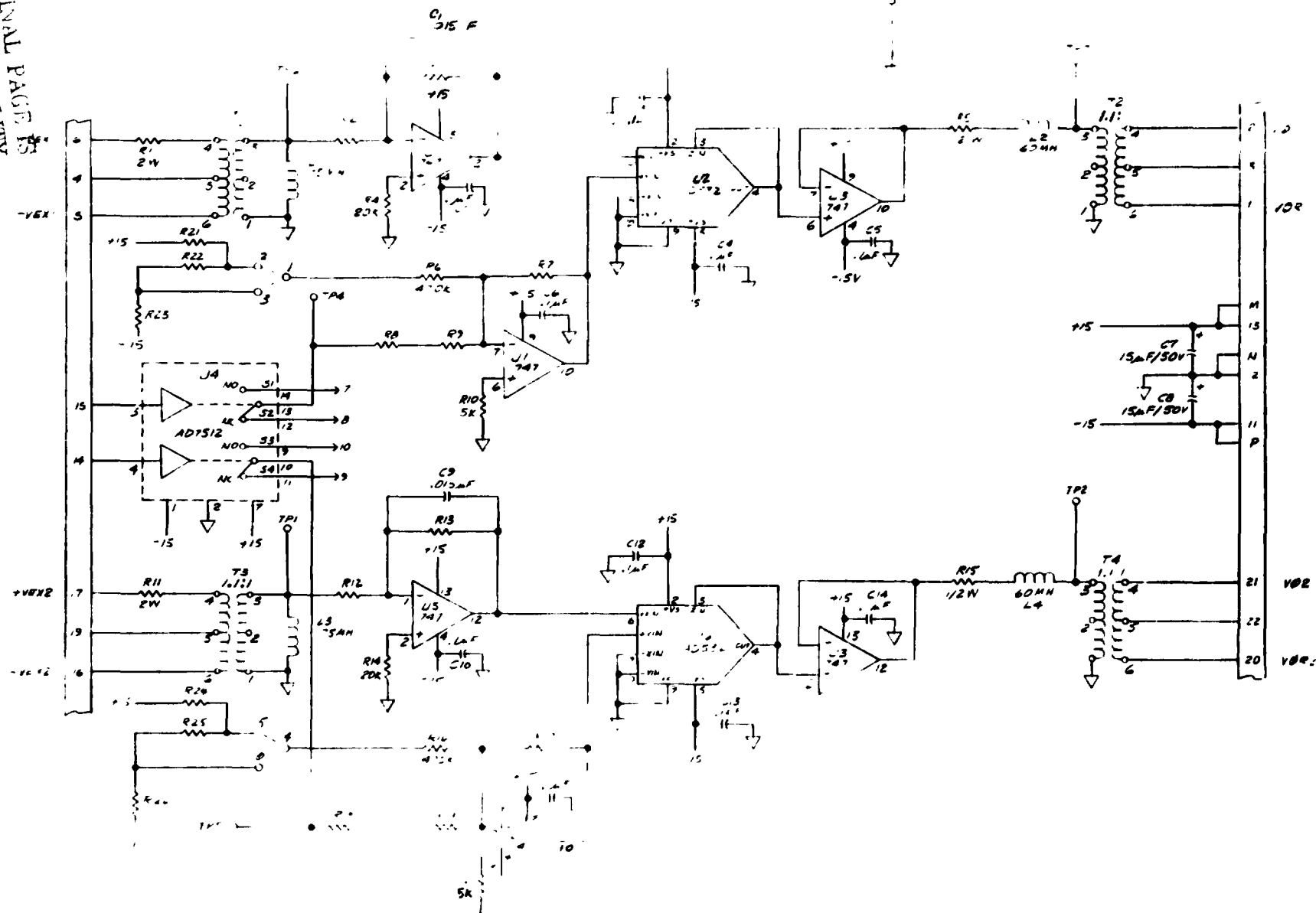
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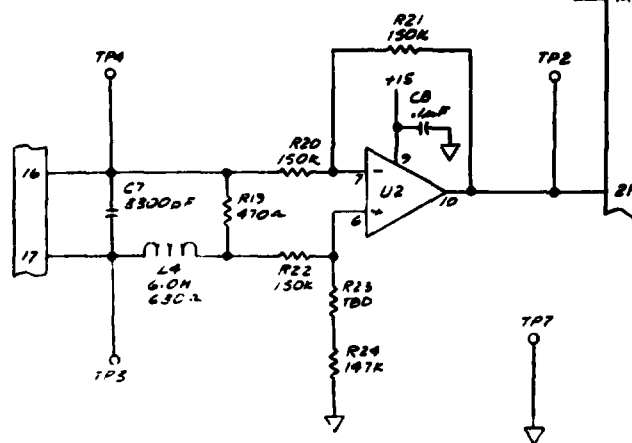
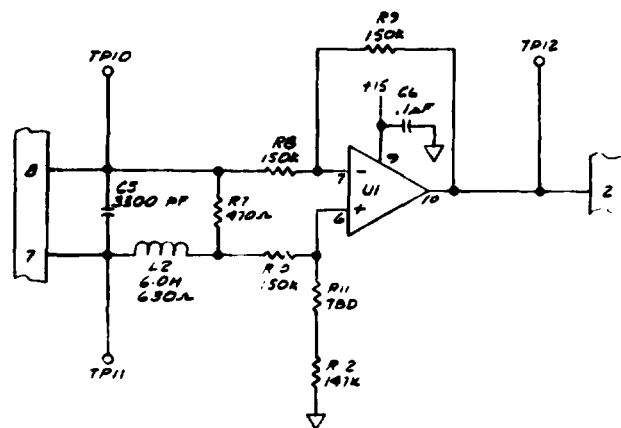
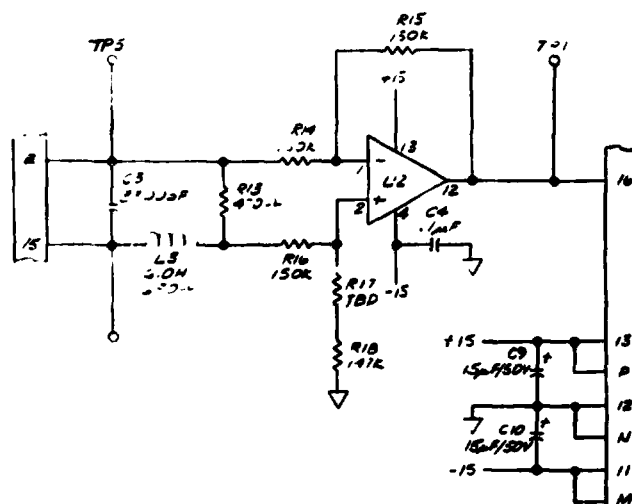
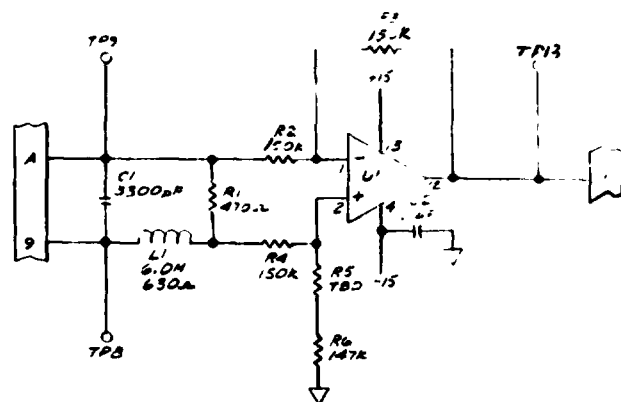
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C-57

PRESSURE/POSITION TRANSDUCER SIMULATOR ACTUATOR SUBSYSTEM



C-58



APPENDIX D

ELEVON CONTROL VALVE MODULE MEMORANDUM

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LOCKHEED ELECTRONICS COMPANY, INC.
AEROSPACE SYSTEMS DIVISION

16811 EL CAMINO REAL

HOUSTON, TEXAS 77058

TELEPHONE (AREA CODE 713) 488-0000

6 January, 1976
GC-5479-620

TO : H. Shelton
NASA-JSC/EG5

FROM : J. C. Barr
LEC-ASD/C08

SUBJECT : DEVELOPMENT OF MODEL FOR ELEVON
CONTROL VALVE MODULE

REFERENCE : MINUTES OF MEETING, SHUTTLE ACTUATORS
COORDINATION WORKING GROUP, DEC. 10, 1975.

The SAS design review for the Elevon resulted in several action items. Item 2 involved anomalies in the implementation model APs time history trace for various inputs. Resolution of this item was assigned to J. Barr.

This document presents the method and results for resolution of action item 2. The paper contains two parts (1) the development and verification of a reduced model, and (2) a comparison of this reduced model with the implementation model that was presented in the design review. The new reduced model will add a slight degree of complexity to the hardware control valve module mechanization.

The anomalies were concerned primarily with the ramp (constant current) input. Mr. Jack Hoke, of Rockwell International, maintained that the APs trace should be very similar for the full-up and reduced models. In order to resolve the discrepancy it was decided to redevelop the reduced model, with additional checks at each stage of the reduction.

The reduction was accomplished in three steps. The first step eliminated the second stage valve dynamics. The second step

Page 2
6 January 1975
GC-5479-620

removed the flapper valve lag time. The third step added a first order lag to X_s . The full-up model and its response is shown in figure 1. The peak pressure is 34.8 psi.

The first reduction affected X_s , second stage valve position, and its velocity. The dynamics and stiction were eliminated, as was the flow feedback term to the flapper valve. The resulting control valve model and its step response is shown in figure 2. The peak pressure is 138 psi. at the time of the step input. Within 40 msec the response has settled to the full-up response.

The second reduction affected P_N , flapper valve pressure. The first order lag was eliminated. The resulting control valve model and its response is shown in figure 3. The peak pressure is 196.4 psi at the time of the step input. Within 40 msec the response has settled to the full-up response.

The final step affected X_s , second stage valve displacement. A first order lag term is added to improve the rate response. The secondary pressure spike of 196.4 psi shown in figure 3 has been reduced to 88.4. The resulting control valve model and its response is shown in figure 4.

Figure 4 is the recommended implementation model. The remainder of this paper presents a comparison of the model of figure 4 and the implementation model presented at the design review on December 10, 1975.

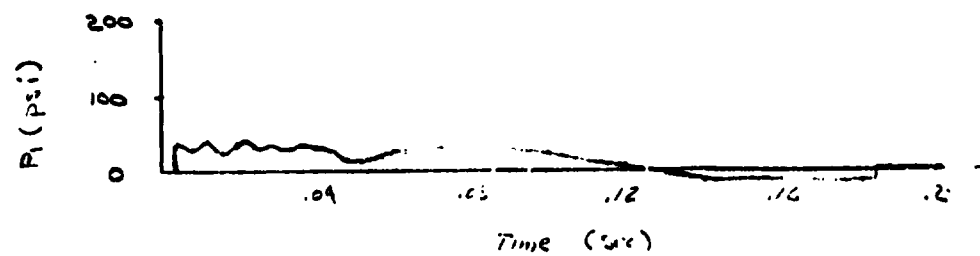


FIGURE 1 - FULL-UP MODEL



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D-5

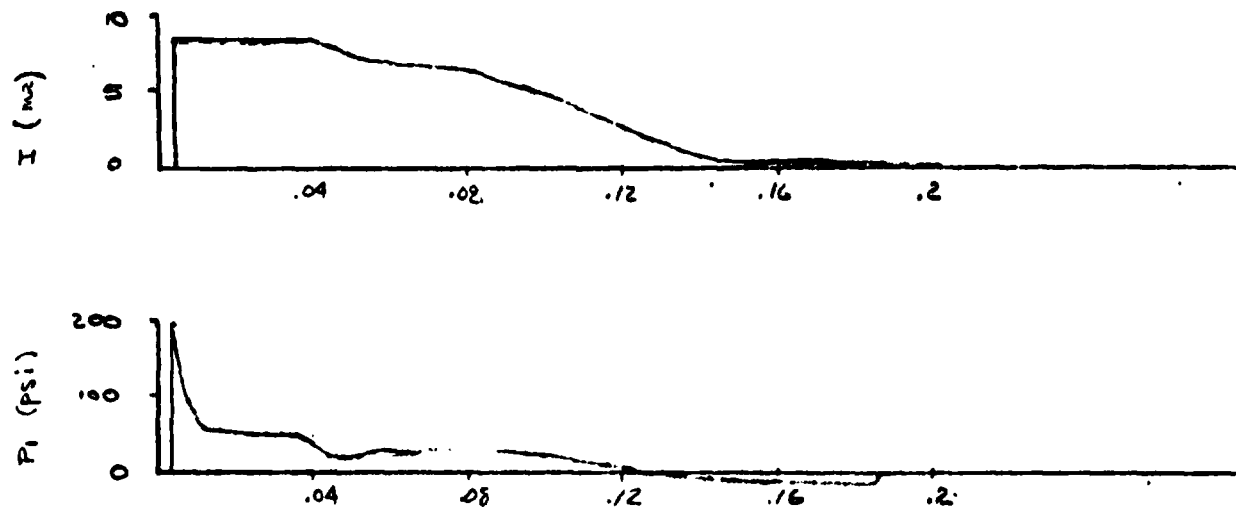


FIGURE 3 - REDUCTION OF FLAPPER VALVE

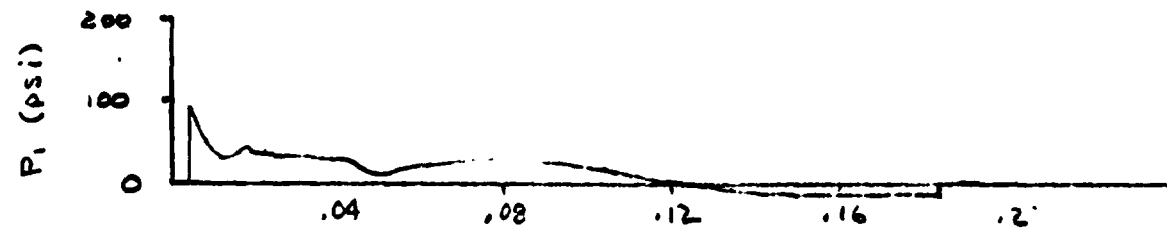
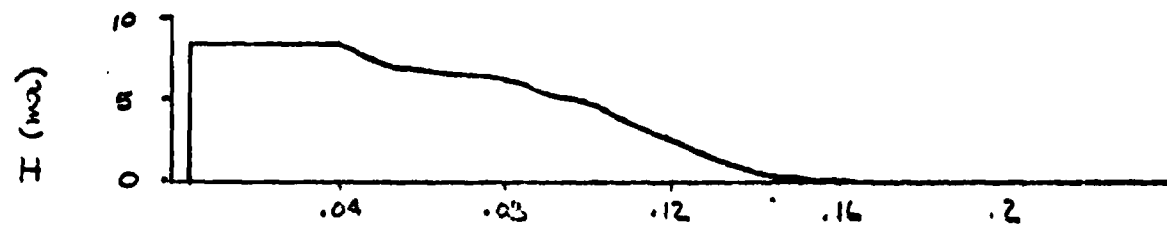
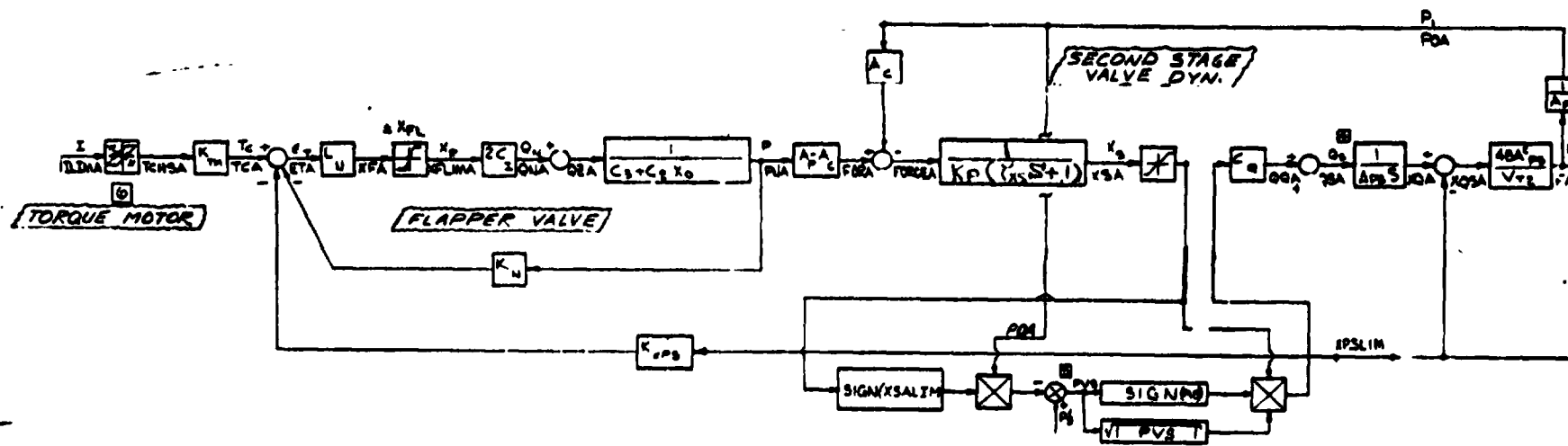


FIGURE 4 - FINAL REFINED MODEL

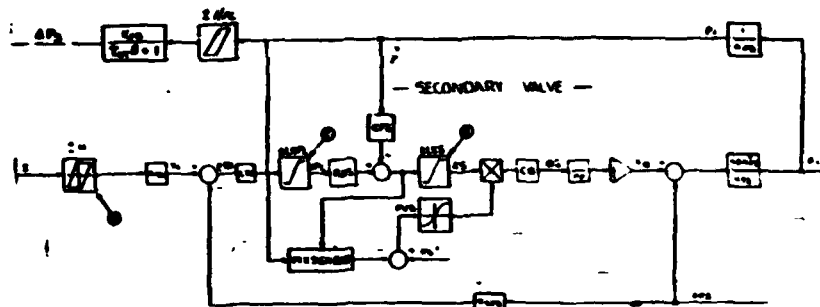
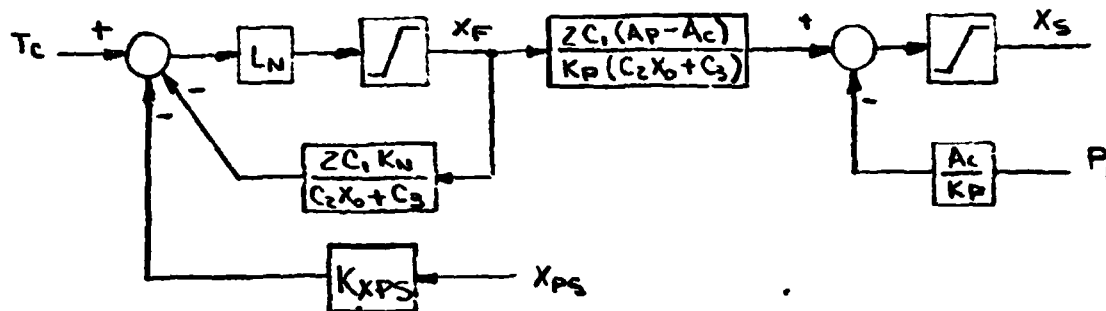


Figure 5.

Figure 5 is the implementation model that was presented at the Elevon design review. Figures 4 and 5 look quite different in the region of the flapper valve. Redrawing the flapper region of figure 4 and combining cascaded gains:



For the implementation model, figure 5, KQS was chosen by a small signal reduction which eliminated the XF limiter. The combined KQS and Kps were thus:

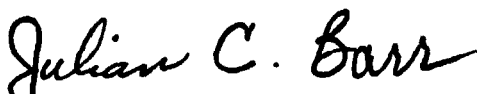
$$KQS = \frac{2C_1(A_p - A_c)}{K_p(C_2X_0 + C_3 + 2C_1L_NK_N)} \quad \dots \dots (1)$$

$$KPS = \frac{A_c}{K_p} \quad \dots \dots (2)$$

And by the small signal assumption the XF limiter is insignificant to the model performance.

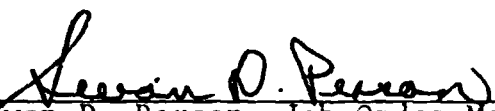
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Large signal analyses run via computer programs demonstrated that the small signal assumption was invalid and the XF limiter was significant to the model. It is recommended that the implementation model be changed to that of figure 4.



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